

Computational Modeling and Validation of Aerosol Deposition in Ventilation Ducts

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Abstract In fire models, the accurate prediction of aerosol & soot concentrations in the gas phase and aerosol & soot deposition thicknesses in the condensed phase is important for a wide range of applications, including human egress calculations, heat transfer in compartment fires, and forensic reconstructions of fires. During a fire, in addition to soot transport by advection and diffusion, a significant amount of soot can be deposited on surfaces due to various mechanisms. As a first approach of quantifying aerosol deposition predictions under non-reacting flow conditions, this study identifies important parameters related to aerosol deposition under various flow conditions and compares predicted aerosol deposition quantities to experimentally measured data. The computational tool used in this study was the computational fluid dynamics (CFD) fire model, Fire Dynamics Simulator (FDS). Model predictions are compared to measured aerosol deposition

velocities for various sizes of monodisperse fluorescent particles and various air velocities at the ceiling, wall, and floor of a ventilation duct.

Keywords aerosol deposition · CFD modeling · gravitational settling · computational modeling

1 Introduction

In fire models, the accurate prediction of aerosol/soot concentrations in the gas phase and aerosol/soot deposition thicknesses in the condensed phase is important for a wide range of applications [1], including human egress calculations, heat transfer in compartment fires, and forensic reconstructions of fires. During a fire, in addition to soot transport by advection and diffusion, a significant amount of soot can be deposited on surfaces due to various mechanisms, and soot can agglomerate to form large particles. These soot transport, deposition, and agglomeration mechanisms are summarized in Fig. 1.

Relatively simple parameterizations of soot chemistry and transport are incorporated into most computational fluid dynamics (CFD) fire models. Soot production is typically specified by means of a fixed soot yield for a given fuel. This assumption typically holds for well-ventilated fire scenarios in the open but has limitations in vitiated conditions and for potential advances in soot production modeling.

The underprediction of soot deposition on walls and surfaces in compartment fire scenarios can result in an overprediction of smoke concentrations in the gas phase [1,3]. Soot deposition to walls can reduce the gas-phase soot concentration, which will tend to delay smoke detector activation and improve visibility. However, gravitational settling can result in increased ob-

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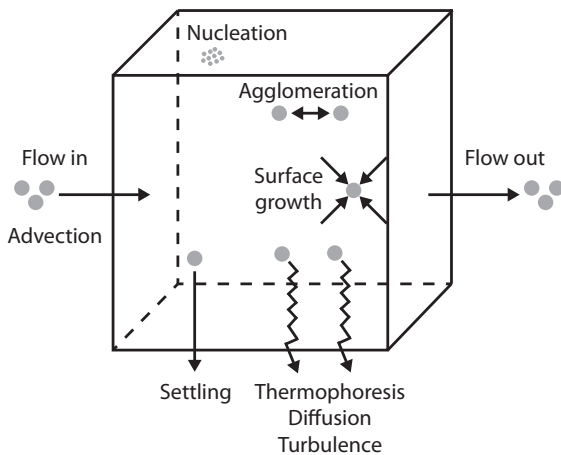


Fig. 1: Soot transport, deposition, and agglomeration processes. Adapted from Friedlander [2].

scuration in the lower layer. Gas-phase measurements of soot volume fraction used to parameterize soot generation models may be in error if significant deposition occurs. By not explicitly accounting for soot deposition on walls and surfaces, errors in soot concentration predictions in the gas phase can impact the calibration parameters for soot models. Errors in predicted quantities related to soot can affect life safety design decisions and can have adverse effects on performance goals related to both occupant safety and property protection [4, 5]. Soot deposition can also be used to reproduce fire and smoke patterns on surfaces, which can be beneficial in forensic fire reconstruction exercises.

Various aerosol/soot deposition mechanisms, including thermophoretic deposition (where temperature gradients push the aerosol towards or away from the surface), gravitational settling, molecular and turbulent diffusion (where the aerosols move along the boundary layer concentration gradient or deposit due to a turbulent boundary layer), and electrophoretic deposition can reduce the soot particle concentration in the gas phase by depositing soot on surfaces such as walls and ceilings.

Soot agglomeration occurs at different scales and can increase the size of soot particles and affect soot deposition rates in both the flaming region and the post-flame environment [1, 6]. In a fire, the largest particle number densities will be located in the flaming region and in the near-field plume, and particle number densities will decrease as air is entrained (particle mass density decreases) and as particles agglomerate. In far-field region, soot agglomeration continues to occur at any location where aerosol particles are present. Aerosol deposition mechanisms are similarly affected. Specifically, thermophoretic forces on a particle are greatest

for small particles in high temperature conditions, and gravitational settling forces are greatest as the average particle size increases due to agglomeration.

As a first approach of quantifying aerosol deposition predictions under non-reacting flow conditions, this study identifies important parameters and physics related to aerosol deposition under various flow conditions. Aerosol deposition mechanisms will be compared, and predicted aerosol deposition rates will be compared to experimentally measured data. Because the experiments considered in this study were conducted under isothermal conditions, there is not an evaluation of the role of thermophoretic deposition in the total amount of deposited aerosol mass. Rather, this validation work focuses on the role of turbulent deposition and gravitational settling rather than thermophoretic deposition or soot agglomeration.

2 Background

Several studies have been conducted that indicate soot deposition is an important factor in compartment fires for the accurate prediction of smoke concentrations, smoke detector activations, and visibility. Gottuk et al. [7] reported that smoke concentrations predicted by Fire Dynamics Simulator (FDS) near smoke alarms in a corridor were two to five times greater than measured smoke concentrations. Hamins et al. [3] conducted full-scale compartment fire experiments for use in validation studies of various fire models, including FDS. The results indicated that smoke concentrations predicted by FDS were up to five times greater than measured smoke concentrations. Floyd and McDermott [8] implemented thermophoretic and turbulent diffusion soot deposition mechanisms in FDS and compared predicted soot concentrations to measurements from small- and large-scale experiments. Riahi [5] conducted bench-scale experiments to measure soot densities and soot deposition patterns on walls for various fuels. In those studies, Riahi identified thermophoretic deposition as an important soot deposition mechanism in a hot gas layer. Cohan [4] used FDS to simulate select cases from the Gottuk [7] corridor tests, Hamins et al. [3] experiments, and Riahi [5] hood experiments with thermophoretic and turbulent diffusion soot deposition mechanisms. Newman et al. [9] conducted experiments to quantify smoke deposition velocities and smoke damage to electrical components used in industrial facilities.

Soot formation and growth processes within a flame involve a range of different particle sizes. Agglomeration can occur at any location where aerosol particles are present. If two particles collide, then there is some chance they will stick together. The probability

of collision is a function of the particle number density. Initially, soot particles that form in a flame are on the order of a few nm [10]. As soot particles interact, collide, and stick to one another, they agglomerate to form larger particle sizes over time. Within the flame region, soot particle sizes can range between $0.02 \mu\text{m}$ and $0.05 \mu\text{m}$ [1], depending on the type of fuel. As the soot particles agglomerate in the post-flame environment, the particle size distribution can range from approximately $0.1 \mu\text{m}$ to $10 \mu\text{m}$ or larger, with peaks in the distribution that depend on the fuel, temperature, flow conditions, etc. Median aerodynamic diameters of soot particles can range from $0.05 \mu\text{m}$ for wood to $10 \mu\text{m}$ for acetylene, and a majority of fuels have median aerodynamic diameters less than $1 \mu\text{m}$ [1]. High-sooting fuels, such as toluene and acetylene, can produce large soot particle sizes (or “superaggregates”) ranging from $10 \mu\text{m}$ to $100 \mu\text{m}$ or larger [11,12,13]. In a CFD fire model such as FDS, grid cell sizes are typically on the order of 10 cm . At this coarse resolution, the flame sheet and dynamics of soot formation cannot be captured. In compartment fire scenarios, we are typically more interested in the bulk transport of soot and its effect on targets some distance from the fire. Therefore, this study focuses on implementing models and empirical correlations that describe soot transport and deposition using accurate, inexpensive techniques.

In soot transport algorithms, it is important to track a meaningful representation of the soot particle size because soot deposition and agglomeration mechanisms are dependent upon the size and stopping distance of the particles. The description and evolution of a detailed soot particle size distribution can be computationally expensive. A soot particle size distribution is shown in Fig. 2. In this figure, d_p is the particle size and $n(d_p)$ is the number of particles in a given size bin. Rather than transporting detailed information about the particle size distribution (left side of Fig. 2), the mean soot diameter $d_{p,\text{mean}}$ can be used to approximately describe and evolve the full soot particle size distribution (right side of Fig. 2).

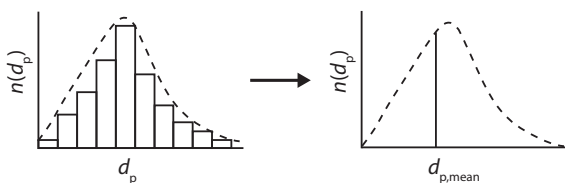


Fig. 2: Detailed particle size distribution (left) represented by mean particle size (right).

3 Governing Equations

3.1 Soot Transport in Gas-Phase Cells

If we define the mass fraction of an aerosol species α as $Y_\alpha = \rho_\alpha/\rho$, where ρ_α is the particle mass concentration and ρ is the density of the gas (kg/m^3), then the species transport equation is given by [14]

$$\frac{\partial}{\partial t}(\rho Y_\alpha) + \nabla \cdot (\rho Y_\alpha \mathbf{u}) = \nabla \cdot (\rho D_\alpha \nabla Y_\alpha) + \dot{m}_\alpha''' \quad (1)$$

where Y_α is the mass fraction of species α (kg/kg), \mathbf{u} is the gas velocity (m/s), D_α is the diffusivity of species α (m^2/s), and \dot{m}_α''' is the source term for species α ($\text{kg}/\text{m}^3\text{-s}$), which can be specified via a bulk mass production rate or a chemical mass production or consumption rate.

For each aerosol species in the gas phase, a gravitational settling velocity is calculated and imposed on the velocity in the convective term in the species transport equation, which results in the following modified form of the equation for aerosols

$$\frac{\partial}{\partial t}(\rho Y_\alpha) + \nabla \cdot (\rho Y_\alpha \mathbf{U}) = \nabla \cdot (\rho D_\alpha \nabla Y_\alpha) + \dot{m}_\alpha''' \quad (2)$$

where \mathbf{U} is the relative particle velocity (m/s), which can be decomposed into the gas velocity and particle velocity as $\mathbf{u} + \mathbf{u}_p$.

In the modified form of the species transport equation (Eq. 2), the particle velocity \mathbf{u}_p is equal to the gravitational settling velocity \mathbf{u}_g . This approach is similar to the drift flux model for smoke transport described in Hu et al. [15]. In Eq. 2, soot deposition is imposed as a boundary flux condition at surfaces, which is described in the following section.

3.2 Soot Deposition to Surfaces

Adapting Eq. 2 for one dimensional flux towards a surface, the change in the soot mass fraction (ρY_α) in a control volume results in a boundary condition for which soot is removed from the gas-phase and deposited onto the wall surface at the rate \dot{m}'' , which is given by

$$\dot{m}'' = \rho Y_\alpha u_{\text{dep}} \quad (3)$$

Using this flux condition, the amount of aerosol that deposits on the surface is removed from the adjacent gas-phase cell, and the amount of aerosol that accumulates on the surface is tracked.

The total aerosol deposition velocity to surfaces, u_{dep} , is determined by assuming the deposition phenomena are independent, computing a deposition velocity for each mechanism, and then summing them as [16]

$$u_{\text{dep}} = u_g + u_{\text{th}} + u_{\text{dt}} \quad (4)$$

where u_g is the gravitational settling velocity (for gas-phase cells adjacent to upward-facing surfaces), u_{th} is the thermophoretic velocity, and u_{dt} is the combined diffusion-turbulence velocity. These deposition mechanisms will be discussed in more detail in the following sections.

3.3 Gravitational Settling

Gravitational deposition occurs due to the force of gravity acting on particles, which results in a downward gravitational settling velocity. As the size of a particle increases, the gravitational force on the particle also increases. The gravitational settling velocity is given by [17]

$$u_g = gm_a \frac{\text{Cn}}{6\pi \chi_d \mu r_a} \quad (5)$$

where m_a is the particle mass and is constant, χ_d is the shape factor, μ is the dynamic viscosity of air, r_a is the particle radius and is constant, and Cn is the Cunningham slip correction factor given by [18]

$$\text{Cn} = 1 + 1.25 \text{Kn} + 0.41 \text{Kn} e^{-0.88/\text{Kn}} \quad (6)$$

where Kn is the particle Knudsen number given by the ratio of the mean free path of the gas to the particle radius. The mean free path of a gas is proportional to its temperature, thus Kn is computed as [19]

$$\text{Kn} = \frac{\lambda}{r_a} \frac{T_g}{T_\infty} \quad (7)$$

where λ is the mean free path of gas molecules and is equal to $0.065 \mu\text{m}$ at a temperature of 25°C and atmospheric pressure.

3.4 Thermophoretic Deposition

Thermophoretic deposition is the result of a temperature gradient near a surface, which can attract particles towards or repel particles away from a nearby surface. The thermophoretic velocity is computed as

$$u_{\text{th}} = \frac{K_{\text{th}} \nu}{T_g} \frac{dT}{dx} \quad (8)$$

where ν is the kinematic viscosity of air, and T_g is the gas temperature. This requires the calculation of the wall temperature gradient dT/dx , which is only resolved in a direct numerical simulation (DNS) with very

small grid cell sizes. For a large eddy simulation (LES) with larger grid cell sizes, the temperature gradient is computed from the wall heat transfer coefficient h as

$$\frac{dT}{dx} = \frac{h(T_g - T_w)}{k_g} \quad (9)$$

where T_w is the wall temperature, and k_g is the thermal conductivity of air. In Eq. 8, K_{th} is the thermophoretic velocity coefficient and is calculated using the following correlation [20]

$$K_{\text{th}} = \frac{2 C_s (\alpha + C_t \text{Kn}) \text{Cn}}{(1 + 3 C_m \text{Kn}) (1 + 2 \alpha + 2 C_t \text{Kn})} \quad (10)$$

where $C_s = 1.17$ is the thermal slip coefficient, α is the ratio of the gas conductivity to the particle conductivity (k_g/k_{solid}), $C_m = 1.14$ is the momentum accommodation coefficient, and $C_t = 2.18$ is the thermal accommodation coefficient.

3.5 Turbulent Deposition

In turbulent flow conditions near surfaces, turbulent deposition can occur when the inertia of particles within the flow causes them to separate from the flow, impact a surface, and deposit (or stick) to the surface.

The diffusion-turbulence deposition velocity depends upon the flow regime (diffusion, diffusion-impaction, or inertia-moderated). The deposition velocity for these regimes is given as [21]

$$u_{\text{dt}} = \begin{cases} 0.086 \text{Sc}^{-0.7} u_\tau & \tau^+ < 0.2 \\ 3.5 \times 10^{-4} \tau^{+2} u_\tau & 0.2 < \tau^+ < 22.9 \\ 0.17 u_\tau & \tau^+ > 22.9 \end{cases} \quad (11)$$

where Sc is the particle Schmidt number, or the ratio of the kinematic viscosity of air to the Brownian diffusion coefficient of the particle (ν/D_B), u_τ is the wall friction velocity computed by the wall model, or $\sqrt{\tau_w/\rho}$ (where τ_w is the wall shear stress and ρ is the density of air), and τ^+ is the dimensionless stopping distance of the particle given by [22]

$$\tau^+ = \frac{\rho_a (2r_a)^2}{18 \mu^2} u_\tau^2 \rho_g \quad (12)$$

The Brownian diffusion coefficient D_B of a particle is given by

$$D_B = \frac{kT_{\text{Cn}}}{3\pi\mu d} \quad (13)$$

where k is the Boltzmann constant ($1.38 \times 10^{-23} \text{J/K}$), T is the gas temperature, Cn is the Cunningham slip factor given in Eq. 6, μ is the dynamic viscosity of air, and d is the particle diameter and is constant.

3.6 Numerical Implementation

In summary, the velocity in the convective term in Eq. 1 for a tracked aerosol species was modified in FDS version 6.0.0 to include the gravitational settling velocity u_g . For soot deposition to surfaces, the FDS source code for gas-phase cells adjacent to walls was modified to include gravitational, thermophoretic, and turbulent deposition mechanisms as a boundary flux condition using Eqs. 3 and 4. A schematic of soot transport in FDS for various deposition mechanisms in a compartment is shown in Fig. 3. For computational cells near the ceiling or wall, the aerosol deposition velocity includes thermophoretic and turbulent mechanisms. For computational cells near the floor, the aerosol deposition velocity includes gravitational, thermophoretic, and turbulent mechanisms. For computational cells that are not near surfaces, the aerosol velocity includes the gas velocity and the gravitational settling velocity. These modifications allow for the removal of soot from the gas phase and the accumulation of soot on surfaces due to the deposition and settling of aerosols.

4 Experimental Setup

To assess the accuracy of the aerosol deposition model under simplified (non-fire) conditions, model predictions will be compared to experiments conducted in a non-reacting flow system. Experimental data measured by Sippola [23,24] were identified as a suitable candidate for model validation because the tests had well-characterized geometry and flow conditions, monodisperse aerosol sizes, and separate measurements of the ceiling, wall, and floor deposition velocities.

Sippola measured aerosol deposition velocities for various sizes of monodisperse fluorescent particles and various air velocities in a ventilation duct [23,24]. For the experiments considered here, a total of 16 aerosol deposition tests were conducted in a steel duct system. The duct had smooth walls and was square with cross-sectional dimensions of 15 cm by 15 cm. The particle diameters were 1 μm , 3 μm , 5 μm , 9 μm , and 16 μm , which is within the range of soot particle sizes that would be expected in a fire scenario (see Section 2). The air velocities in the duct were 2.2 m/s, 5.3 m/s, and 9.0 m/s with Reynolds numbers of approximately 22,000; 53,000; and 89,000, respectively. The air velocity in the duct was verified by using the average value of 16 air velocity measurements via pitot tubes arranged in a 4 by 4 grid immediately upstream of the aerosol deposition test panels.

Twelve panels measuring 20 cm by 10 cm were cut from an instrumented duct section and used to mea-

sure the amount of particles deposited on the duct surfaces: four panels each from the duct ceiling, wall, and floor surfaces. Fluorescent measurement techniques and aerosol concentration measurements were used to calculate the deposition velocities of the particles to duct surfaces (ceiling, wall, and floor) at a straight duct section where the turbulent flow profile was fully developed. Sippola reported a 10 % relative experimental uncertainty in the measured aerosol deposition velocities [23].

Figure 4a shows a schematic of the experimental setup. Figure 4b shows a detailed view of Section 2 in the duct and the panels that were used to collect the deposited aerosol and calculate the aerosol deposition velocity. A summary of the 16 aerosol deposition tests is shown in Table 1.

Table 1: Summary of Sippola aerosol deposition experiments performed in a duct.

Test Number	Air Speed (m/s)	Particle Diameter (μm)	Particle Density (kg/m^3)	Ambient Temperature ($^{\circ}\text{C}$)
1	2.2	1.0	1350	22.0
2	2.2	2.8	1170	22.0
3	2.1	5.2	1210	21.8
4	2.2	9.1	1030	22.2
5	2.2	16	950	22.4
6	5.3	1.0	1350	24.1
7	5.2	1.0	1350	23.0
8	5.2	3.1	1170	23.0
9	5.4	5.2	1210	22.9
10	5.3	9.8	1030	23.0
11	5.3	16	950	23.1
12	9.0	1.0	1350	26.9
13	9.0	3.1	1170	25.4
14	8.8	5.4	1210	25.6
15	9.2	8.7	1030	25.9
16	9.1	15	950	25.9

5 Computational Setup

Figure 4a shows a schematic of the experimental setup and the 5 m duct section that was modeled (blue shaded section). In the model, an additional 3 m of duct section (equal to 20 duct diameters) was included upstream of the instrumented portion of the duct (Section 2) to allow for the flow to become fully developed.

In the numerical simulations, the grid cell size was 1 cm on all sides. The sensitivity of the aerosol deposition rate to the grid cell resolution is discussed in Section 6.2. The air velocity in the duct was specified for each test as an upstream inlet boundary condition, and

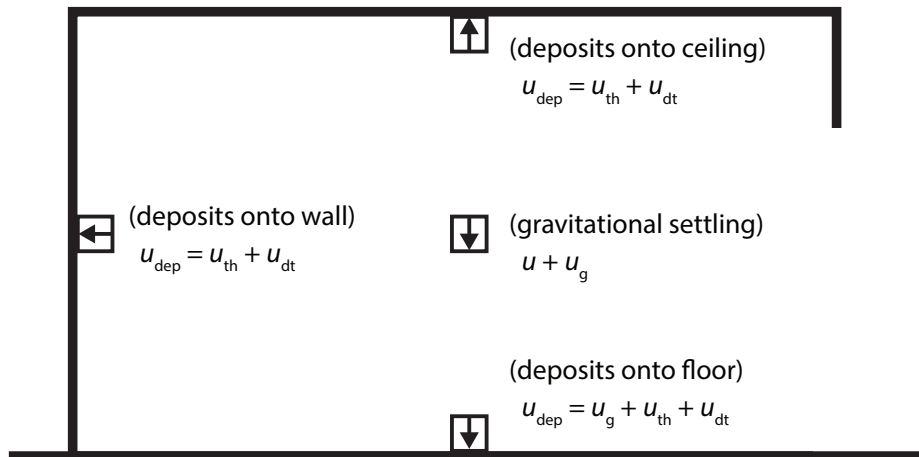
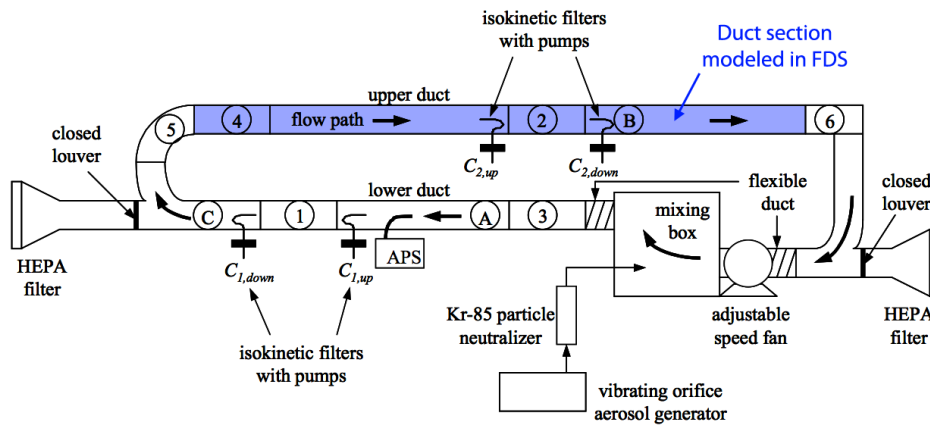
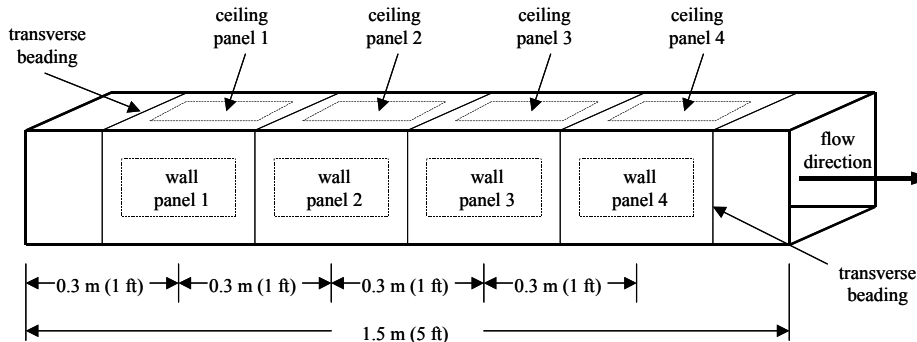


Fig. 3: Soot deposition in a simplified compartment via various mechanisms. Representative computational cells are shown in four different locations: one in the middle of the compartment, and three others near the ceiling, wall, and floor.



(a) Overview of experimental setup. The shaded region was modeled in FDS.



(b) Detailed view of instrumented duct Section 2

Fig. 4: Schematic of experimental apparatus used in Sippola aerosol deposition experiments [23].

the outlet was specified as an open boundary condition (open to ambient conditions). The duct walls were spec-

ified as a 1-mm thick, smooth material. The wall thickness did not impact the aerosol deposition rate, but is a

required input parameter and is included for completeness and reproducibility. All of the experiments were isothermal, and thus this validation will focus on the accuracy of the aerosol deposition results with respect to the gravitational settling and turbulent deposition mechanisms.

In the simulations, the aerosol species was tracked explicitly, and all of the aerosol deposition mechanisms were enabled. The measured aerosol concentrations were not given in the test reports; therefore, an aerosol concentration of 100 mg/m^3 was specified at the inlet (upstream) duct boundary. The sensitivity of the aerosol deposition rate to the inlet aerosol concentration is discussed in Section 6.3.

6 Results

6.1 Calculation of Aerosol Deposition Velocity

In the experimental setup, there was only one upstream and one downstream concentration measurement available. Therefore, to make an accurate comparison between the measured and predicted deposition velocities, we will use the same data reduction technique as Sippola [23,24].

Following the procedure by Sippola, the particle deposition velocity V_d was calculated as

$$V_d = \frac{J_1 + J_2 + J_3 + J_4}{4 C_{\text{avg}}} \quad (14)$$

where J_1 through J_4 are the deposition fluxes ($\text{kg/m}^2\text{-s}$) for duct panels 1 through 4, respectively, which is given by

$$J = \frac{\Delta m_d}{A_d \Delta t} \quad (15)$$

where Δm_d is the change in mass on the duct panel (kg), A_d is the area of the duct panel (m^2), Δt is the duration over which the aerosol deposits on the panel (s), and C_{avg} is the average of the upstream and downstream aerosol concentration in the duct test section (kg/m^3).

6.2 Grid Sensitivity

In these simulations, the critical length scale is the diameter of the duct. The velocity profile (and thus the aerosol deposition rates) in the duct will be affected by the grid resolution. To assess the impact of different grid resolutions, multiple simulations were run with grid cell sizes of 0.5 cm, 0.75 cm, 1 cm, and 2 cm at the smallest and largest air speeds and particle diameters.

The aerosol deposition rates for each case are shown in Table 2.

The baseline case had a grid cell resolution of 1 cm, and the following results compare the predicted aerosol deposition velocity for cases with different grid cell sizes compared to the baseline case. For the fine grid resolution, a grid cell size of 0.50 cm (50 % decrease) resulted in a maximum change in the aerosol deposition velocity of about 4 % for the 2.2 m/s air speed and 26 % for the 9.1 m/s air speed. A grid cell size of 0.75 cm (25 % decrease) resulted in a maximum change in the aerosol deposition velocity of about 1.5 % for the 2.2 m/s air speed and 23 % for the 9.1 m/s air speed. For the coarse grid resolution, a grid cell size of 2 cm (100 % increase) resulted in a maximum change in the aerosol deposition velocity of about 0.6 % for the 2.2 m/s air speed and 30 % for the 9.1 m/s air speed. As expected, the predicted aerosol deposition velocity changes along with the grid resolution because the velocity profile is resolved differently at different grid cell sizes.

6.3 Inlet Concentration Sensitivity

To assess the impact of different inlet aerosol concentrations, multiple simulations were run with inlet aerosol concentrations of 50 mg/m^3 , 100 mg/m^3 , 200 mg/m^3 , and 1000 mg/m^3 at the smallest and largest air speeds and particle diameters. The aerosol deposition rates at the ceiling, wall, and floor for each case are shown in Table 3.

The baseline case had a specified inlet aerosol concentration of 100 mg/m^3 , and the following results compare the predicted aerosol deposition velocity for cases with different inlet aerosol concentrations compared to the baseline case. An inlet aerosol concentration of 50 mg/m^3 (50 % decrease) resulted in a 0 % to 0.8 % change in the aerosol deposition velocity. An inlet aerosol concentration of 200 mg/m^3 (100 % increase) resulted in a 0 % to 0.3 % change in the aerosol deposition velocity. An inlet aerosol concentration of 1000 mg/m^3 (1000 % increase) resulted in a 0 % to 0.5 % change in the aerosol deposition velocity. In summary, the effect of different inlet aerosol concentrations on the predicted aerosol deposition velocity is relatively small because the aerosol deposition rates are a linear function of the aerosol concentration.

6.4 Model Predictions of Aerosol Deposition Velocity

Figures 5 through 7 show comparisons of the measured and predicted aerosol deposition velocities for the ceiling, wall, and floor surfaces. Figure 8 shows a summary

Table 2: Sensitivity of aerosol deposition rate to grid cell resolution.

Test Number	Air Speed (m/s)	Particle Diameter (μm)	Grid Cell Size (cm)	Aerosol Deposition Velocity (Ceiling) (m/s)	Aerosol Deposition Velocity (Wall) (m/s)	Aerosol Deposition Velocity (Floor) (m/s)
1	2.2	1.0	0.50	8.96×10^{-7}	9.00×10^{-7}	4.64×10^{-5}
1	2.2	1.0	0.75	8.54×10^{-7}	8.57×10^{-7}	4.60×10^{-5}
1	2.2	1.0	1.0	8.64×10^{-7}	8.68×10^{-7}	4.67×10^{-5}
1	2.2	1.0	2.0	9.16×10^{-7}	9.17×10^{-7}	4.39×10^{-5}
16	9.1	15	0.50	4.41×10^{-3}	5.08×10^{-3}	1.01×10^{-2}
16	9.1	15	0.75	4.62×10^{-3}	5.24×10^{-3}	1.05×10^{-2}
16	9.1	15	1.0	5.97×10^{-3}	6.53×10^{-3}	1.20×10^{-2}
16	9.1	15	2.0	7.74×10^{-3}	8.11×10^{-3}	1.36×10^{-2}

Table 3: Sensitivity of aerosol deposition rate to inlet aerosol concentration.

Test Number	Air Speed (m/s)	Particle Diameter (μm)	Inlet Aerosol Concentration (mg/m^3)	Aerosol Deposition Velocity (Ceiling) (m/s)	Aerosol Deposition Velocity (Wall) (m/s)	Aerosol Deposition Velocity (Floor) (m/s)
1	2.2	1.0	50	8.66×10^{-7}	8.68×10^{-7}	4.67×10^{-5}
1	2.2	1.0	100	8.64×10^{-7}	8.68×10^{-7}	4.67×10^{-5}
1	2.2	1.0	200	8.65×10^{-7}	8.68×10^{-7}	4.67×10^{-5}
1	2.2	1.0	1000	8.68×10^{-7}	8.68×10^{-7}	4.67×10^{-5}
16	9.1	15	50	5.93×10^{-3}	6.58×10^{-3}	1.20×10^{-2}
16	9.1	15	100	5.97×10^{-3}	6.53×10^{-3}	1.20×10^{-2}
16	9.1	15	200	5.97×10^{-3}	6.51×10^{-3}	1.20×10^{-2}
16	9.1	15	1000	5.95×10^{-3}	6.56×10^{-3}	1.20×10^{-2}

of the measured and predicted results for all of the particle sizes, duct velocities, and surfaces that were considered. The model generally underpredicts the measured aerosol deposition velocity by 45 %, on average.

For all of the cases that were considered, the results indicate that, as the duct velocity and particle size increase, the deposition velocity also increases. The ceiling and wall aerosol deposition velocities ranged from approximately 1×10^{-7} m/s to 1×10^{-2} m/s. The floor aerosol deposition velocities ranged from approximately 1×10^{-5} m/s to 1×10^{-1} m/s, which is most likely due to the effect of gravitational settling.

For the model predictions, the effect of increased particle sizes was the greatest for the floor aerosol deposition velocities, and the effect of increased duct velocity was the greatest for the wall and ceiling. The floor aerosol deposition velocities were the least sensitive to an increase in the duct velocity because gravitational settling was more dominant than turbulent deposition at the floor.

In the simulation of Test 1, gravitational settling and turbulent deposition accounted for about 98 % and 2 % of the total aerosol deposition velocity to the floor, respectively. In the simulation of Test 5, gravita-

tional settling and turbulent deposition accounted for about 100 % and 0 % of the total aerosol deposition velocity to the floor, respectively. In the simulation of Test 12, gravitational settling and turbulent deposition accounted for about 93 % and 7 % of the total aerosol deposition velocity to the floor, respectively. In the simulation of Test 16, gravitational settling and turbulent deposition accounted for about 49 % and 51 % of the total aerosol deposition velocity to the floor, respectively.

The cases that exhibited the largest error in the predicted aerosol deposition velocities were the 3 μm and 5 μm particle sizes at the lowest air velocities, especially at the wall. The measured deposition velocity for these cases is on the order of 1×10^{-5} m/s, which is a relatively small deposition velocity and would result in only a small amount of error in the deposited aerosol mass, as demonstrated in the following example.

Consider the case with the lowest air velocity (2.2 m/s). For an aerosol concentration of 100 mg/m^3 and a time period of 100 s, a total of 495 mg of aerosol mass flows through the duct. Using Eqs. 14 and 15 with an aerosol deposition velocity of 1×10^{-5} m/s and representative values from the Sippola experiments (aerosol concentration of 100 mg/m^3 , deposition area of 0.02 m^2 , and

time of 100 s) results in only 0.002 mg of deposited aerosol mass out of the 495 mg of total aerosol mass that flowed through the duct. Therefore, errors in the relatively small deposition velocities (on the order of 1×10^{-5} m/s) would not result in a significant amount of error in the deposited aerosol mass.

The remaining cases that used other particle sizes and duct velocities represent our best current ability to predict aerosol deposition velocities for the Sippola aerosol deposition experiments using FDS.

7 Conclusions

As a first approach of quantifying aerosol deposition predictions under non-reacting flow conditions, this study identified important parameters related to aerosol deposition under various flow conditions and compared predicted aerosol deposition quantities to experimentally measured data. To compare the predicted results to experimentally measured data, 16 tests conducted by Sippola [19] were used in which ceiling, wall, and floor aerosol deposition velocities for various sizes of monodisperse fluorescent particles and various air velocities in a ventilation duct were measured. The experiments were numerically simulated, and the measured and predicted aerosol deposition velocities were compared and found to be underpredicted by 45 %.

In general, for all of the cases that were considered, the measurements and predictions indicate that, as the duct velocity and particle size increase, the deposition velocity also increases. The effect of increased particle sizes on the aerosol deposition velocities was the greatest at the floor, and the effect of increased duct velocity on the aerosol deposition velocities was the greatest at the wall and ceiling. The floor aerosol deposition velocities were the least sensitive to an increase in the duct velocity because gravitational settling was more dominant than turbulent deposition at the floor.

Sippola reported a 10 % relative experimental uncertainty in the measured aerosol deposition velocities [23]. The model generally underpredicted the measured aerosol deposition velocity by 45 %, on average. The results were mostly biased by errors in relatively small aerosol deposition velocities (on the order of 1×10^{-5} m/s) for small particle sizes ($3 \mu\text{m}$ and $5 \mu\text{m}$) at the lowest air velocities. However, it was demonstrated that the effect of this error in the overall deposited soot mass would not be significant.

For all of the particle sizes and duct velocities that were considered, the results represent our best current ability to predict aerosol deposition velocities for the Sippola aerosol deposition experiments using FDS. Some

limitations of this work are that the soot is currently represented in the model with a single (mean) particle size, and soot agglomeration is not currently considered. Additionally, because the experiments considered in this study were conducted under isothermal conditions, there was not an evaluation of the role of thermophoretic deposition in the total amount of deposited aerosol mass. Future work includes conducting validation studies for aerosol deposition in cases with soot generated from well-characterized fires and implementing a simple soot agglomeration mechanism to account for soot particles that can grow over time, which can also impact the rate of soot deposition to surfaces. Additionally, the ability to define, track, and evolve aerosol size distributions will likely improve the accuracy of the aerosol transport predictions. As these improved soot transport, deposition, and agglomeration mechanisms are implemented in the model, we can expect that the total amount of aerosol and soot deposition as well as the gas phase aerosol and soot concentrations will be more accurate. Additional verification and validation work on the soot and aerosol deposition routines in FDS can be found in the FDS Verification Guide [25], FDS Validation Guide [26], and Overholt [27].

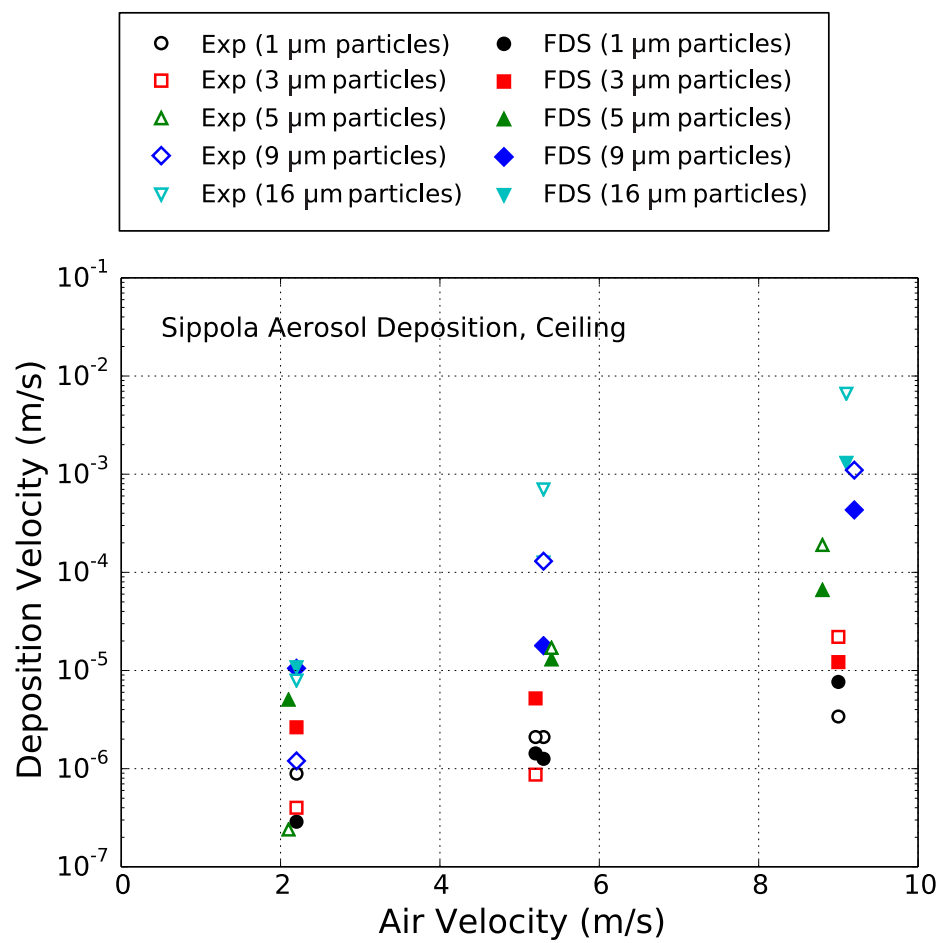


Fig. 5: Measured and predicted ceiling aerosol deposition velocities.

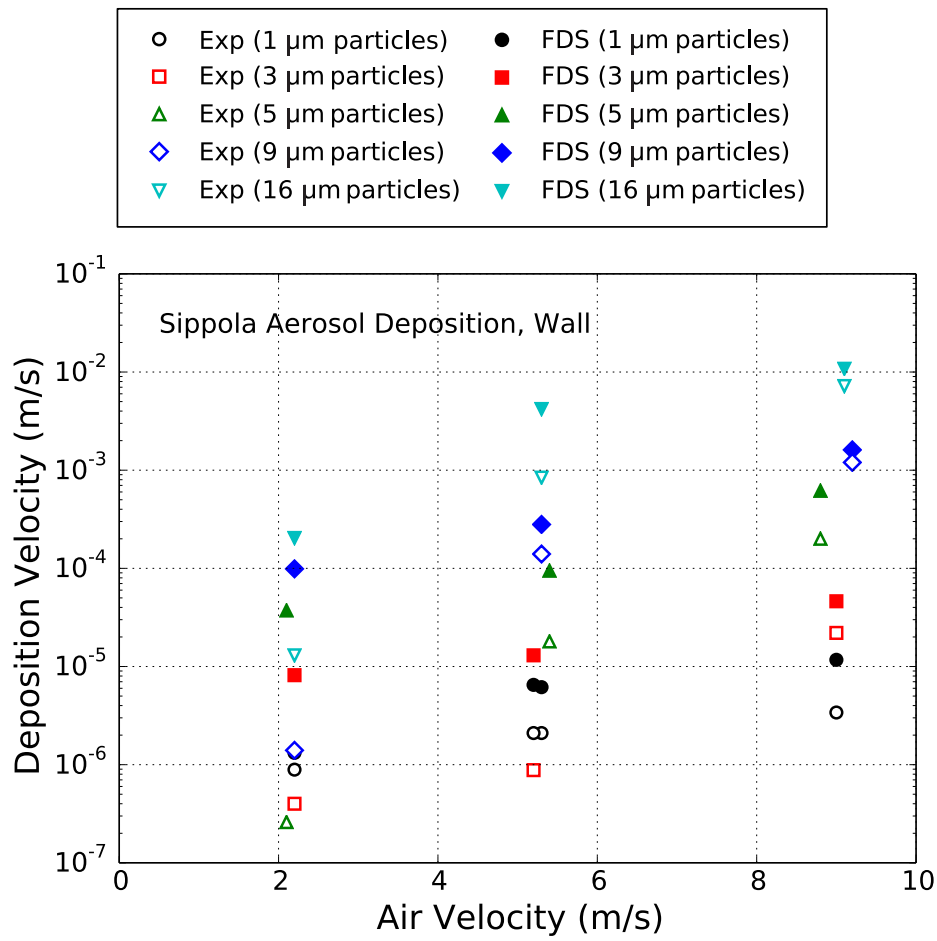


Fig. 6: Measured and predicted wall aerosol deposition velocities.

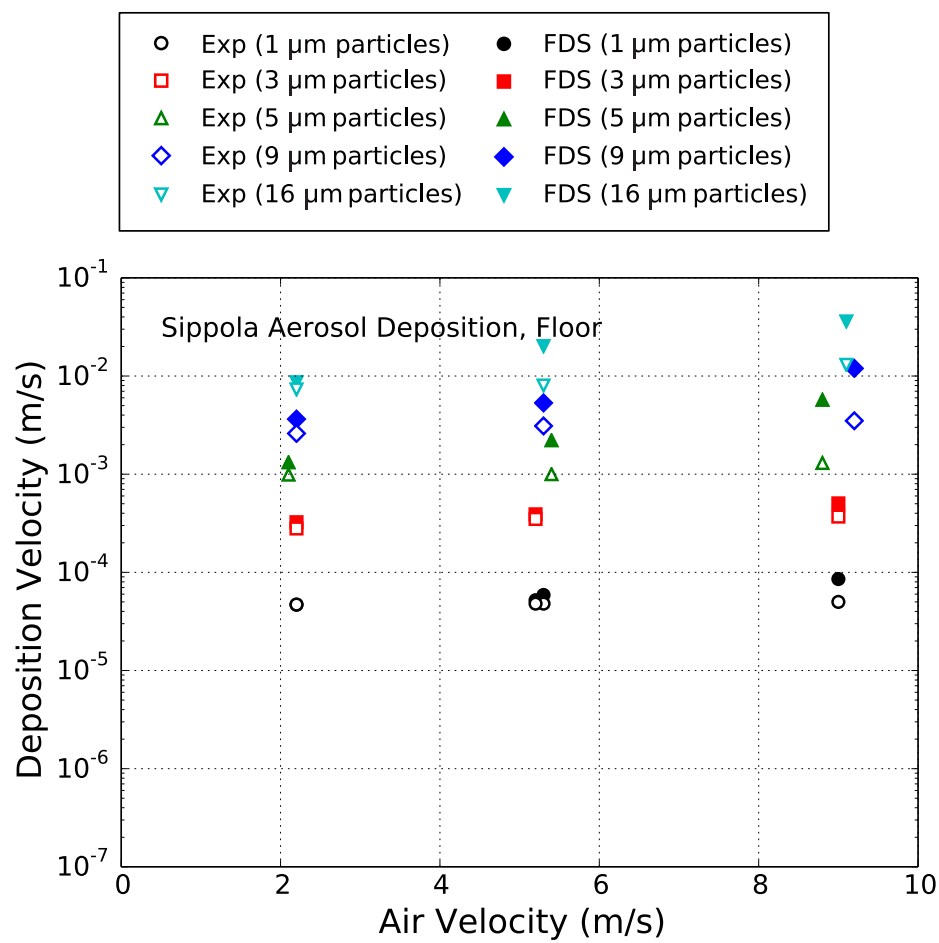


Fig. 7: Measured and predicted floor aerosol deposition velocities.

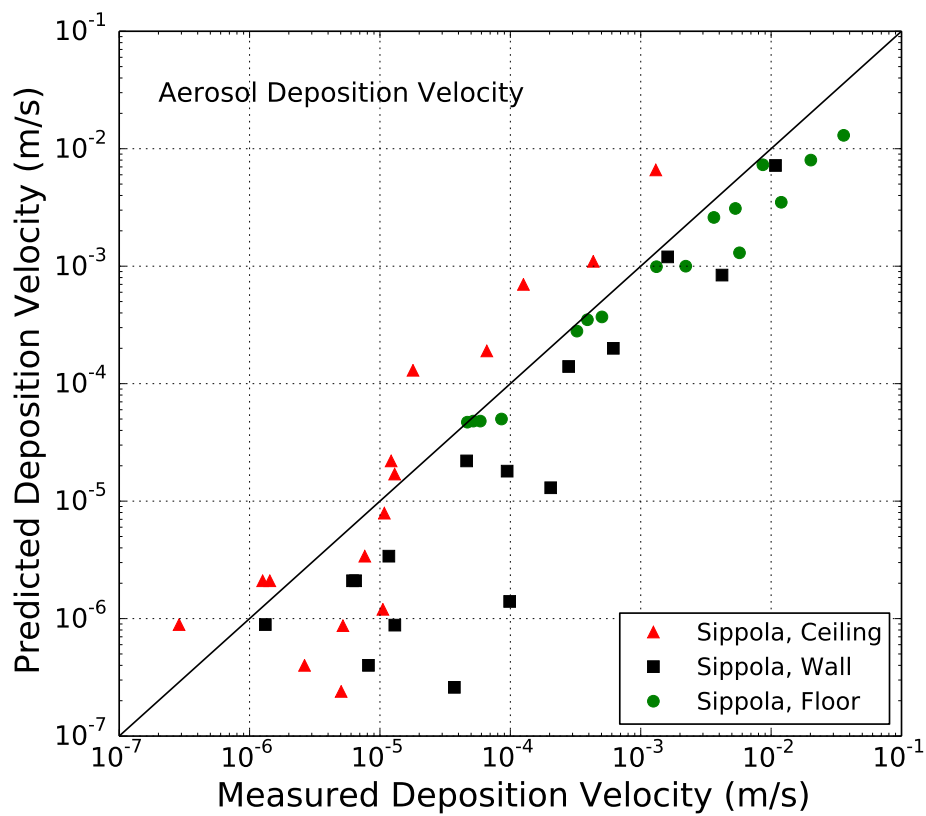


Fig. 8: Summary of aerosol deposition velocities.

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