



**NIST Technical Note  
NIST TN 2249**

# **A Gray-Box Model of a Two-Stage Heat Pump for Electrical Load Forecasting in a Single-Family Residence**

Farhad Omar, Ph.D.

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March 2023



U.S. Department of Commerce  
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National Institute of Standards and Technology  
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NIST TN 2249  
March 2023

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### **Publication History**

Approved by the NIST Editorial Review Board on 2023-03-09

### **How to Cite this NIST Technical Series Publication**

Farhad O (2023) A Gray-Box Model of a Two-Stage Heat Pump for Electrical Load Forecasting in a Single-Family Residence. (National Institute of Standards and Technology, Gaithersburg, MD), NIST Technical Note (TN) NIST TN 2249. <https://doi.org/10.6028/NIST.TN.2249>

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## **Abstract**

Buildings as a set of electric loads and generation resources can play an essential role in managing the stability of the power system in a smart grid. Traditionally, buildings assumed a passive role in the day-to-day operation of the electric grid, with utilities controlling the supply of energy to match the demand for buildings. However, the dynamics of the power distribution grid are rapidly changing. A critical aspect of managing the balance between supply and demand in the distribution grid is estimating future demand through load forecasting. Load forecasting requires accurate, adaptive, and simple models to account for the dynamic behavior of the electrical demand. In residential buildings, heat pumps are a significant source of energy consumption. This document describes a novel gray-box model (RL Model) to forecast the steady-state and transient current consumption for a two-stage, air-source heat pump in heating and cooling seasons. The RL Model is derived from a solution of a first-order differential equation describing the voltage and outdoor temperature dependent characteristics of a resistor-inductor equivalent circuit. Key parameters of the RL Model were estimated using a learning algorithm. The performance of the RL Model is validated using measurement data from the Net-Zero Energy Residential Test Facility located on the National Institute of Standards and Technology (NIST) campus in Gaithersburg, MD. The predicted output of the RL Model, current consumption, is used to estimate the heat pump's real and reactive power consumption for different operating stages and seasons. A list of key parameters, e.g., the time constant and temperature-dependent coefficients of the RL Model, are tabulated. Knowledge of the time constant provides critical information for analyzing the aggregated effect of controlling many heat pumps as flexible loads on grid stability and provision of ancillary services. The average root mean squared error between the RL Model's predicted current output and the measured current consumption is 0.06.

## **Keywords**

Data-driven model; energy use forecasting; gray-box model, heat pump modeling; heat pump real and reactive power; heat pump time constant; load forecasting; load modeling; parameter fitting; parameter optimization; RL model; temperature dependent heat pump model.

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## 1. Introduction

The electrical grid is undergoing significant changes due to a two-way flow of energy and information. Buildings as a set of electric loads and generation resources can be essential in managing grid stability and supporting decarbonization efforts while maintaining thermal comfort and reducing cost. Usually, buildings assumed a passive role in the day-to-day operation of the electric grid, with utilities controlling the supply of energy to match the demand for buildings. However, the dynamic behavior of the power distribution grid is rapidly changing. These changes are due to the integration of renewable energy sources, the proliferation of electric vehicles, including electric storage, and the dynamic behavior of energy-consuming loads [1, 2].

Recent studies show that building loads can provide ancillary services such as voltage control, frequency regulation, reserve, and load following to maintain grid stability [2–5]. Grid stability also requires a balance between supply and demand for electricity. An essential component of managing grid stability is forecasting the expected load. Load forecasting enables utility planners, aggregators, and customers to devise mitigating control strategies for efficient and cost-effective grid operation. Developing effective and adaptive control strategies requires accurate load models for capturing the dynamic behavior of the electrical load. Load modeling is a critical component of power system studies [4, 6–8].

Traditionally, utilities relied on aggregated load models to maintain the grid's stability by managing the balance between supply and demand, encompassing the residential, commercial, and industrial sectors. However, the proliferation of energy-efficient and flexible non-linear loads [9], integration of renewable energy sources, and demand-side management implementation highlighted the need to update the existing load model [8]. In power system studies, electrical load modeling can broadly be classified into static and dynamic approaches [8, 9]. Static models describe the instantaneous real and reactive power consumptions as a function of the bus voltage magnitudes and system frequency. These models, such as power flow studies, are typically used for steady-state analysis. In contrast, dynamic load models are used for stability analysis because they exhibit a time-dependent response to changes in the voltage and frequency of the system. A comprehensive review of load modeling approaches for power system studies are presented in [8, 9].

Heating, ventilating, and air-conditioning (HVAC) is one of the largest electrical loads in a typical building. Air-source heat pumps are a common type of HVAC equipment in residential homes and are typically the largest load. It is important to predict the electrical energy consumption of the air-source heat pumps to evaluate control strategies involving demand response, voltage control, frequency regulation, or reserve for grid stability or delivery of ancillary services. A review of heat pump modeling and control strategies used for ancillary services and grid stability is provided in [3, 10–12].

There are many ways to model the performance of residential heat pumps [13]. However, for modeling the performance of a heat pump, the three most common approaches are: physics-based (white-box), data-driven (black-box), and a hybrid of physics-based and data-driven (gray-box) models [8, 9, 13]. Developing a white-box model requires detailed knowledge of the underlying physical principles governing the operation of a heat pump and its components. In contrast, black-

box models are purely data-driven. Black-box models lack knowledge of the underlying physical structure of the components of a heat pump and their operation. These models require a large dataset for training and learning purposes and do not provide a precise meaning for the underlying physics of the units. Gray-box modeling is a hybrid method often developed based on a simplified version of the physical process using equivalent circuits and measurement data. Measurement data is used to estimate the parameters of the simplified physics-based model. The gray-box models do not require a large set of data for training. Another advantage of the gray-box approach is that it can provide a clear physical meaning for specific parameters of the model compared to the black-box model of the same process. Gray-box models are also more adaptable because the values of new parameters can be estimated from a different dataset, capturing the unique characteristics of another residential heat pump.

Essential components of a heat pump include an evaporator, a compressor, a condenser, an expansion valve, and power electronics. Generally, it is hard to model the exact characteristics of individual heat pump components and describe their overall impact on power consumption. Therefore, a common technique, in power system studies, for capturing the overall performance of a heat pump is to create a lumped model, representing the combined effect of individual components. These lumped techniques include using equivalent circuit representation of an induction motor (IM) [8, 9, 14, 15], exponential and polynomial ZIP (constant impedance, constant current, and constant power, respectively), and composite load models (ZIP + IM) [8, 9, 16] to model residential heat pumps. Poudel et al. [14] presented an aggregate model of single-phase induction motors to represent residential heat pump loads in steady-state and transient operation for fault analysis. The aggregated model was derived by combining equivalent circuit models of single-phase induction motors. The parameters of the aggregate equivalent circuit were determined from no-load and locked rotor tests. Schneider et al. [17] presented a multi-state (off, cooling, heating-normal, and heating-auxiliary) model of a residential heat pump. The multi-state model in [17] is a single-stage heat pump that provides thermal energy as a function of heat flow rate through the cooling unit and the compressor's electrical losses. The heat flow rate is obtained from the heat transfer equations describing the mechanical torque of the compressor. The electrical losses of the compressor are captured by an equivalent circuit model of a split-phase IM.

However, using the IM and ZIP+IM models to represent many heat pumps for power system studies and simulation is computationally expensive because they contain many parameters. ZIP models can only describe the on or off state of a heat pump's operation [17] and capture the steady-state power consumption. ZIP models are inadequate to properly capture the dynamic behavior of heat pumps. ZIP models are insufficient because a heat pump's energy consumption is not only a function of terminal voltage but also a function of the outdoor temperature and control algorithms used to manage the heating and cooling operation. Therefore, a simpler approach is needed to model the operation of a heat pump that might be used in a residential home.

In this work, a novel gray-box model for predicting the current consumption of a two-stage heat pump in a single-family residence was developed. The approach was to define a simple resistor-inductor (RL Model) equivalent circuit and modify it to account for the outdoor temperature's impact on the heat pump's power consumption in both transient and steady-state conditions. A learning algorithm (LA) was defined to learn the key parameters of the RL Model using measurement data instead deriving them from detailed knowledge of the physical thermal and electrical characteristics of the heat pump. The performance of the RL Model was validated using

performance measurements from the Net-Zero Energy Residential Test Facility (NZERTF) at the National Institute of Standards and Technology (NIST) in Gaithersburg, Maryland [18, 19]. The NZERTF house is a typical four-bedroom house in the Washington DC metro area with a detached garage. The exterior of the NZERTF is shown in Fig. 1.



**Fig. 1.** The NZERTF exterior

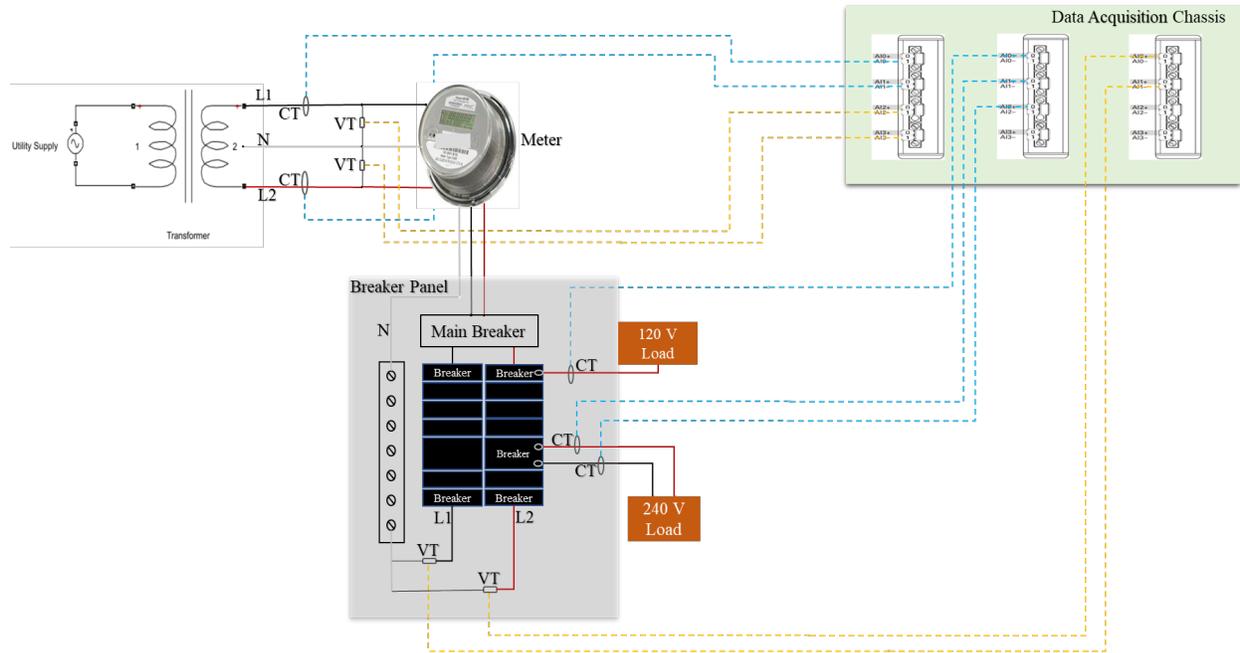
The main objectives of this technical note are:

1. To provide a brief description of the data acquisition setup for measuring the performance of a two-stage air-source heat pump in the NZERTF,
2. To describe a first-order RL Model for estimating the steady-state and transient current consumption of the heat pump in both heating and cooling seasons,
3. To describe the LA used for estimating the coefficients of the first-order RL Model, and
4. To estimate the heat pump's real and reactive power consumption using its predicted current consumption, measured line voltage, and displacement power factor.

## **2. Measuring the Electrical Performance of a Two-Stage Air-Source Heat Pump**

The main objectives of this research were realized by simultaneously measuring the performance of the air-source heat pump at the main circuit breaker panel inside the NZERTF and on the utility side of the meter located outside of the NZERTF. Fig. 2 shows a schematic representation of the monitoring approach. It shows a typical split-phase configuration of alternating current (AC) 240 V nominal voltage of power distribution in residential homes in the U.S. In this configuration, power is supplied to various loads inside the NZERTF by two live (hot) wires, designated Line 1 (L1) and Line 2 (L2). A neutral (N) wire carries the current difference between L1 and L2. In an ideal and balanced system, zero current flows through the neutral wire. The split-phase configuration supports 240 V (line-to-line) and 120 V (line-to-neutral) loads. Typically, the L1, L2, and N cables are color-coded as black, red, and white, respectively. Installation of the ground

connection (green cable not shown in the schematic) is a national electric code requirement in buildings.



**Fig. 2.** Schematic representation of the sensors placement for measuring the voltage and current waveforms of various appliances in the NZERTF

A detailed description of the experimental design for measuring the voltage and current waveforms of various appliances, the uncertainty of measurement, and the gain and frequency response of the sensors are documented in [20]. A list of relevant sensor types, maximum ratings, functionalities, and abbreviated designations are shown in Table 1. In this study, the root mean square (RMS) outputs of the sensors on the utility side of the meter, as shown in Fig. 2, were used to model the performance of the air-source heat pump.

**Table 1. Sensor Designations**

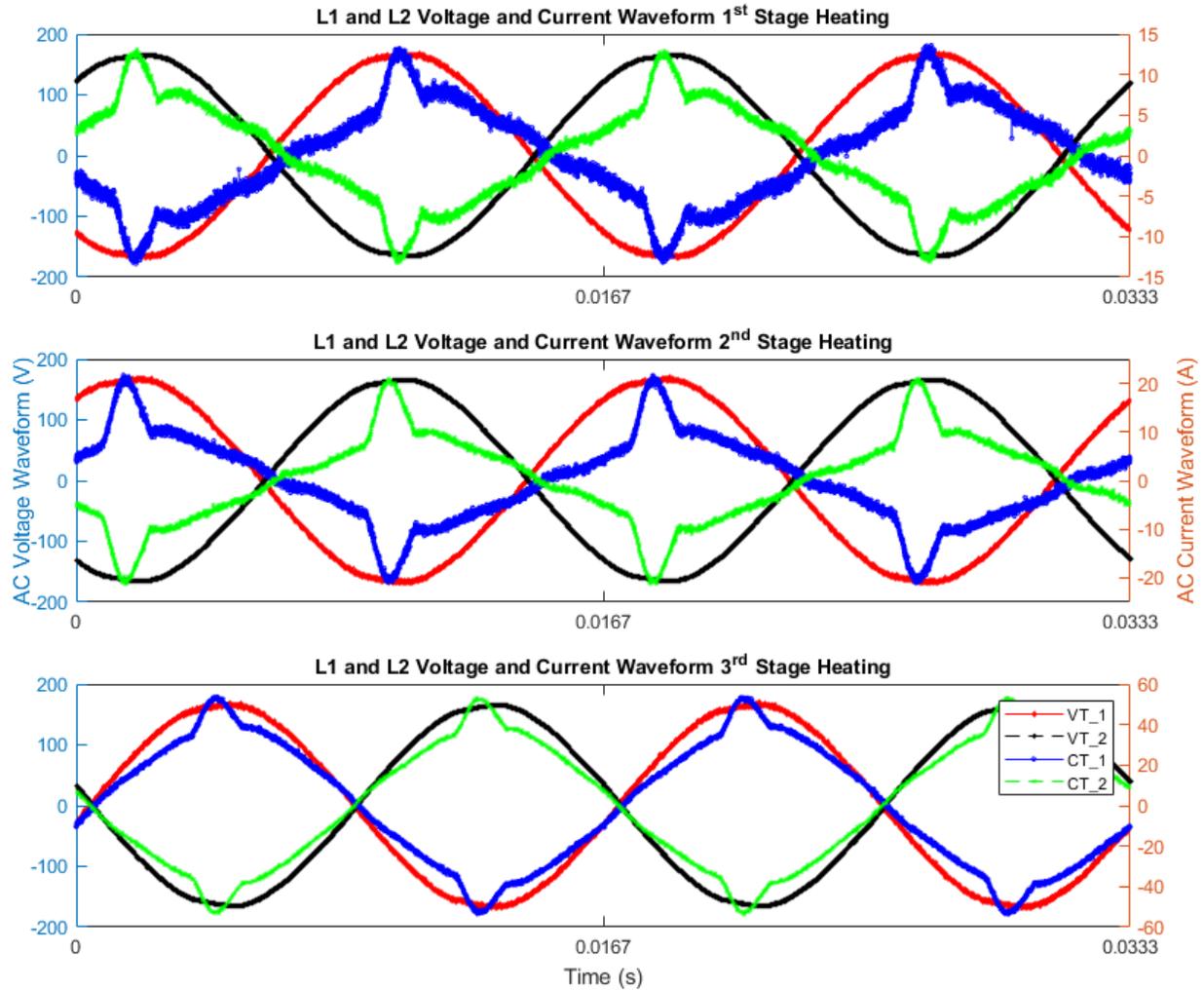
Sensor Designation	Measurement Function	Type Description
CT_1	Current	Magnetic Induction (300 A RMS)
CT_2	Current	Magnetic Induction (120 A RMS)
VT_1	Voltage	Passive voltage divider (300 V RMS)
VT_2	Voltage	Passive voltage divider (300 V, peak)

As shown in Table 1, there are two voltage transducers (VTs) and two clamp-on current transducers (CTs). The outputs of the sensors are scaled analog voltages of the inputs. Current sensors CT\_1 and CT\_2 are magnetic induction transducers. These sensors do not require external power supplies to operate. Similarly, voltage sensors VT\_1 and VT\_2 are single-ended passive voltage dividers that do not require external power supplies. These voltage dividers measure the potential difference between a line and the ground or neutral (N) in buildings. In residential homes,

the ground and neutral wires are directly tied inside the main circuit breaker panel; therefore, all voltage measurements were conducted between lines L1 and L2 and the neutral.

## **2.1. Current and Voltage Waveform of the Air-Source Heat Pump**

The air-source heat pump requires a 240 V AC power supply. It has two main components, an air-handling unit inside the house and an outdoor unit containing a compressor and a coil that serves as an evaporator in the heating mode and as a condenser in the cooling mode. The air-handling unit uses a variable-speed blower to provide conditioned air to all areas of the home. The outdoor unit utilizes a two-stage scroll compressor to provide different heating or cooling capacities. The waveform current consumption of the air-source heat pump for the heating season is shown in Fig. 3. Fig. 3 shows two cycles of the voltage and current waveforms recorded between L1-N and L2-N during the 1<sup>st</sup> Stage, 2<sup>nd</sup> Stage, and 3<sup>rd</sup> Stage of the heat pump operation in the heating mode. In the cooling season, the heat pump's electrical performance data show similar waveform patterns as shown in Fig. 3. However, in the cooling season, the heat pump only operates in the 1<sup>st</sup> Stage or 2<sup>nd</sup> Stage, and there is no auxiliary or 3<sup>rd</sup> Stage. For brevity, the plots for the cooling season are not shown.

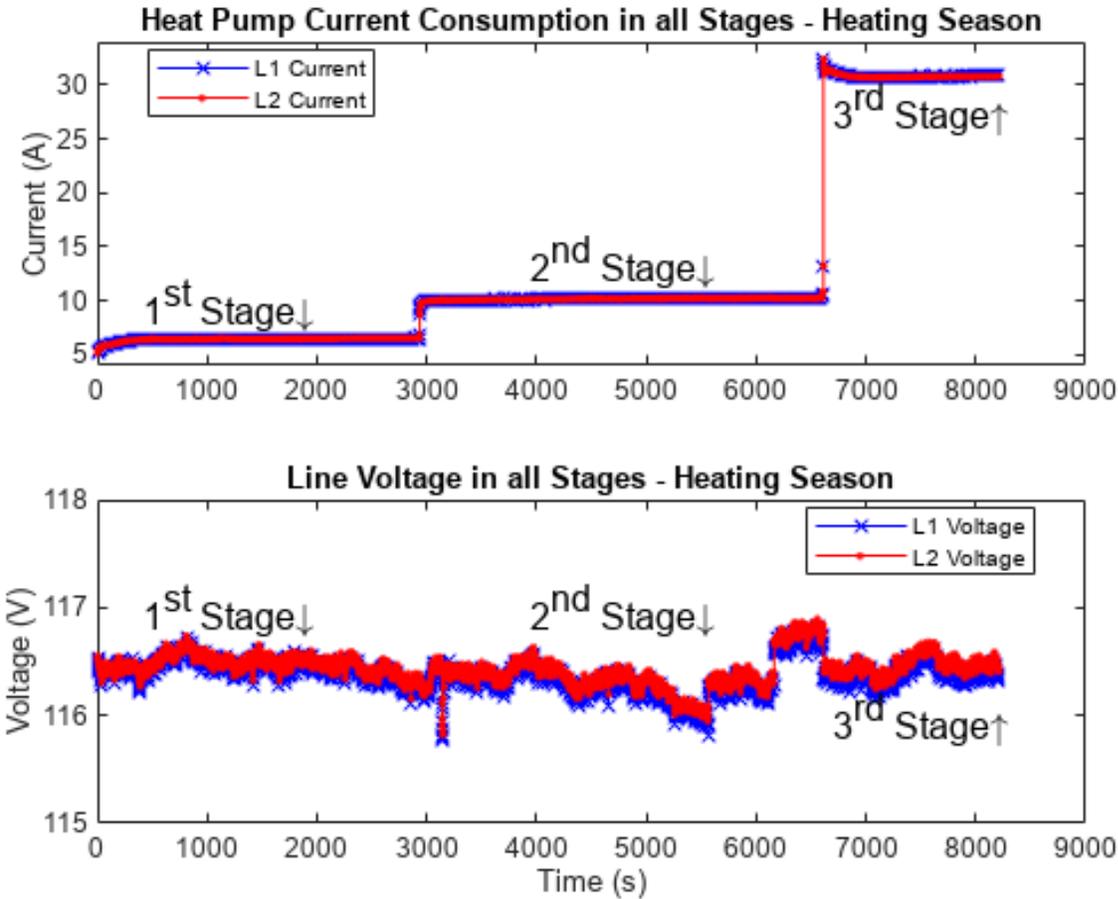


**Fig. 3.** The voltage and current characteristics of the heat pump waveform in the heating season

As shown in Fig. 3, The voltage and current waveforms between L1 and L2 are 180 degrees out of phase. This phase shift is due to the split phase configuration in residential power distribution networks, as the potential difference are measured between lines and the N. The voltage and current consumption of L1 is approximately the mirror image of the voltage and current consumption of L2. The impact of power electronics on the current consumption waveforms is more visible in the 1<sup>st</sup> Stage and 2<sup>nd</sup> Stage of operation, compared to the 3<sup>rd</sup> Stage. In the 3<sup>rd</sup> Stage, the heat pump relies on a 5 kW auxiliary resistive load to provide heating, reducing the impact of the power electronics on the waveform distortion. Further analysis of power quality associated with the heat pump waveform data is beyond the scope of this document.

Generally, it is difficult and computationally expensive to model the performance of various heat pump components for load analysis and load forecasting using waveform data. A simpler approach for developing load models to conduct stability and load flow analysis is to use the RMS values computed from the waveform data. The RMS amplitude and phase angles of the waveforms measured by CT\_1, CT\_2, VT\_1, and VT\_2 sensors on the utility side of the meter were used. The current consumption of the heat pump on the utility side of the meter is the sum of the current

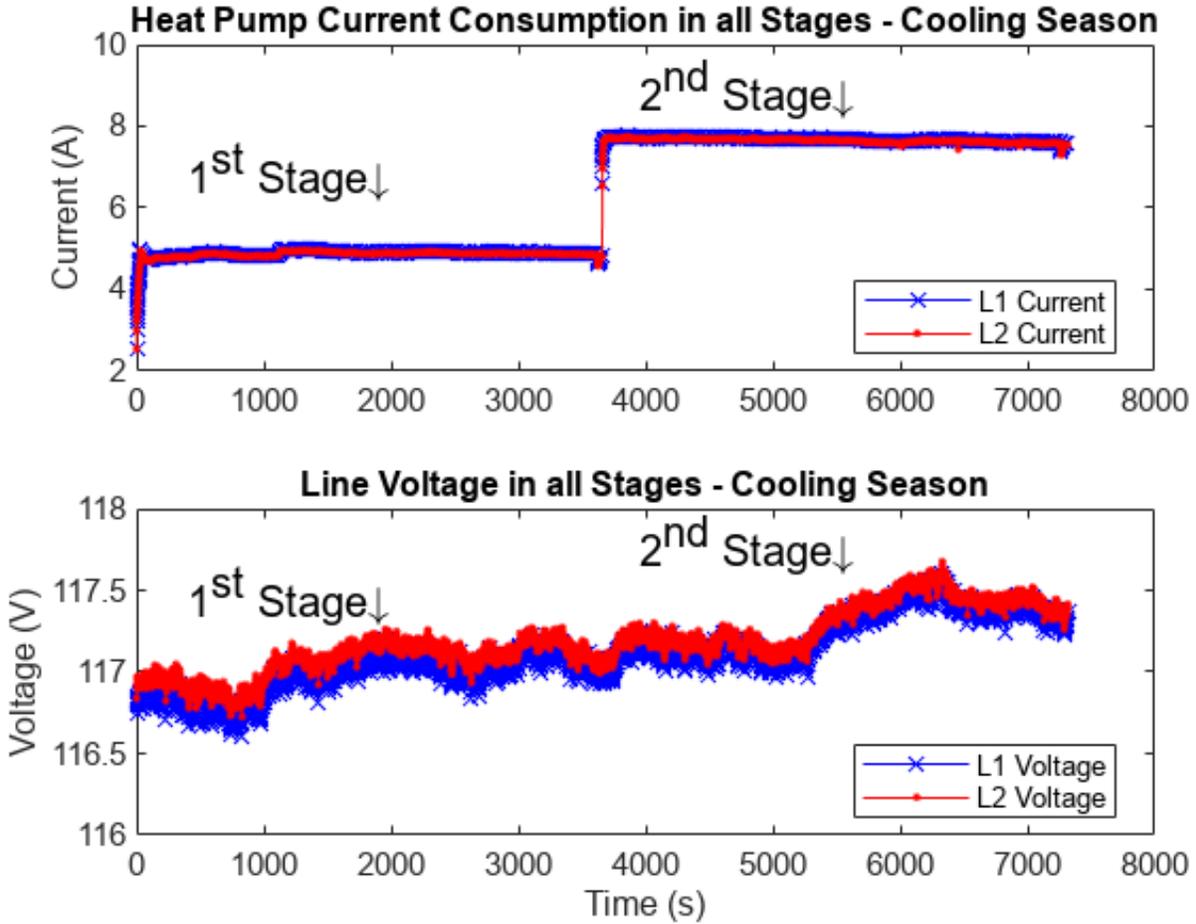
consumption of all individual components of the air-handling and the outdoor units. The Harmonic Parameters Identification algorithm described in [21] was utilized to extract the RMS amplitude and phase angles. The Harmonic Parameters Identification algorithm uses the Hanning window function and interpolated Discrete Fast Fourier Transform to extract the amplitude and phase angles of the voltage and current measurements. Fig. 4 shows the RMS voltage and RMS current consumption of the three stages of the heat pump operating in the heating mode.



**Fig. 4.** The RMS values of voltage and current characteristics of the heat pump in the heating season

As shown in Fig. 4, the minimum RMS current consumption is 5.2 A (1<sup>st</sup> Stage), and the maximum RMS current consumption is 32.5 A (3<sup>rd</sup> Stage). The current consumption of the heat pump in the 3<sup>rd</sup> Stage is the sum of the current consumption of the 2<sup>nd</sup> Stage, and the current consumption resulted from activating a 5 kW resistive load. The minimum RMS voltage is 115.8 V, and the maximum RMS voltage is 116.9 V.

Fig. 5 shows the RMS voltage and RMS current consumption for the two stages of the heat pump operating in the cooling season. In the cooling season, immediately after the 1<sup>st</sup> Stage is activated, the magnitude of the inrush current reaches approximately 44 A, lasting only 1 s. The inrush current is not shown in Fig. 5. The inrush current was not included in the heat pump modeling process because it severely diminished the accuracy of the LA discussed in Sec. 4.1.



**Fig. 5.** The RMS values of voltage and current characteristics of the heat pump in the cooling season

As shown in Fig. 5, the minimum RMS current consumption is 2.5 A (1<sup>st</sup> Stage), and the maximum RMS current consumption is 7.8 A (2<sup>nd</sup> Stage). The minimum RMS voltage is 116.6 V, and the maximum RMS voltage is 117.7 V. The RMS values shown in Fig. 4 and Fig. 5 were used to develop the data-driven RL Model to predict the current consumption of the two-stage heat pump in heating and cooling seasons.

### 3. Resistor-Inductor (RL) Model

Consider a simple first-order RL circuit where the resistor  $R$  and inductor  $L$  elements are in series with each other. The voltage drop across the first-order RL circuit is given by

$$-V_s(t) + i_L(t)R + V_L(t) = 0, \quad (1.1)$$

where,  $V_s(t)$  is the source voltage (V),  $i_L(t)$  is the inductor current (A),  $R$  is the resistor ( $\Omega$ ), and  $V_L(t)$  is the voltage across the inductor element (V). The differential form of Eq. (1.1) is given by

$$\begin{aligned}
 -V_s(t) + i_L(t)R + L \frac{di}{dt} &= 0 \\
 \frac{di}{dt} &= \frac{V_s(t)}{L} - i_L(t) \frac{R}{L},
 \end{aligned}
 \tag{1.2}$$

where, the voltage across the inductor is  $V_L(t) = L \frac{di}{dt}$ . The integration form of Eq. (1.2) is given by

$$\int_{i_L(t_0)}^{i_L(t)} \frac{di}{\left(i_L(t) - \frac{V_s(t)}{R}\right)} = \int_{t_0}^t \frac{-R}{L} dt.
 \tag{1.3}$$

Integrating Eq. (1.3) and solving for  $i_L(t)$  provides the time dependent inductor current across the RL circuit given by

$$i_L(t) = \left(i_L(t_0) - \frac{V_s(t)}{R}\right) e^{-\frac{R}{L}(t-t_0)} + \frac{V_s(t)}{R}.
 \tag{1.4}$$

The inductor current described by Eq. (1.4) has a transient component and a steady-state term. The transient component is shown in Eq. (1.5),

$$\left(i_L(t_0) - \frac{V_s(t)}{R}\right) e^{-\frac{R}{L}(t-t_0)}.
 \tag{1.5}$$

After some time has elapsed the transient part of Eq. (1.4) decays and the current consumption is dominated by the steady-state term of Eq. (1.4). The steady state current ( $I_L(t)$ ) is given in Eq. (1.6),

$$I_L(t) = \frac{V_s(t)}{R}.
 \tag{1.6}$$

In a typical energy analysis approach, the steady-state power consumption of a heat pump is linearly dependent on the outdoor temperature [11, 22]. Consequently, the instantaneous steady-state power consumption of the heat pump is expressed as a first-order linear model

$$P_{hp}(t) = m \times T_\infty(t) + b,
 \tag{1.7}$$

where,  $m$  is the slope of the linear model (W/°C),  $T_\infty(t)$  is the outdoor dry bulb temperature (°C), and  $b$  is the intercept (W). Alternatively, the instantaneous power consumption of an electrical circuit can be calculated by

$$P = V_{rms} \times I_{rms} \times \cos(\theta),
 \tag{1.8}$$

where,  $\cos(\theta)$  is the displacement power factor. Substituting  $I_L(t)$  for  $I_{rms}$  and  $V_s(t)$  for  $V_{rms}$  in Eq. (1.8) give us

$$P_L(t) = V_s(t) \times I_L(t) \times \cos(\theta). \quad (1.9)$$

$P_L(t)$  is the time dependent (at each time step) instantaneous steady-state power consumption as function of the source voltage  $V_s(t)$  and steady-state current consumption of the RL circuit.

Assuming a unity power factor and neglecting the impact of the distortion power factor, solving Eq. (1.9) for the steady-state current  $I_L(t)$  give us the following relationship

$$I_L(t) = \frac{P_L(t)}{V_s(t)}. \quad (1.10)$$

Furthermore, substituting  $P_{hp}(t)$  (given in Eq. (1.7)) for the value of  $P_L(t)$  (given in Eq. (1.10)) enable us to express the steady-state current consumption of the heat pump as a function of  $T_\infty(t)$  and line voltage  $V_s(t)$ . The updated steady-state current consumption, considering the temperature dependency of the heat pump's power consumption, is given by

$$I_L(t) = \frac{P_{hp}(t)}{V_s(t)} = \frac{m \times T_\infty(t) + b}{V_s(t)}. \quad (1.11)$$

The objective is to derive a mathematical expression for capturing the current consumption of each stage of the two-stage heat pump not only as a function of the voltage and displacement power factor but also as a function of the outdoor temperature. Let's assume that Eq. (1.4) provides the foundation for this mathematical expression to predict the heat pump current consumption. Substituting Eq. (1.11) into the steady-state term of Eq. (1.4) give us

$$i_L(t) = \left( i_L(t_0) - \frac{V_s(t)}{R} \right) e^{-\frac{R}{L}(t-t_0)} + \frac{m \times T_\infty(t) + b}{V_s(t)}. \quad (1.12)$$

If the assumption about Eq. (1.4) holds true, then Eq. (1.12) can model the voltage and outdoor temperature dependent performance of the heat pump in both transient and steady-state conditions.

Let  $\tau = L/R$  represent the overall time constant (in seconds) of a particular stage of the heat pump, then Eq. (1.12) can be written as

$$i_L(t) = \left( i_L(t_0) - \frac{V_s(t)}{R} \right) e^{-\frac{(t-t_0)}{\tau}} + \frac{m \times T_\infty(t) + b}{V_s(t)}. \quad (1.13)$$

The value of  $\tau$  (s) in Eq. (1.13) is the time it takes for a particular stage of the heat pump to reach its steady-state current consumption. Understanding the value of  $\tau$  is a critical information for analyzing the aggregated effect of controlling many heat pumps, as flexible loads, on grid stability and provision of ancillary services. The larger the value of the  $\tau$ , the slower the response of the

heat pump would be in reaching its steady-state current consumption. Consequently, the values of real and reactive power consumptions, reaching their steady state, reflect the influence of  $\tau$ . Using Eq. (1.13) to predict the heat pump current consumption requires a data-driven approach because certain parameters must be estimated from measurement data.

The values of  $V_s(t)$  and  $T_\infty(t)$  are known from measurement for all  $t = 1, 2, \dots, T$  (s), where  $T$  is the length of time in seconds equivalent to the number of measured data points, assuming each data point is 1 s. However, the values of  $R, L, m$ , and  $b$  are not known a priori. A learning algorithm (LA) is used to estimate these values from measured data. In this paper these unknown values are denoted as effective quantities ( $R_e, L_e, m_e, b_e$ ) because they are a lumped approximation of the true behavior of the heat pump equipment. These approximations will enable us to predict the current consumption of the heat pump.

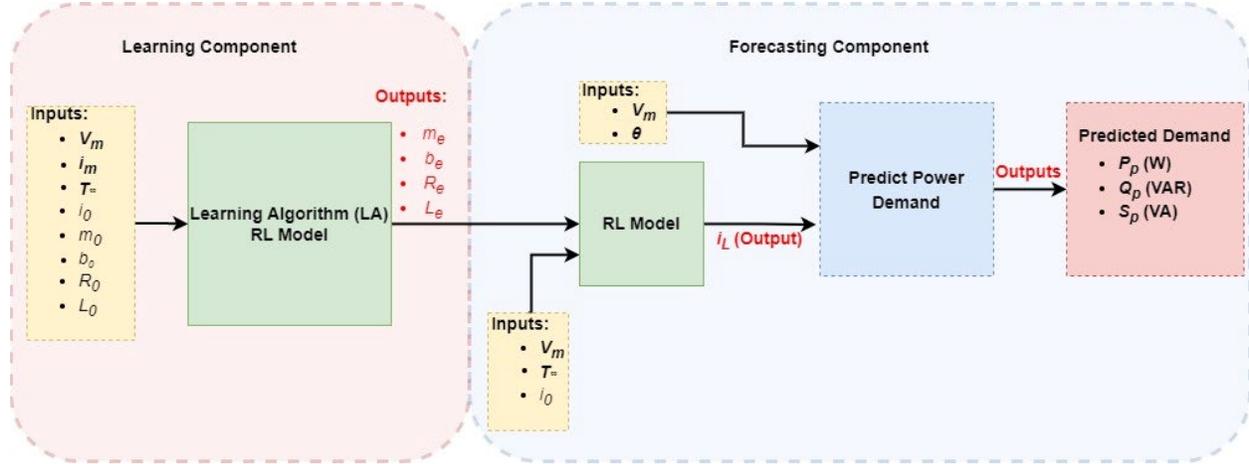
A discrete form of Eq. (1.13) is developed by defining  $t - t_0$  as  $\Delta t = t_{k+1} - t_k$  where  $k = 0, 1, 2, \dots, n$  are the discrete time steps and  $n$  is the number of data points. Let  $i_L(k)$  represent the discrete current consumption, the one-step learning and current prediction model is

$$i_{L,k+1} = \left( i_{L,k} - \frac{V_{s,k}}{R_e} \right) e^{-\frac{\Delta t}{\tau_e}} + \frac{m_e \times T_{\infty,k} + b_e}{V_{s,k}}, \quad (1.14)$$

where  $\tau_e = L_e/R_e$  in (s) is the effective time constant.

#### 4. Learning and Forecasting Schema

The task of developing a data-driven model to predict the current consumption of the heat pump is separated into learning and forecasting components. Fig. 6 shows a schematic representation of the data flows through the learning and forecasting components, leading to predicting power demand. As shown in Fig. 6, the RL Model given by Eq. (1.14) is common to both the Learning Component and the Forecasting Component. The Learning Component is utilized to learn effective parameters of the RL Model, representing various operating stages of the heat pump in both heating and cooling seasons. Using these effective parameters, the Forecasting Component is utilized to predict the current consumption and, ultimately, the power demand of the heat pump in various operating stages and different seasons.



**Fig. 6.** A schematic representation of the Learning Component and Forecasting Component utilized to learn effective parameters and predict power consumption demand

As shown in Fig. 6, the learning algorithm (LA) utilizes a set of inputs described in Table 2 to estimate the values of the effective parameters  $m_e$ ,  $b_e$ ,  $R_e$ , and  $L_e$ . The LA estimates these effective parameters by reducing the error between the output of the RL Model and the measured RMS current consumption of the heat pump  $i_m$ . The Forecasting Component utilizes these effective parameters, the measured RMS voltage  $V_m$ , outdoor temperature  $T_\infty$ , and the initial starting point  $i_0$  to predict the current consumption of the heat pump  $i_L$ . Similarly, using  $i_L$ ,  $V_m$ , and the phase angle difference between the voltage and current waveforms  $\theta$ , the Forecasting Component predicts the real ( $P_p$ ), reactive ( $Q_p$ ), and apparent power ( $S_p$ ) demand of the heat pump. The modular approach between learning and forecasting enables us to test the LA with new data sets and evaluate the impact of the changes in voltage and outdoor temperature on predicting load demand.

**Table 2. List of Inputs Used in the Learning and Forecasting Methodologies**

Inputs	Description
$V_m$	A vector of RMS voltage calculated from measured voltage waveform $V_s(t)$ in every second (V). The RMS voltages for the heating and cooling seasons are shown in the bottom images of Fig. 4 and Fig. 5, respectively.
$i_m$	A vector of RMS current calculated from measured current waveform $i_L(t)$ in every second (A). The RMS current for the heating and cooling seasons are shown in the top images of Fig. 4 and Fig. 5, respectively.
$T_\infty$	A vector of the outdoor dry bulb temperature in every second obtained from interpolating minutely measured outdoor temperature data ( $^\circ\text{C}$ ).
$i_0$	Initial current value obtained from $i_m(0)$ (A)
$m_0$	Initial value of the slope of the linear model given in Eq. (1.7) ( $\text{W}/^\circ\text{C}$ )
$b_0$	Initial value of the intercept of the linear model given in Eq. (1.7) (W)
$R_0$	Initial value of the resistor in the RL Model calculated by dividing $\frac{V_m(0)}{i_m(0)}$ ( $\Omega$ )
$L_0$	Initial value of the inductor, arbitrary selected (H)
$\theta$	A vector of the phase angle differences between the voltage and current waveforms computed every second $\theta = (\theta_v - \theta_i)$ (rad)

The initial values of  $i_0$ ,  $R_0$ ,  $L_0$ ,  $m_0$ , and  $b_0$  are given in Table 3. The mechanism for obtaining the values of  $i_0$ ,  $R_0$ , and  $L_0$  were provided in Table 2. The initial values for  $m_0$ , and  $b_0$ , describing the slope and intercept of Eq. (1.7) were obtained from performance of the two-stage air-source heat pump in the NZERTF [23].

**Table 3. Initial Values of Current and Effective Parameters Used in LA**

Phase	Season	Stages	$i_0(A)$	$R_0(\Omega)$	$L_0(H)$	$m_0(W/^\circ C)$	$b_0(W)$
L1-N & L2-N	Heat	1 <sup>st</sup> Stage	5.2	22.6	1.0E-06	7.0	1096.8
		2 <sup>nd</sup> Stage	6.5	17.8	1.0E-06	6.7	1747.6
		3 <sup>rd</sup> Stage	32.5	3.6	1.0E-06	6.7	1747.6
	Cool	1 <sup>st</sup> Stage	2.5	45.9	1.0E-06	16.3	-105.0
		2 <sup>nd</sup> Stage	4.8	24.2	1.0E-06	19.3	438.6

#### 4.1. The Learning Algorithm (LA)

To predict  $i_L$ , the effective parameters  $\tau_e$ ,  $m_e$ , and  $b_e$  of the RL Model, given by Eq. (1.14), were estimated using a curve fitting optimization technique. The objective function of the LA was defined as least-squares error ( $l_2$  norm) between the measured current consumption of the heat pump ( $i_m$ ) and the predicted current ( $i_L$ ). The objective function is given by

$$f(R_e, L_e, m_e, b_e) = \|i_m - i_L\|_2, \text{ and} \quad (1.15)$$

the optimization problem is

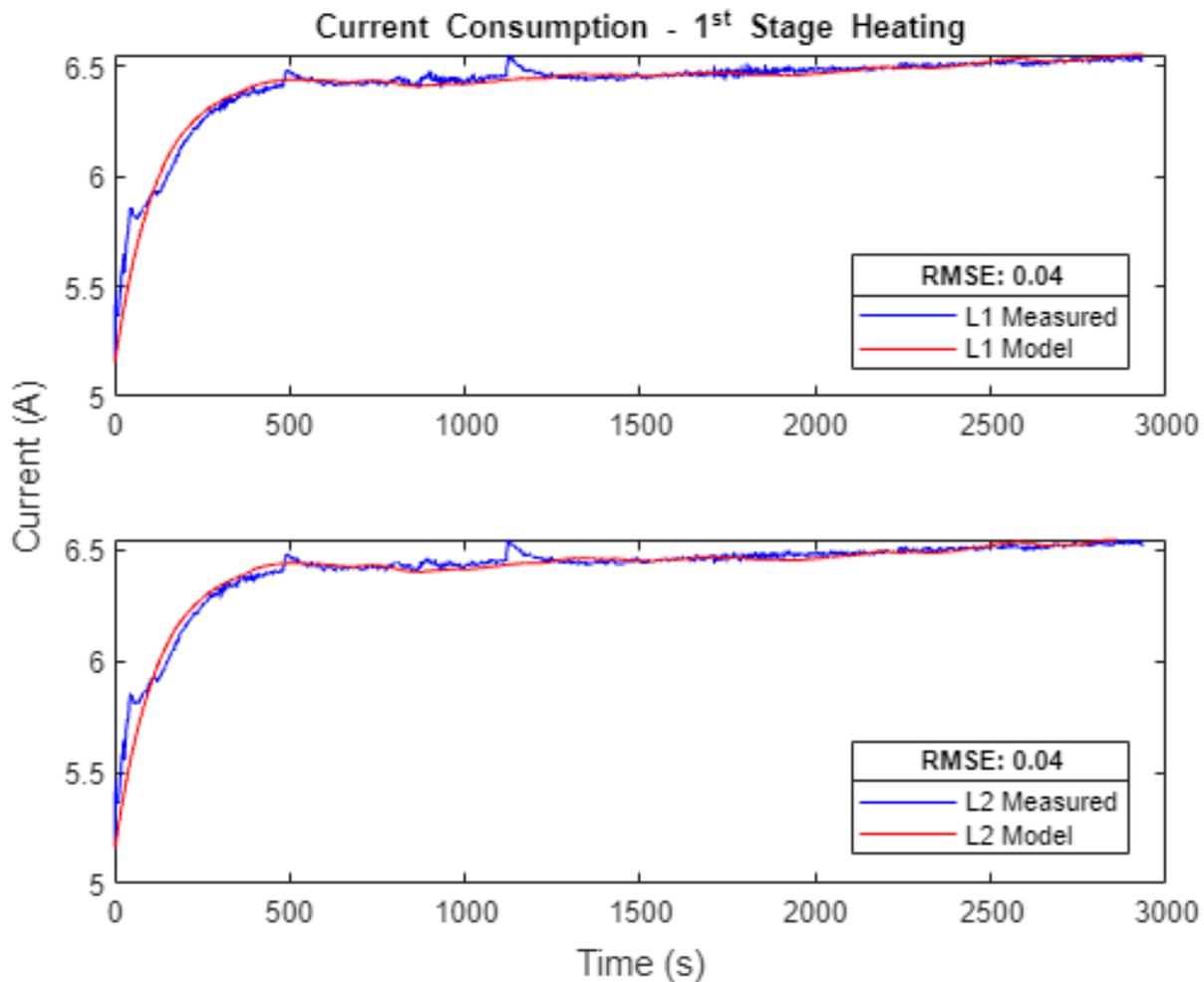
$$\begin{aligned} \min_{R_e, L_e, m_e, b_e} f(R_e, L_e, m_e, b_e) \\ lb \leq R_e \leq ub \\ lb \leq L_e \leq ub \\ lb \leq m_e \leq ub \\ lb \leq b_e \leq ub, \end{aligned} \quad (1.16)$$

where, all effective parameters are constrained within a lower  $lb = 1e - 6$  and upper bounds of  $ub = 1e5$ . The choices for  $lb$  and  $ub$  were arbitrary selected for numerical convergence and stability.

The choice of initial values (starting points) for the effective parameters  $m_e$ ,  $b_e$ ,  $R_e$ , and  $L_e$  could significantly impact the LA performance and the RL Model's output quality. These choices affect the convergence of the LA to an optimum solution because the function given in Eq. (1.16) can have multiple local minimum values satisfying the non-linear objective function given in Eq. (1.15). To address these issues, the MATLAB's MultiStart optimization algorithm was used to evaluate up to fifty randomly generated starting points. For reproducibility of random numbers, the random number generator in the MATLAB R2022b release was set to its default settings (`rng default`). In the MultiStart routine, the `fmincon` algorithm was selected as the local solver.

The local solver algorithm was prevented from evaluating infeasible starting points by setting the `StartPointsToRun` option of `MultiStart` algorithm to bounds.

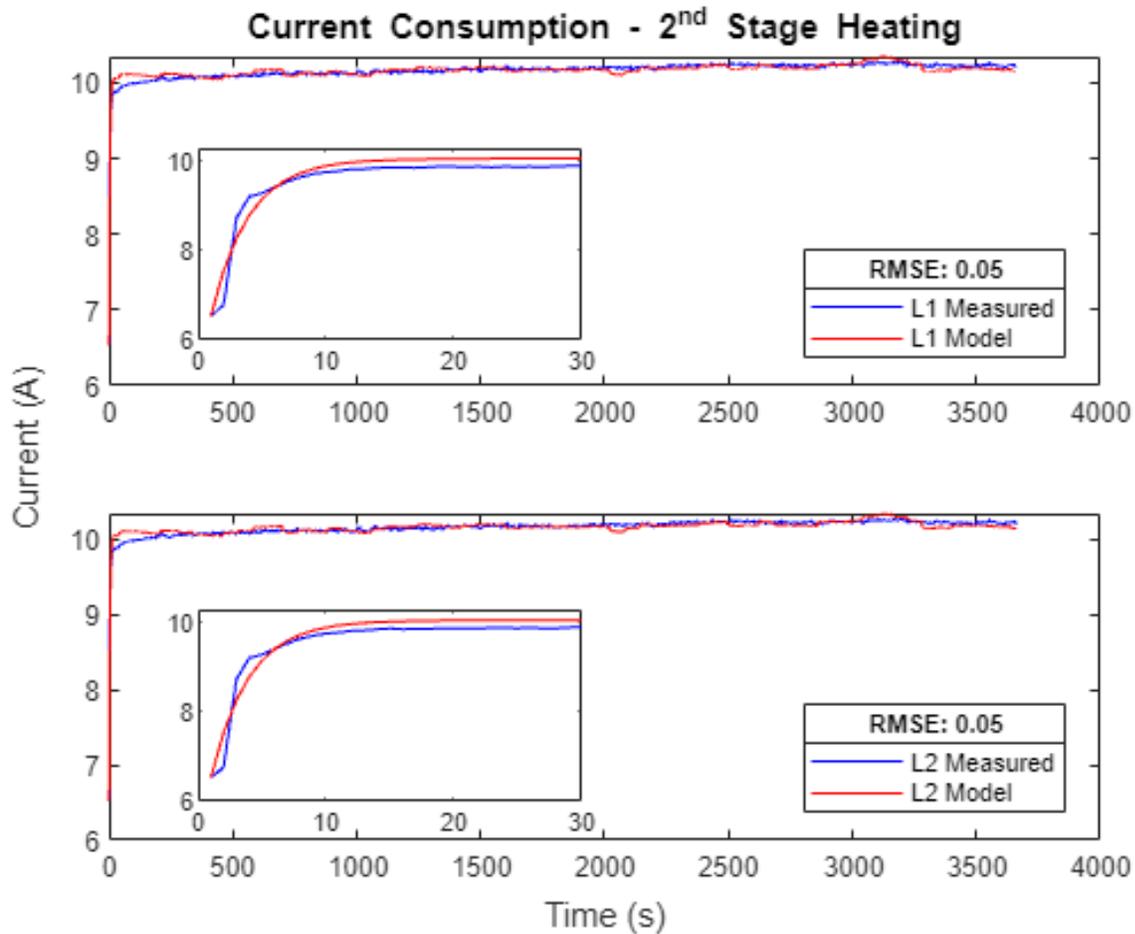
The prediction capability of the LA for modeling the current consumption of the two-stage heat pump, three cases shown in Fig. 7, Fig. 8, and Fig. 9 were selected. These cases provide a comparison between the measured ( $i_m$ ) and predicted ( $i_L$ ) current consumption of the heat pump for the 1<sup>st</sup> Stage, 2<sup>nd</sup> Stage, and 3<sup>rd</sup> Stage of heating. All cases include the plot for L1 and L2 phases in the heating season. The current consumption on L1 and L2 were measured at nominal 116 V AC. The LA minimized the least-squares differences ( $l_2$  norm) between the output of the model and the measured current values. To evaluate the merit of the RL Model, the root mean square error (RMSE) statistical method was used. The RMSE of individual fit for each stage of the heat pump and L1 and L2 phases are reported in their respective plots.



**Fig. 7.** Measured versus predicted current consumption of the heat pump on L1 and L2 phases during the 1<sup>st</sup> Stage of heating

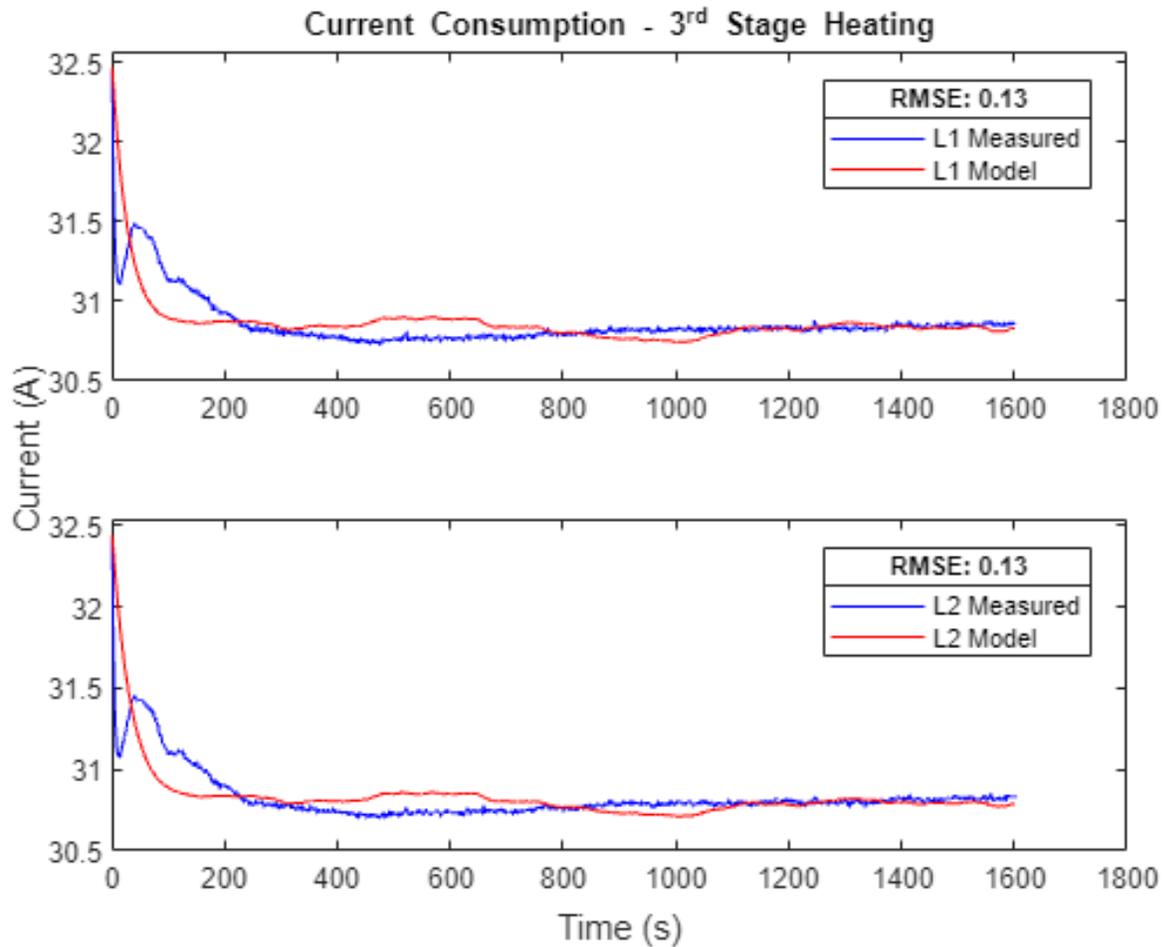
As shown in Fig. 7, the heat pump's response to reach 63.2 % of its steady-state current consumption in 125 s given by the effective time constant  $\tau_e$  in Table 4. In inductive or capacitive loads, five-time constants ( $5\tau_e$ ) provide a good approximation for reaching within 1 % of the

steady-state value of the circuits. In contrast, the effective time constant  $\tau_e$  for the 2<sup>nd</sup> Stage, shown in Fig. 8, is 3 s. In this study, the 2<sup>nd</sup> Stage of the heat pump was activated while the heat pump was already operating in its 1<sup>st</sup> Stage as shown in the top plot of Fig. 4.



**Fig. 8.** Measured versus predicted current consumption of the heat pump on L1 and L2 phases during the 2<sup>nd</sup> Stage of heating

As shown in Fig. 7 and Fig. 8, there are good agreements between the measured current consumption and predicted current consumption in both transient and steady-state conditions. For visual clarity, embedded subplots in Fig. 7 show the first 30 s of the heat pump’s current consumption in the 2<sup>nd</sup> Stage of operation.



**Fig. 9.** Measured versus predicted current consumption of the heat pump on L1 and L2 phases during the 3<sup>rd</sup> Stage of heating

In contrast to Fig. 7 and Fig. 8, the transient behavior of the heat pump operating in the 3<sup>rd</sup> Stage exhibits an exponential decay, as shown in Fig. 9. The effective time constant  $\tau_e$  in the 3<sup>rd</sup> Stage is 28 s. As shown in Fig. 9, the RL Model cannot capture the exact behavior of the heat pump during the transient period; however, it provides a good approximation for the current consumption as validated by the lower values of RMSE. The 3<sup>rd</sup> Stage of the heat pump was activated while the heat pump was already operating in its 2<sup>nd</sup> Stage as shown in the top plot of Fig. 4.

The prediction capability of the LA for modeling the current consumption of the two-stage heat pump during the cooling season exhibits similar patterns and low RMSE values, as shown in Fig. 7, Fig. 8, and Fig. 9 for the heating season. The plots for the 1<sup>st</sup> Stage and 2<sup>nd</sup> Stage operating of the heat pump in the cooling season are shown in Appendix A of this document. A summary of estimating effective parameters, time constants, and RMSE values for all modeling activities in this study are provided in Table 4.

#### 4.1.1. Estimated Effective Parameters, Time Constants, and RMSE

Application of the LA resulted in a set of effective parameters for the RL Model, representing all stages of operation and various seasons. A summary of effective parameters and the calculated time constants for each stage of the heat pump is shown in Table 4. Table 4 also shows the calculated values of RMSE between measured and predicted values.

**Table 4. The Values of Effective Parameters, Time Constants, and RMSE, Resulting from the Application of the LA**

Phase	Season	Stages	$R_e(\Omega)$	$L_e(H)$	$m_e(W/^\circ C)$	$b_e(W)$	$\tau_e(s)$	RMSE
L1-N	Heat	1 <sup>st</sup> Stage	727.8	90817.0	0.1	23.4	124.8	0.04
		2 <sup>nd</sup> Stage	5096.8	14984.4	6.9	194.5	2.9	0.05
		3 <sup>rd</sup> Stage	199.3	5570.8	0.0	191.7	28.0	0.13
	Cool	1 <sup>st</sup> Stage	8598.9	99992.1	0.0	48.6	11.6	0.05
		2 <sup>nd</sup> Stage	29.5	45.2	13.7	373.6	1.5	0.03
L2-N	Heat	1 <sup>st</sup> Stage	617.4	78434.3	0.1	26.6	127.0	0.04
		2 <sup>nd</sup> Stage	5215.1	15307.2	7.0	192.8	2.9	0.05
		3 <sup>rd</sup> Stage	267.2	8713.2	0.0	157.5	32.6	0.13
	Cool	1 <sup>st</sup> Stage	8728.1	99991.4	0.0	49.1	11.5	0.05
		2 <sup>nd</sup> Stage	26.4	40.5	13.7	401.5	1.5	0.03

#### 4.2. Predicting Real and Reactive Power Consumption

The equations for calculating the real power (active power) and reactive power consumption of the heat pump are given by

$$\begin{aligned}
 P_p &= V_m \times i_{m,L} \times \cos(\theta), \text{ and} \\
 Q_p &= V_m \times i_{m,L} \times \sin(\theta),
 \end{aligned}
 \tag{1.17}$$

where:

$V_m$  is the measured RMS voltage (V),

$i_{m,L}$  is the measured current  $i_m$  or the predicted current  $i_L$  (A),

$\theta = \theta_v - \theta_i$  is the phase angle difference between the voltage and current waveforms (rad).

The procedure for obtaining the voltage phase angle  $\theta_v$  and the current phase angles  $\theta_i$  were described in Sec. 2.1.,

$P_p$  is the real power (W),

$Q_p$  is the reactive power (VAR).

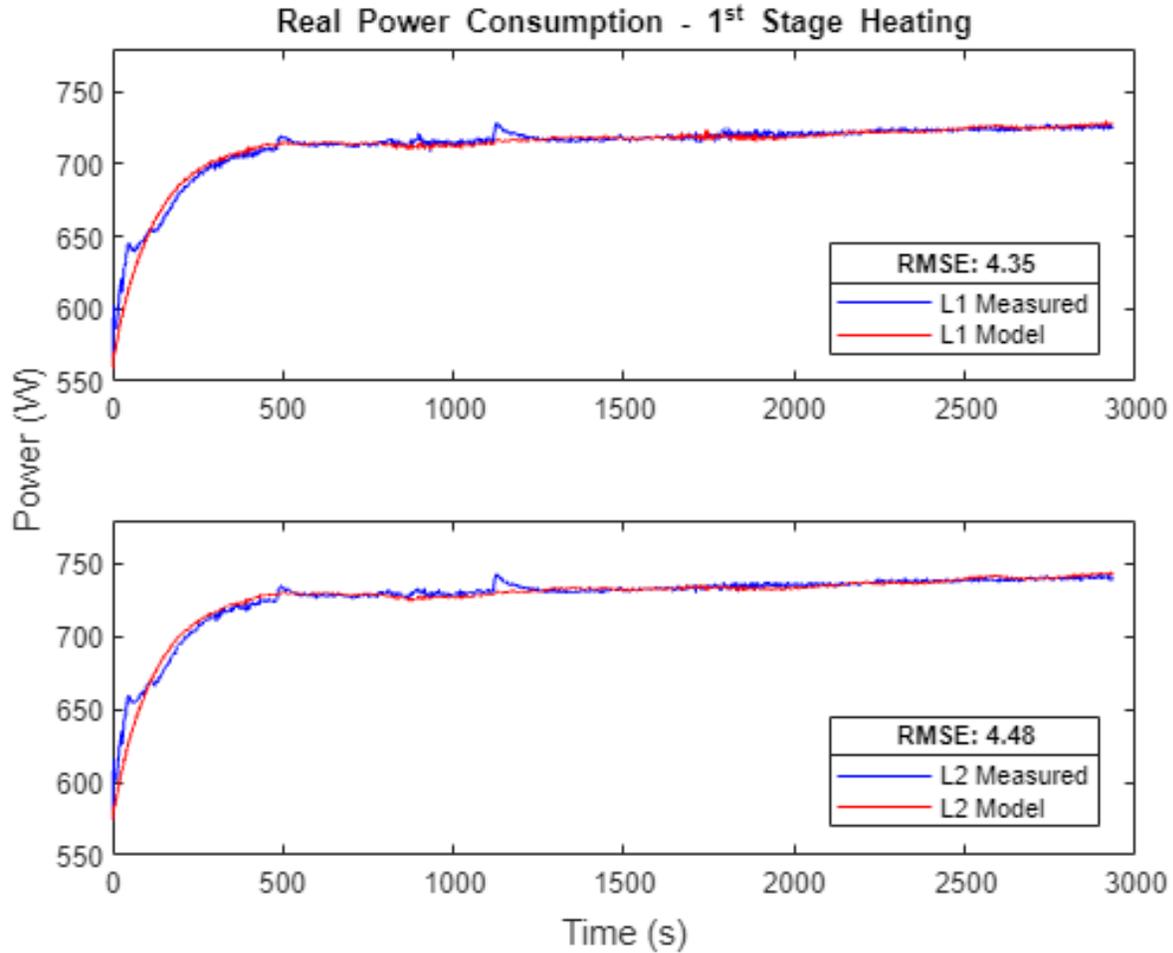
The Forecasting Component, as shown in Fig. 6, uses  $i_L$  in Eq. (1.17) to calculate the predicted real, reactive, and apparent power consumption of the heat pump. Whereas  $i_m$  in Eq. (1.17) was used to calculate the measured real and reactive power consumption of the heat pump. The apparent power consumption can be calculated by

$$S_p = \sqrt{(P_p)^2 + (Q_p)^2}, \quad (1.18)$$

where, the apparent power  $S_p$  is in (VA), and  $P_p$  and  $Q_p$  are described in Eq. (1.17).

#### 4.2.1. Real Power Consumption

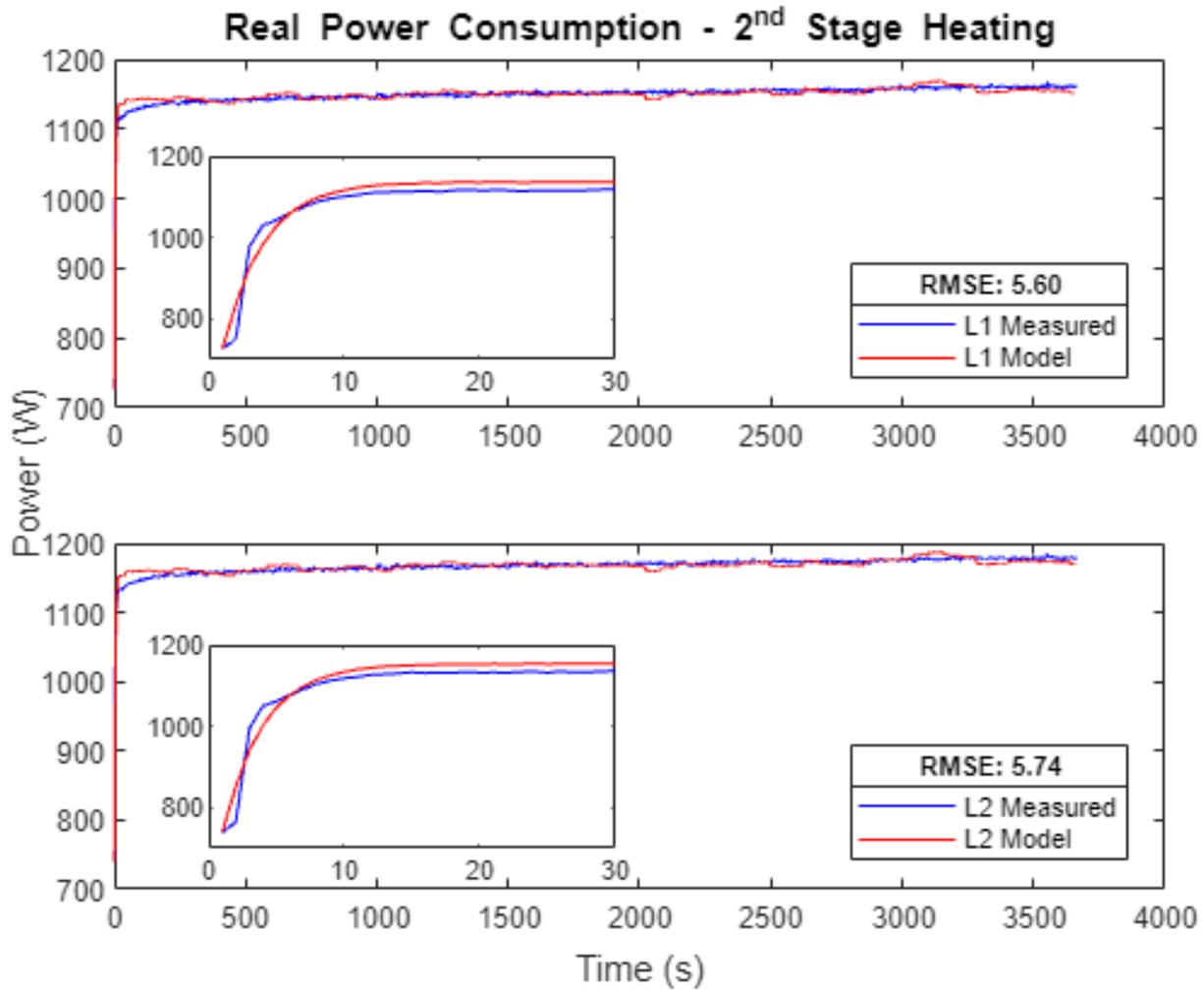
A comparison between the measured and predicted real power consumption of the heat pump during the 1<sup>st</sup> Stage of heating is shown Fig. 10.



**Fig. 10.** Measured versus predicted real power consumption ( $P_p$ ) of the heat pump on L1 and L2 phases during the 1<sup>st</sup> Stage of heating

As shown in Fig. 10, the predicted real power consumption ( $P_p$ ) of the 1<sup>st</sup> Stage of heating closely follows the shape of the current consumption in the 1<sup>st</sup> Stage of heating, as shown in Fig. 7. The RMSE values for the predicted versus measured  $P_p$  is higher than the RMSE values reported in Fig. 7 for the predicted current consumption  $i_L$ . This increase in the RMSE values is expected because the output of the RL Model, given by Eq. (1.14), is independent of the phase angle differences and their impact on its forecasting capabilities.

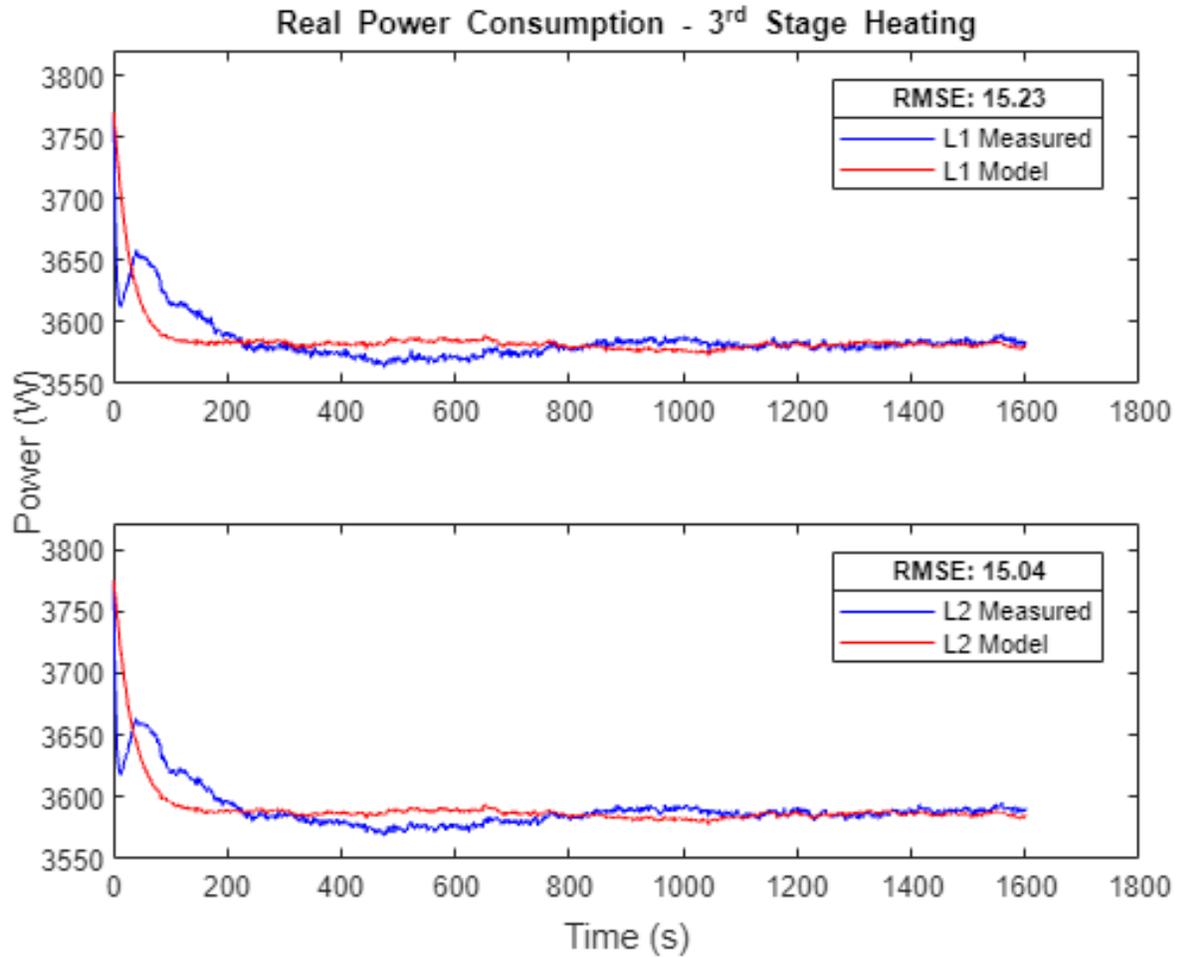
A comparison between the measured and predicted real power consumption of the heat pump during the 2<sup>nd</sup> Stage of heating is shown Fig. 11.



**Fig. 11.** Measured versus predicted real power consumption ( $P_p$ ) of the heat pump on L1 and L2 phases during the 2<sup>nd</sup> Stage of heating

As shown in Fig. 11, the predicted real power consumption ( $P_p$ ) of the 2<sup>nd</sup> Stage of heating also closely follows the shape of the current consumption in the 2<sup>nd</sup> Stage of heating, as shown in Fig. 8. The RMSE values for the predicted versus measured  $P_p$  is higher than the RMSE values reported in Fig. 8 for  $i_L$ . For visual clarity, the embedded subplots in Fig. 11 show the first 30 s of the heat pump's power consumption in the 2<sup>nd</sup> Stage of operation.

A comparison between the measured and predicted real power consumption of the heat pump during the 3<sup>rd</sup> Stage of heating is shown Fig. 12.



**Fig. 12.** Measured versus predicted real power consumption ( $P_p$ ) of the heat pump on L1 and L2 phases during the 3<sup>rd</sup> Stage of heating

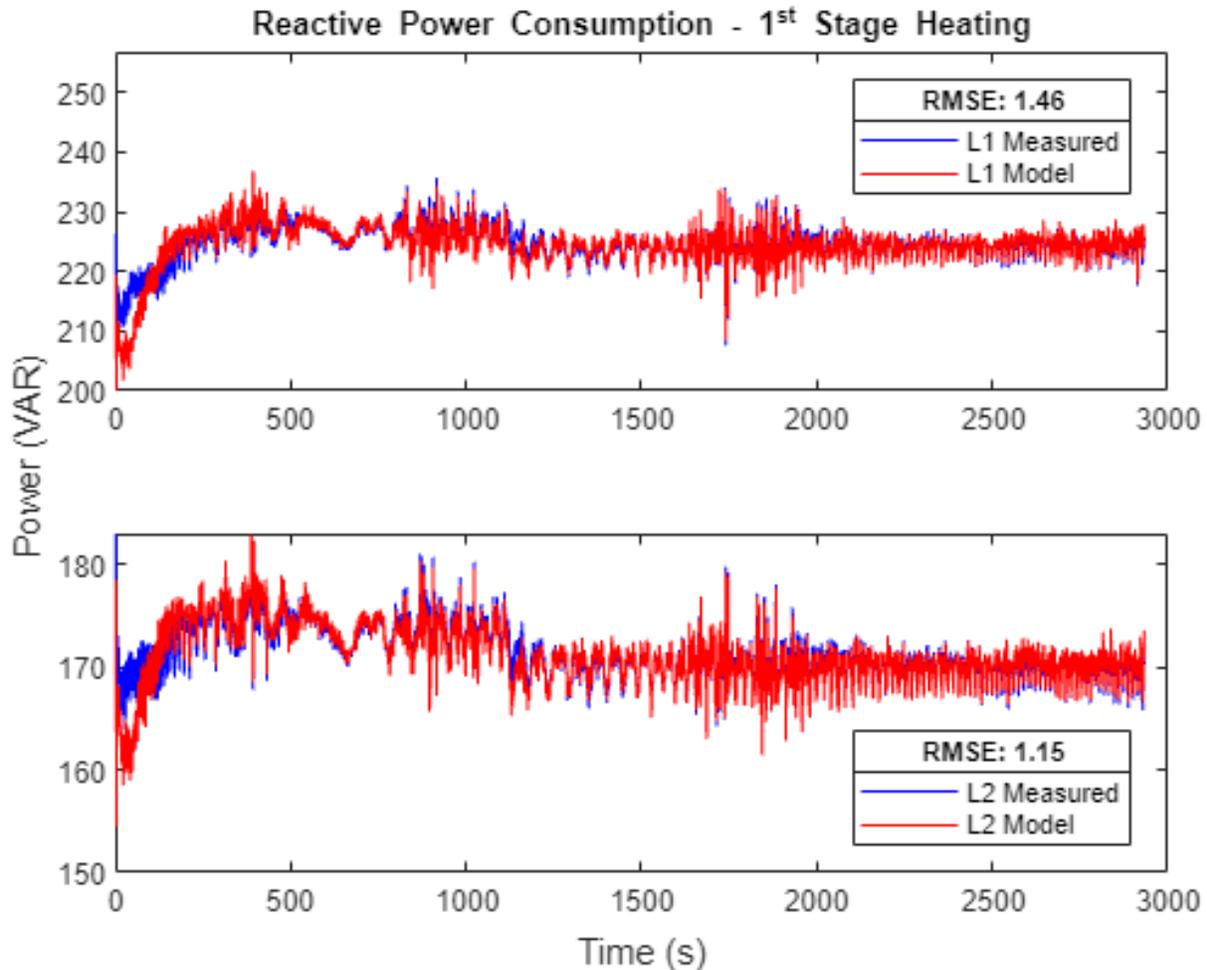
As shown in Fig. 12, the predicted real power consumption ( $P_p$ ) of the 3<sup>rd</sup> Stage of heating also closely follows the shape of the current consumption in the 3<sup>rd</sup> Stage of heating, as shown in Fig. 9. The RMSE values for the predicted versus measured  $P_p$  is much higher than the RMSE values reported in Fig. 9 for  $i_L$ . The RMSE values for the 3<sup>rd</sup> Stage, as shown in Fig. 12, are approximately three times higher than the RMSE values for the 1<sup>st</sup> Stage and 2<sup>nd</sup> Stage of operation. This noticeable increase in the prediction error for 3<sup>rd</sup> Stage power consumption can be attributed to the limitation of the RL Model in capturing the exact behavior of the heat pump during the transient period and the impact of the phase angle difference  $\theta$ . In contrast to the 1<sup>st</sup> Stage and 2<sup>nd</sup> Stage heating, the initial power consumption of the 3<sup>rd</sup> Stage is much higher than its steady-state values. The initial transient portion of the 3<sup>rd</sup> Stage quickly decays to a certain value just above the steady-state power consumption; however, it rises again by a certain value and then starts its slower exponential decay towards the steady-state power consumption.

Knowing the behavior of the transient portion of the heat pump's real power consumption could be consequential for aggregators and system operators looking to develop control strategies for

providing ancillary services. The plots representing the real power consumption of the heat pump in the cooling season are provided in Appendix B of this document.

#### 4.2.2. Reactive Power Consumption

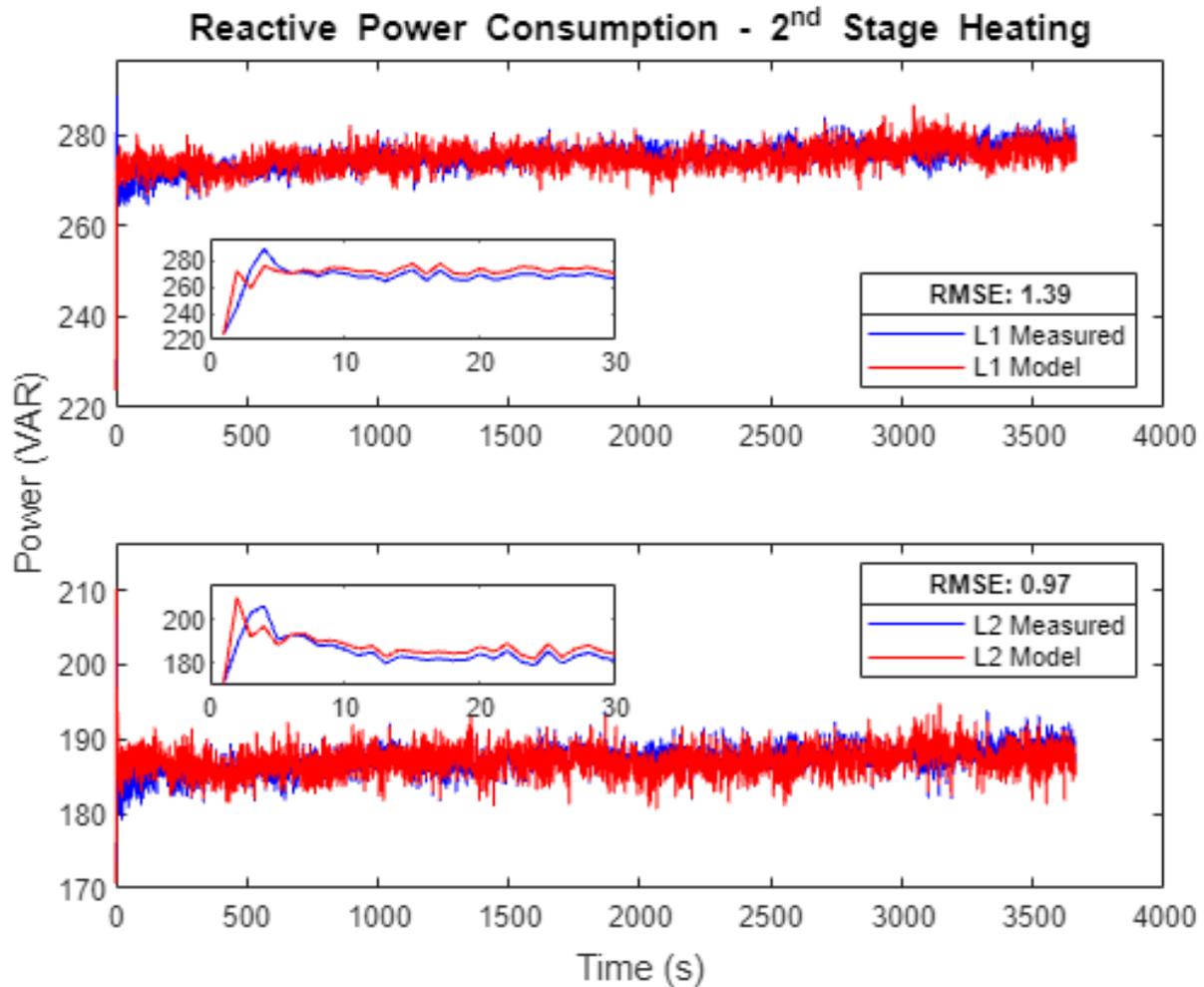
A comparison between the measured and predicted reactive power consumption of the heat pump during the 1<sup>st</sup> Stage of the heating Fig. 13.



**Fig. 13.** Measured versus predicted reactive power consumption ( $Q_p$ ) of the heat pump on L1 and L2 phases during the 1<sup>st</sup> Stage of heating

As shown in Fig. 13, the predicted reactive power consumption closely matches the measured reactive power. The reactive power on L1 and L2 phases exhibit non-linear exponential behavior during the transient part of the 1<sup>st</sup> Stage operation. The traces in Fig. 13 show more fluctuations than the traces shown for the 1<sup>st</sup> Stage real power consumption in Sec. 4.2.1. These fluctuations may be due to the rapid charging and discharging of the inductive and capacitive elements of the heat pump.

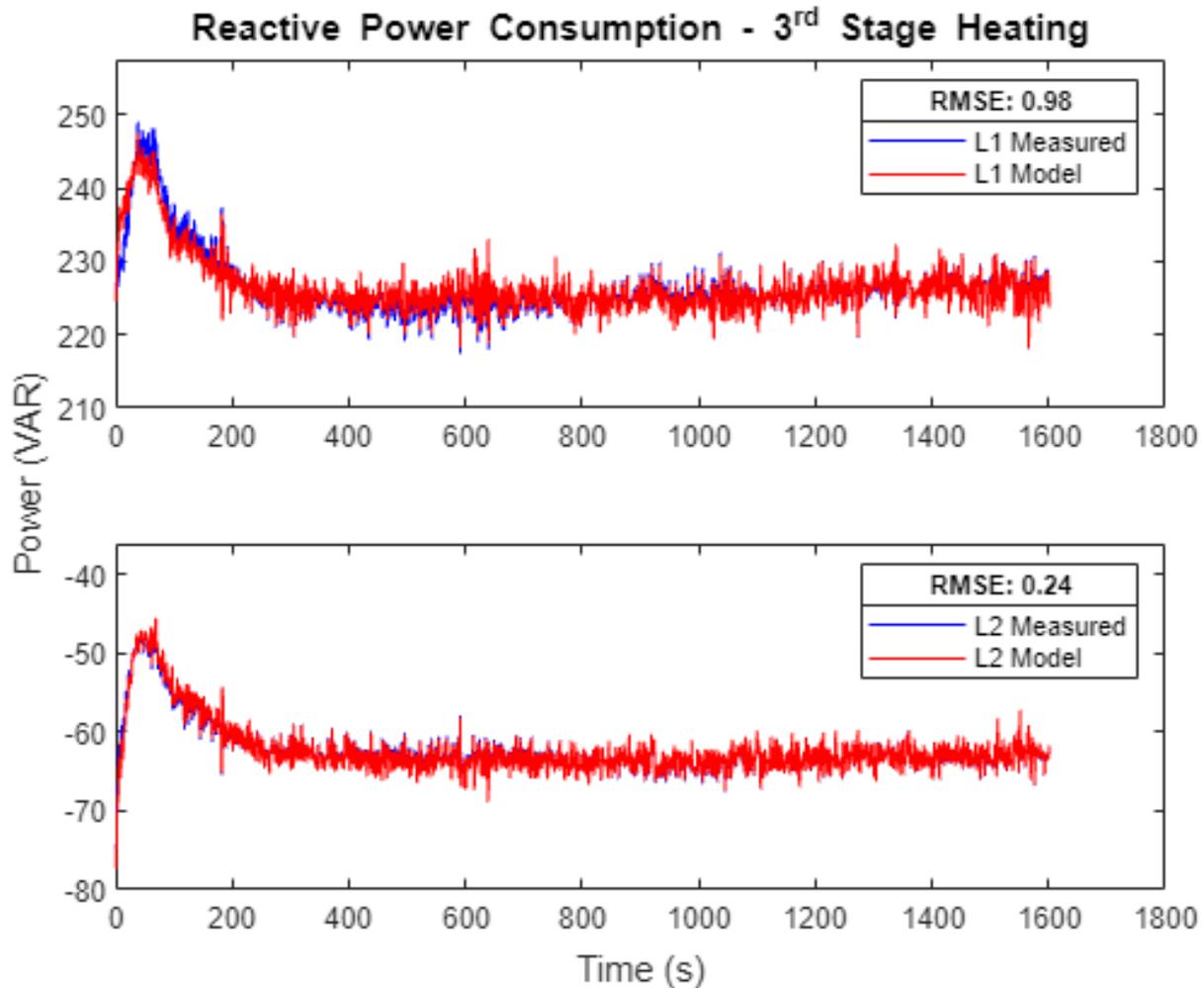
A comparison between the measured and predicted reactive power consumption of the heat pump during the 2<sup>nd</sup> Stage of heating is shown Fig. 14.



**Fig. 14.** Measured versus predicted reactive power consumption ( $Q_p$ ) of the heat pump on L1 and L2 phases during the 2<sup>nd</sup> Stage of heating

As shown in Fig. 14, the predicted reactive power consumption closely matches the measured reactive power. The reactive power on L1 and L2 phases exhibit non-linear exponential behavior during the transient part of the 2<sup>nd</sup> Stage operation. The traces in Fig. 14 show more fluctuations than those shown for the 2<sup>nd</sup> Stage's real power consumption in Sec. 4.2.1.

A comparison between the measured and predicted reactive power consumption of the heat pump during the 3<sup>rd</sup> Stage of heating is shown Fig. 15.



**Fig. 15.** Measured versus predicted reactive power consumption ( $Q_p$ ) of the heat pump on L1 and L2 phases during the 3<sup>rd</sup> Stage of heating

As shown in Fig. 15, the predicted reactive power consumption closely matches the measured reactive power. The reactive powers on L1 and L2 phases exhibit non-linear exponential behavior during the transient part of the 3<sup>rd</sup> Stage operation. The traces in Fig. 15 show more fluctuations than the traces shown for the 3<sup>rd</sup> Stage of real power consumption in Sec. 4.2.1. However, compared to the traces of the reactive power shown in Fig. 13 and Fig. 14, the fluctuations are less pronounced because a significant portion of the heat pump load during the 3<sup>rd</sup> Stage is a 5 kW resistive load. A major difference between the traces of the reactive power in the 3<sup>rd</sup> Stage and other stages of the heat pump operation is that the reactive power is negative on L2. A positive reactive power indicates the presence of an inductive load, whereas a negative reactive power is indicative of a capacitive load.

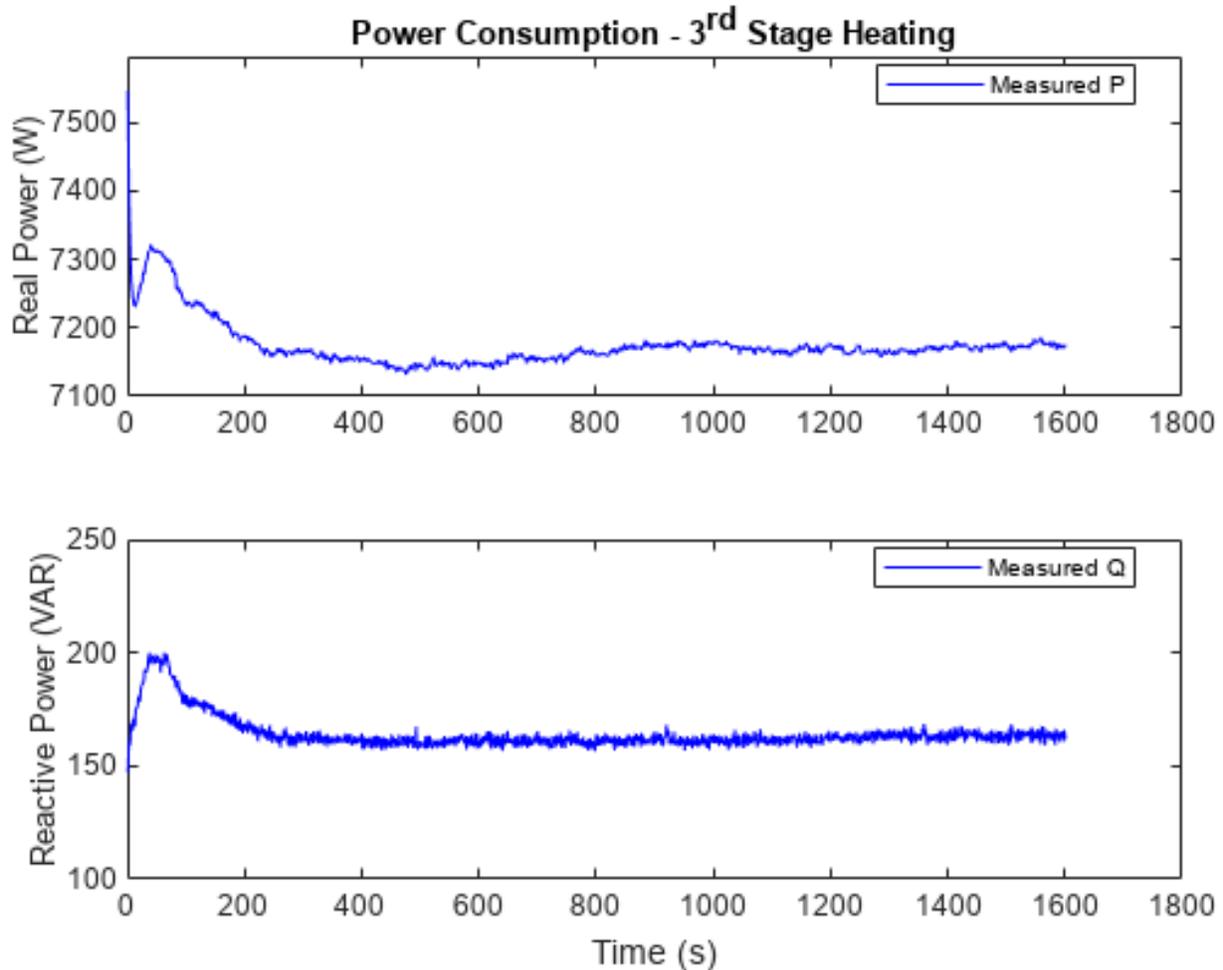
The plots representing the real power consumption of the heat pump in the cooling season are provided in Appendix B of this document.

### 4.2.3. Discussion on Power Flow

The Institute of Electrical and Electronics Engineers (IEEE) Std 1459-2010 [24] provides the four-quadrant power flow diagram for determining the type of loads and the direction of the power flow from the utilities to customer premises. In the IEEE Std 1459-2010 description of the four-quadrant power flow directions, the power flow from a customer location to the utility is termed *received*. In contrast, the power flow from a utility to the customer premise is termed *delivered*.

Using the IEEE Std 1459-2010 descriptions, the real and reactive power consumption in the 1<sup>st</sup> Stage and 2<sup>nd</sup> Stage, on L1 and L2 phases, are positive. Since the real and reactive power are positive values, the power is *delivered* from the utility to the load, and the heat pump unit acts like an inductive load. This observation is correct in both the heating and cooling seasons.

The real and reactive power consumption in the 3<sup>rd</sup> Stage of heating is positive on L1. However, on L2, the real power is positive, but the reactive power is negative, as shown in the bottom subplot of Fig. 15. When the real power is positive, and the reactive power is negative, the heat pump acts like a capacitive load. In this case, the real power is *delivered* to the customer from the utility, and the reactive power is *received* by the utility. In other words, the heat pump injected reactive power back into the utility grid on L2. However, the assertion that the heat pump acts like a capacitive load on one leg of the split phases could be misleading. The real and reactive power consumption of a split-phase system is additive, as shown in Fig. 16. Therefore, the total real and reactive power is positive, and the heat pump is an inductive load.



**Fig. 16.** Measured total real and reactive power consumption of the heat pump during the 3rd Stage of heating

As shown in Fig. 16, the total power consumption of the heat pump is the sum of real and reactive power consumptions on L1 and L2 phases.

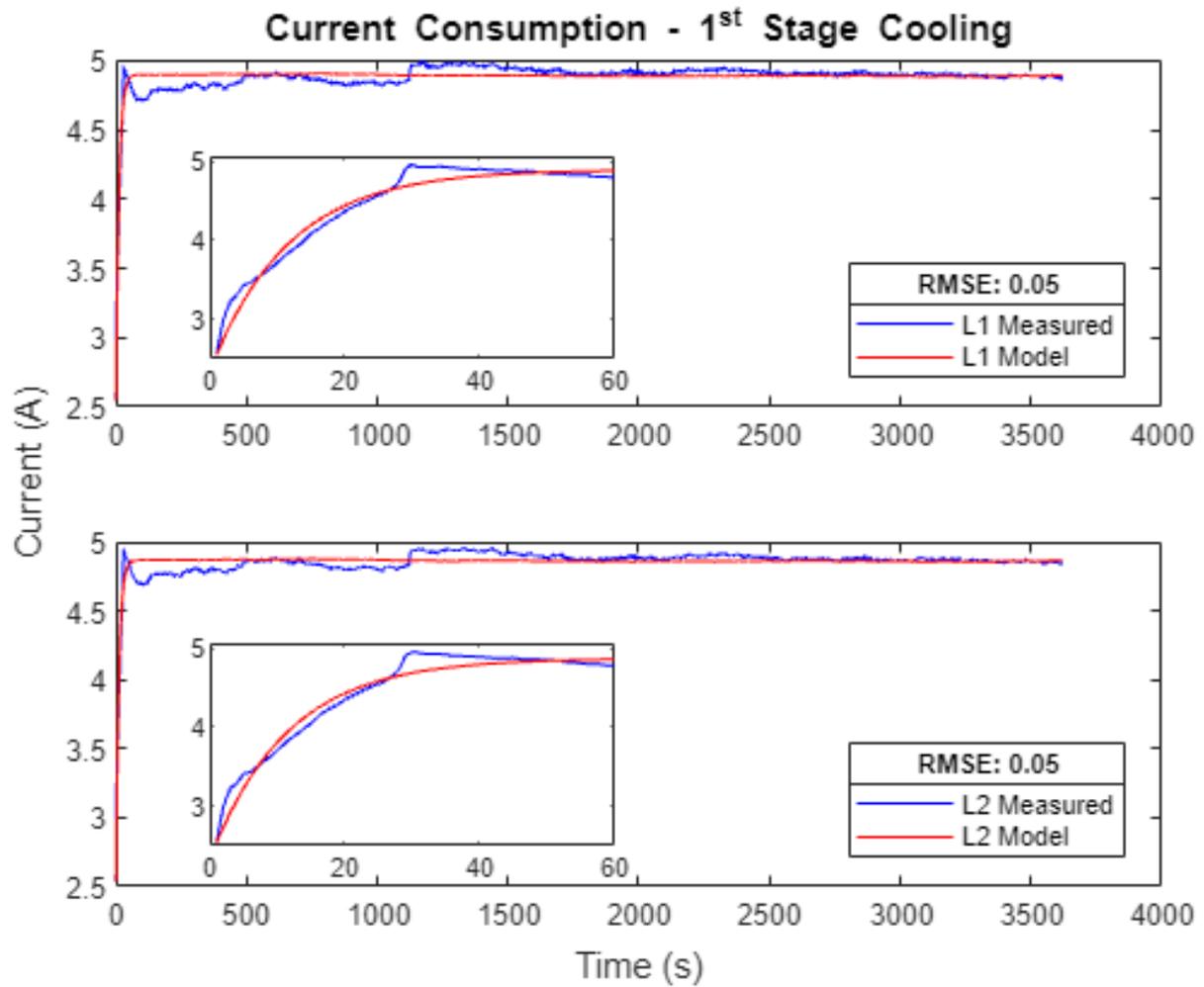
## 5. Conclusion

Recent work has documented the importance of buildings, as a set of electric loads and generation sources, in managing the stability of the power system in a smart grid. Traditionally, utilities controlled the supply of electric energy to match the demand for buildings to maintain stability. A critical tool for managing the balance between supply and demand is load forecasting. Load forecasting requires models that are accurate, adaptive, and simple to implement. Prior work has documented complicated dynamic models or simplified steady-state modeling approaches for power flow and stability analysis. However, these traditional modeling approaches cannot address the dynamic behavior of the distribution grid that is rapidly changing due to the proliferation of renewable energy sources, electrical storage, including electric vehicles, and non-linear loads such as heat pumps. Heat pumps are a significant source of electric energy consumption in residential buildings. This document demonstrated the development of the novel gray-box model (RL Model) to forecast the two-stage air-source heat pump's steady-state and transient current consumption in

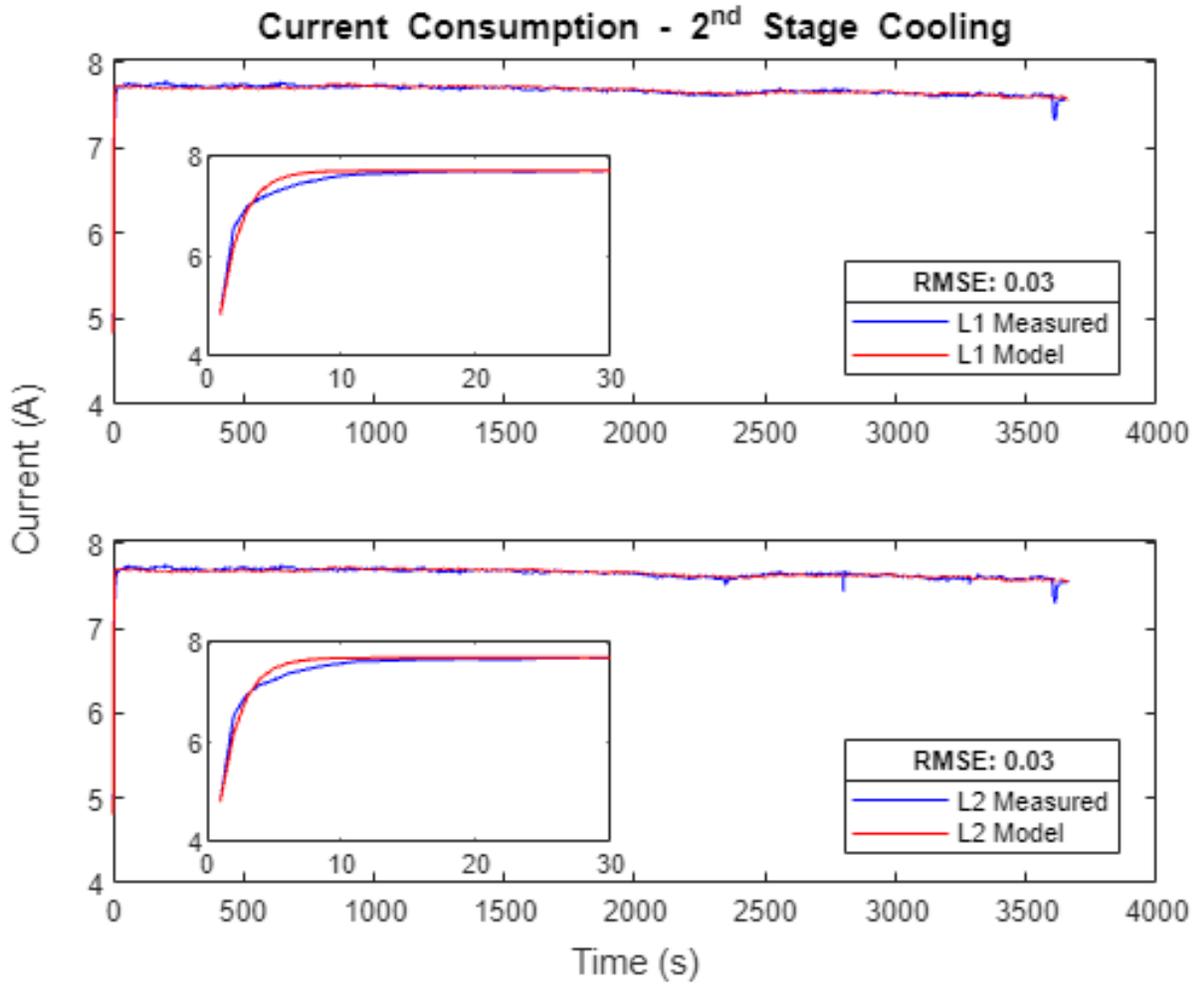
heating and cooling seasons. The RL Model was derived from a solution of a first-order differential equation describing the characteristics of a resistor-inductor equivalent circuit and the steady-state power consumption of a heat pump. Key parameters of the RL Model were estimated using the LA. The performance of the RL Model was validated using measurement data from the Net-Zero Energy Residential Test Facility located on the National Institute of Standards and Technology (NIST) campus in Gaithersburg, MD. The predicted output of the RL Model, current consumption, was used to estimate the heat pump's real and reactive power consumption for different operating stages and seasons. A list of key parameters, like the time constant and temperature-dependent coefficients of the RL Model, were described and tabulated. It was noted that the knowledge of the time constant could provide critical information for analyzing the aggregated effect of controlling many heat pumps, as flexible loads, on grid stability and the provision of ancillary services. The average root mean squared error between the RL Model's predicted current output and the measured current consumption was 0.06.

## Appendix A

The following figures depict the prediction capability of the LA for modeling the current consumption of the two-stage heat pump during the cooling season. Each figure shows the comparison between the predicted and measured values.



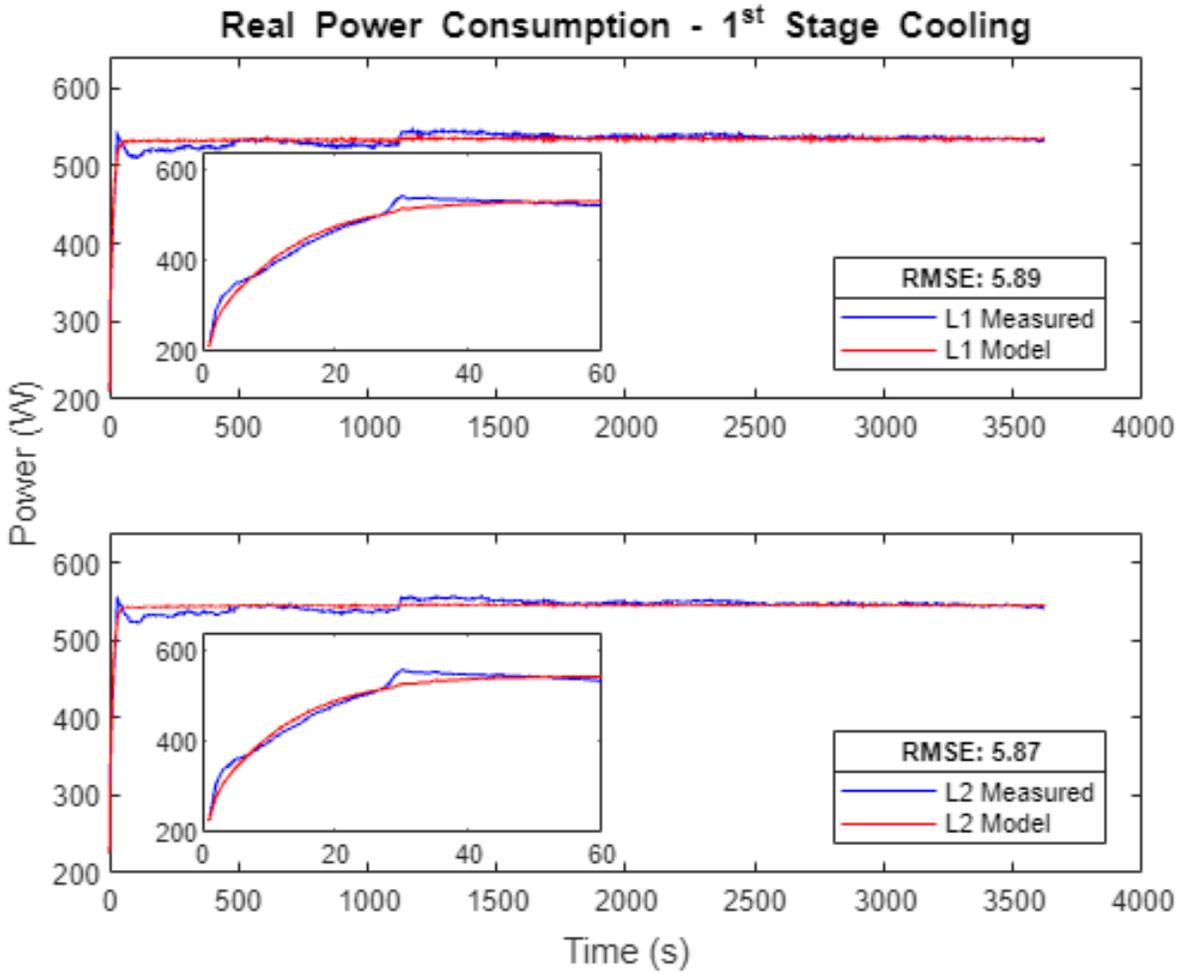
**Fig. 17.** Measured versus predicted current consumption of the heat pump on L1 and L2 phases during the 1<sup>st</sup> Stage of cooling



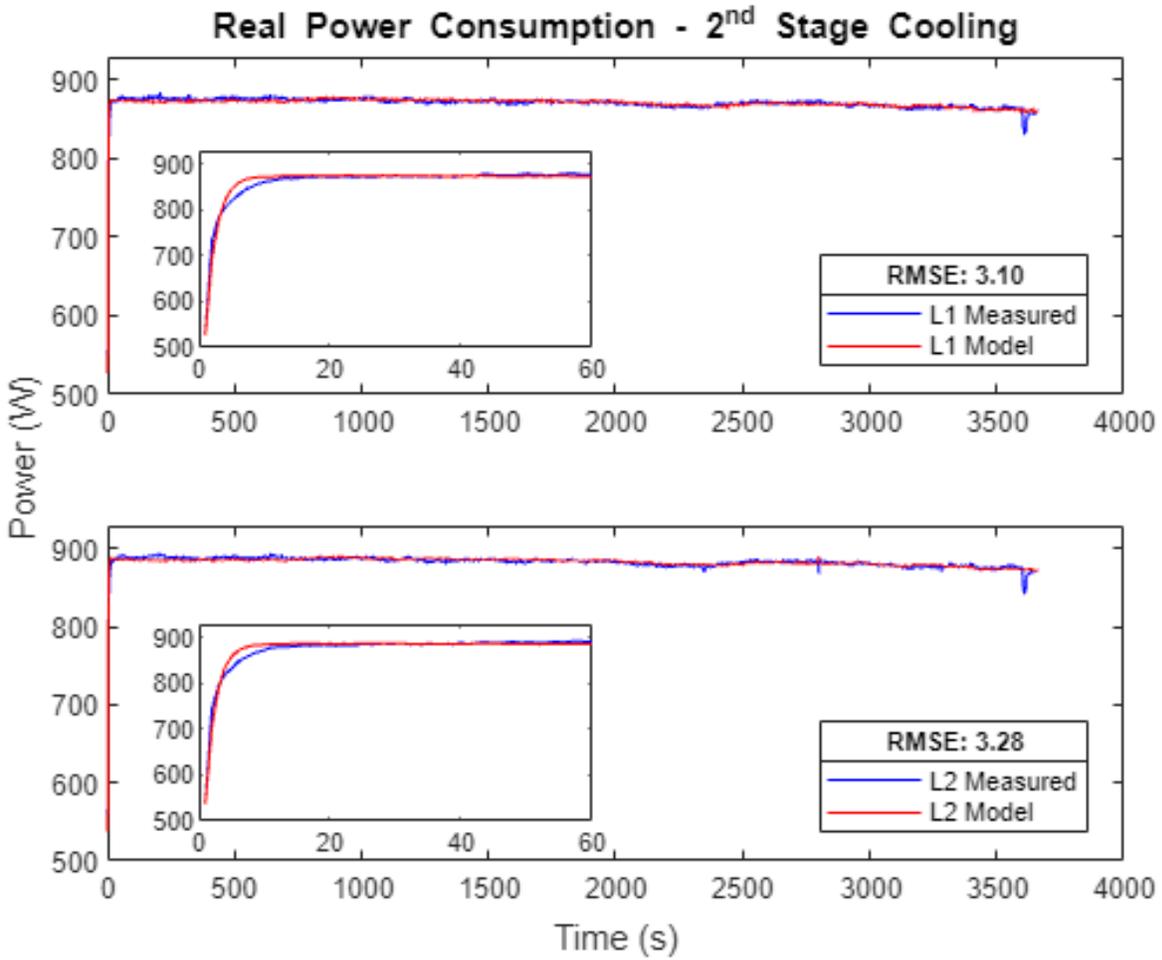
**Fig. 18.** Measured versus predicted current consumption of the heat pump on L1 and L2 phases during the 2<sup>nd</sup> Stage of cooling

## Appendix B

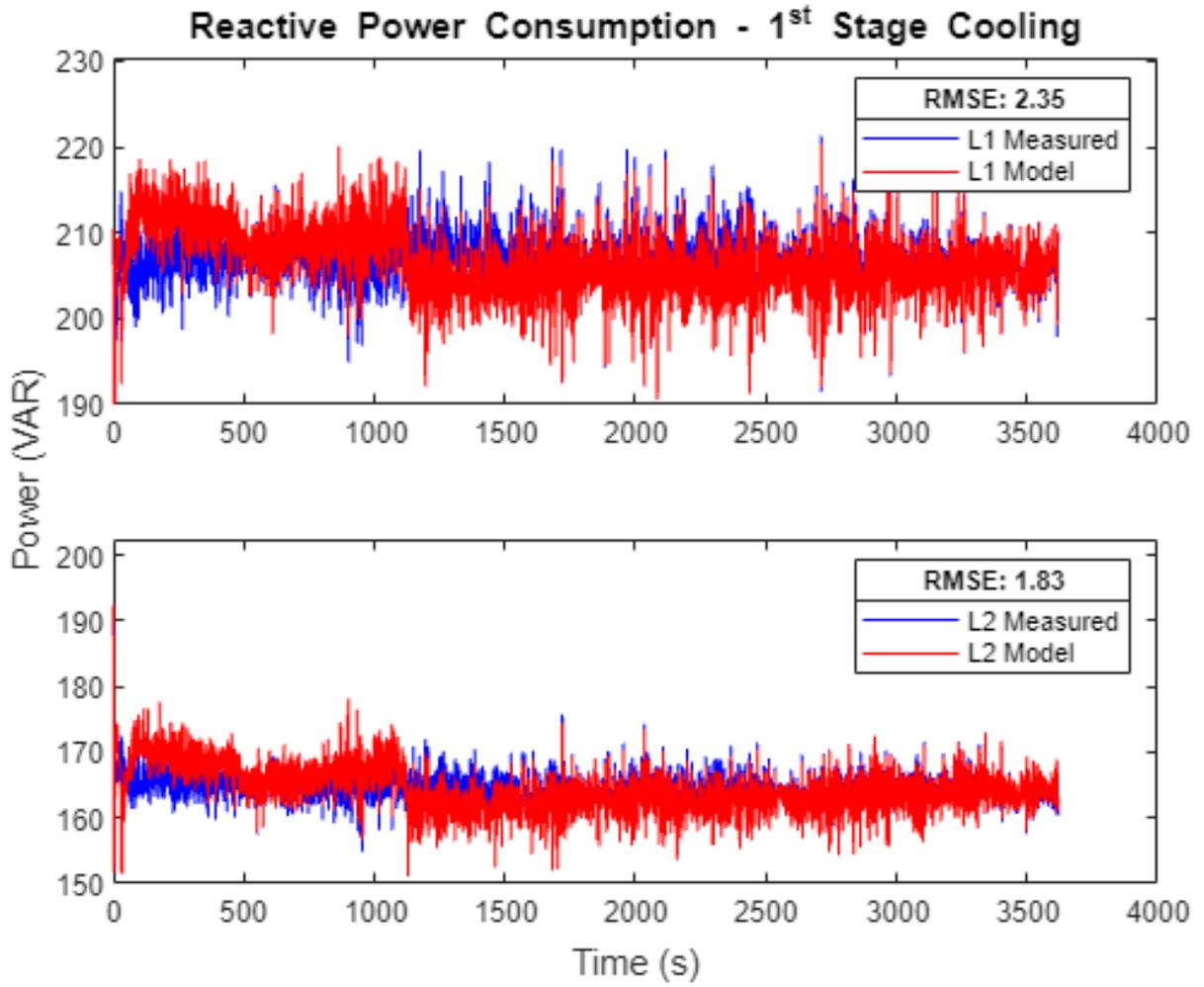
The following figures depict a comparison between measured and predicted values of the real and reactive power consumption of the two-stage heat pump during the cooling season.



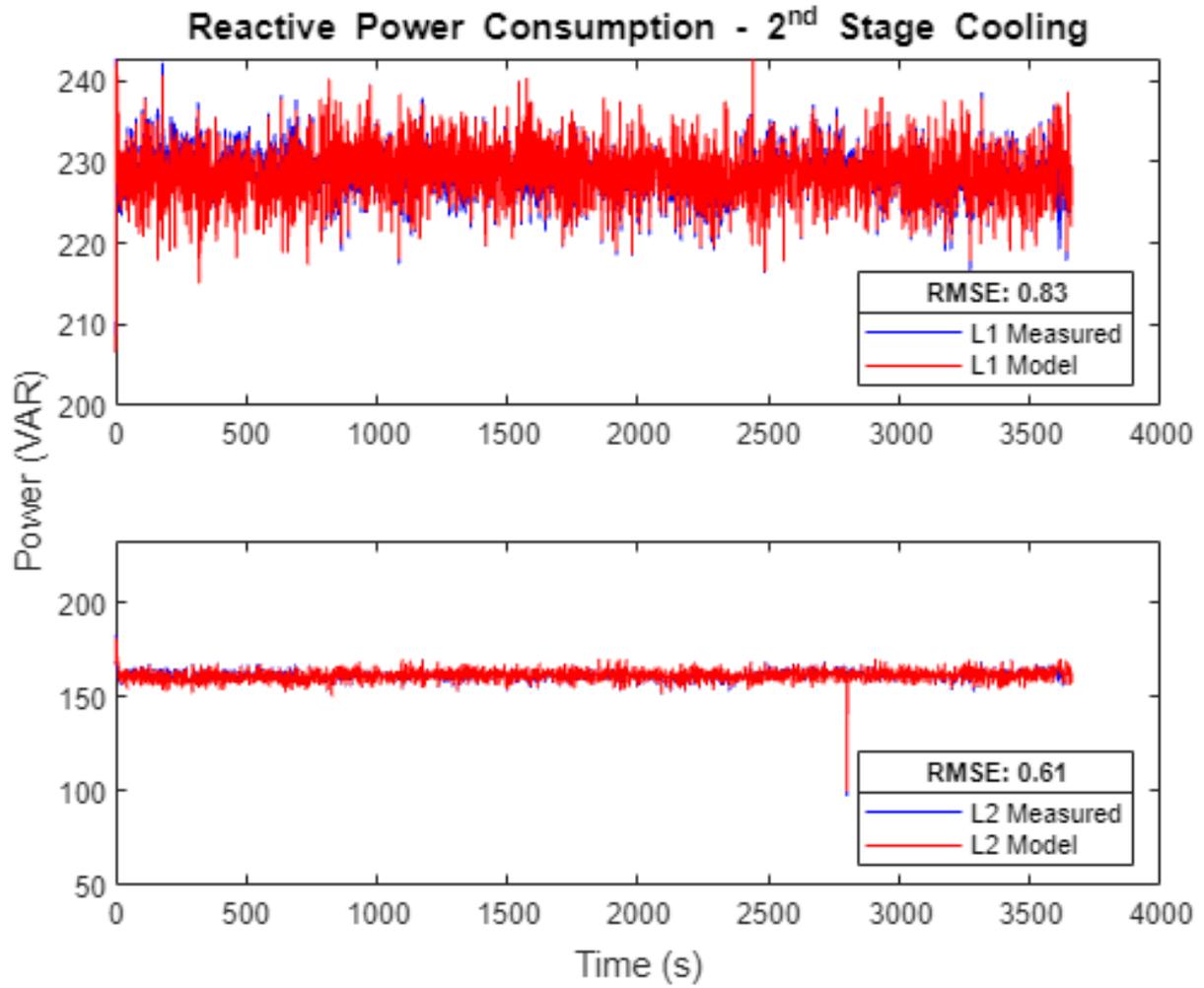
**Fig. 19.** Measured versus predicted real power consumption of the heat pump on L1 and L2 phases during the 1<sup>st</sup> Stage of cooling



**Fig. 20.** Measured versus predicted real power consumption of the heat pump on L1 and L2 phases during the 2<sup>nd</sup> Stage of cooling



**Fig. 21.** Measured versus predicted reactive power consumption of the heat pump on L1 and L2 phases during the 1<sup>st</sup> Stage of cooling



**Fig. 22.** Measured versus predicted reactive power consumption of the heat pump on L1 and L2 phases during the 2<sup>nd</sup> Stage of cooling

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