

# **Nanolubricants for Improving Chiller Performance**

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# Nanolubricants for Improving Chiller Performance

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# Presentation Outline

**N**anoparticle Basics

**R**eview Boiling Measurement Results

....with Mechanistic Speculation

**B**oiling Model

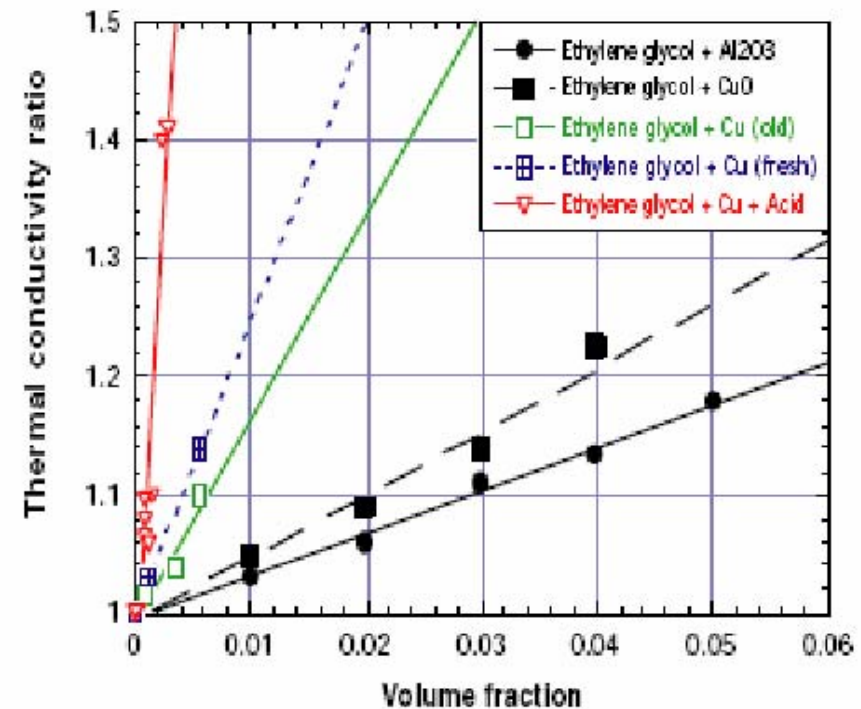
**V**iscosity Measurements and Model

**C**onclusions

# Can Nanofluids Improve Refrigerant Boiling?

*40 % improvement in  
thermal conductivity for  
a 0.4 % volume fraction*

*metal & metal-oxide dispersions*



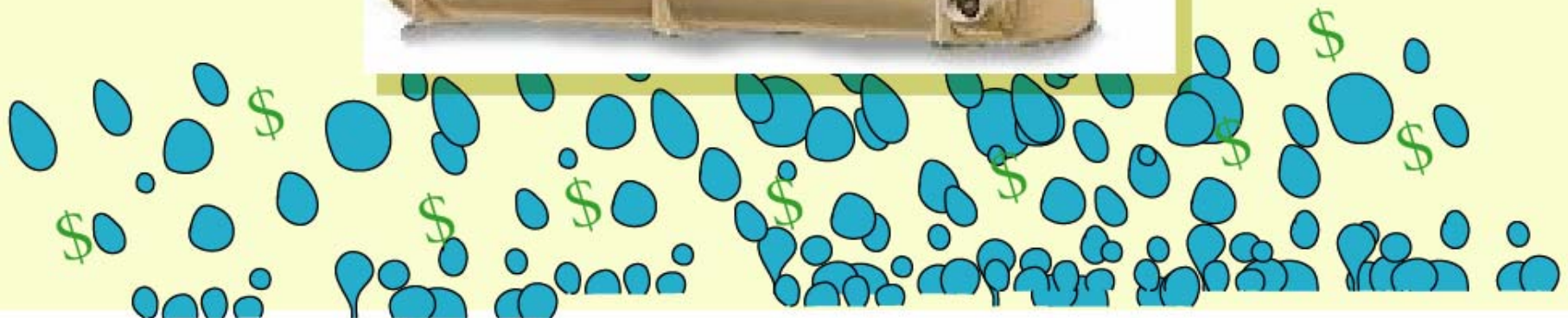
*Appl. Phys. Lett. 78, 718-720, Eastman et al. (2001)*

*gives opportunity for improving air-conditioning chiller performance*



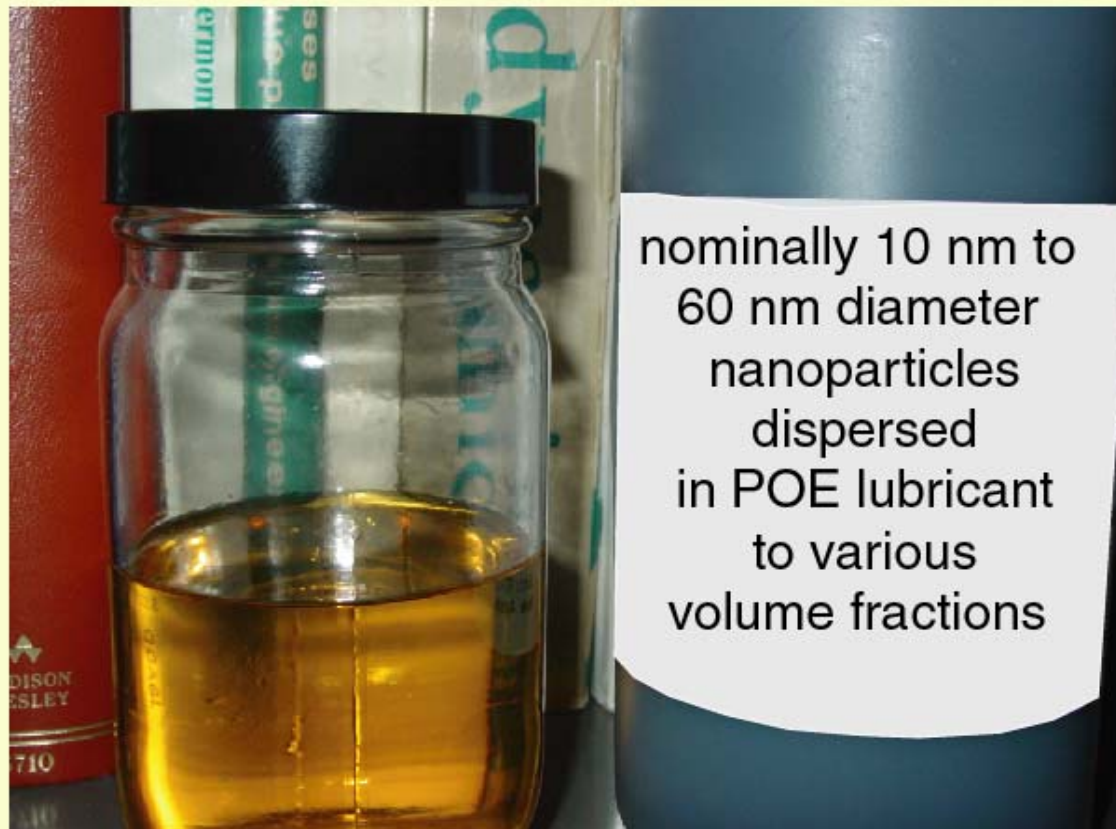
# Potential Impact of Nanofluids for Chillers

If nanofluids improve chiller efficiency by 1 %,  
a savings of 320 billion kWh of electricity or  
an equivalent 5.5 million barrels of oil per year  
would be realized in the US alone

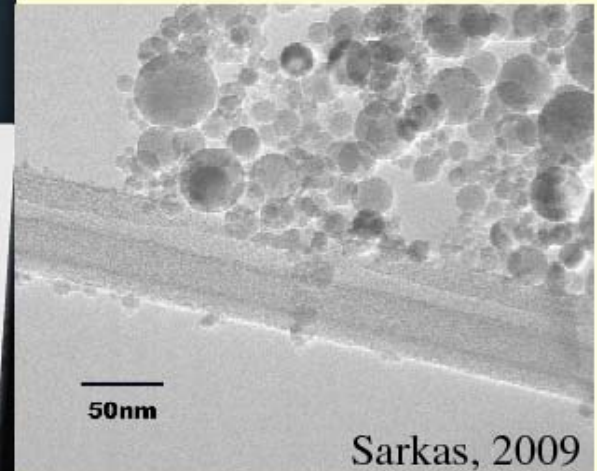


# Lubricant Based, CuO, Al<sub>2</sub>O<sub>3</sub>, and Diamond Nanofluids

Base lubricant was a POE with a nominal kinematic viscosity of  $72.3 \mu\text{m}^2/\text{s}$  at 313.15 K



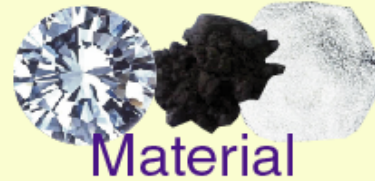
nominally 10 nm to  
60 nm diameter  
nanoparticles  
dispersed  
in POE lubricant  
to various  
volume fractions



# Nanoparticles Can Improve Refrigerant/Lubricant Boiling

*But this depends on the:*

*Properties of the nanoparticles:*



*Concentration of nanoparticles in nanolubricant:*



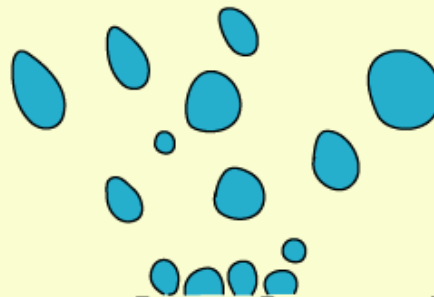
*Concentration of nanolubricant in refrigerant:*



*Boiling heat flux:*

*other factors*

*(surface geometry, etc.)*



**gives opportunity for improving air-conditioning chiller performance**



# Semi-Empirical Model for Refrigerant/Nanolubricant Boiling Fitted to Single Constant

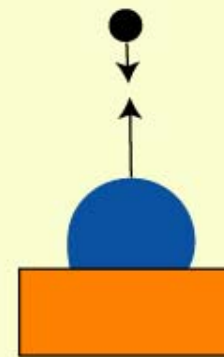
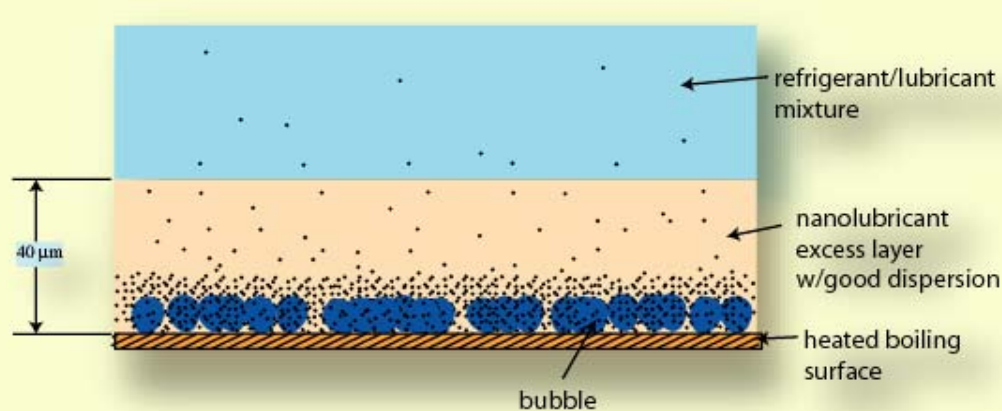
$$\frac{q''_{np}}{q''_{PL}} = 1 + \frac{3.45 \times 10^{-9} [\text{s}] \phi \sigma v_L \rho_v x_b^2}{D_{np}^4 (q''_n)^{3/2} \rho_L (\rho_{np} - \rho_L) g (1 - x_b)^2}$$

*nanoparticle volume fraction*  
*nanolubricant mass fraction*  
*pure refrigerant properties*  
*nanoparticle properties*  
*pure lubricant properties*  
*refrigerant/nanolubricant heat flux*  
*refrigerant/pure lubricant heat flux*

**does not include the boiling enhancement due to the enhancement of the lubricant properties as contributed by the properties of the nanoparticles, but this could be easily included**



# Semi-Empirical Model for Refrigerant/Nanolubricant Boiling



$M$  = mass  
 $u$  = velocity  
 $r$  = radius  
 $D$  = diameter  
 $N$  = number

*Conservation of momentum for nanoparticles impacting a single bubble*

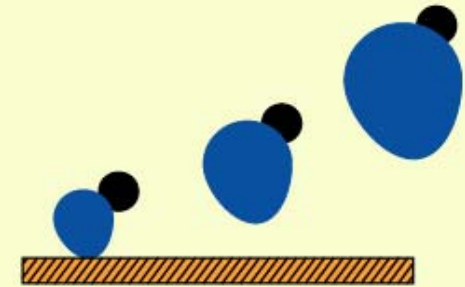
$$\frac{N_{np}}{N_b} M_{np} u_{np_i} + M_{b_i} u_{b_i} = \frac{N_{np}}{N_b} M_{np} u_{np} + M_{b_f} u_{b_f}$$

*Change in kinetic energy of the nanoparticle is equal bubble surface work*

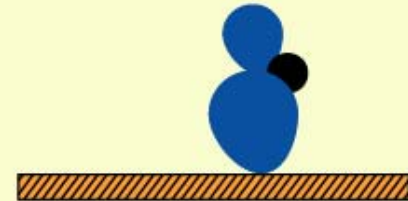
$$\frac{1}{2} M_{np} u_{np_f}^2 - \frac{1}{2} M_{np} u_{np}^2 = 4\pi\sigma(r_{b_f}^2 - r_{b_i}^2)$$

# Proposed Enhancement Mechanism

enhanced bubble growth caused by bubble/  
"hot" particle interaction

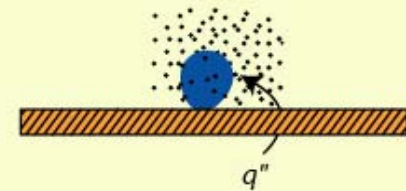


secondary nucleation on "hot" particle in fluid



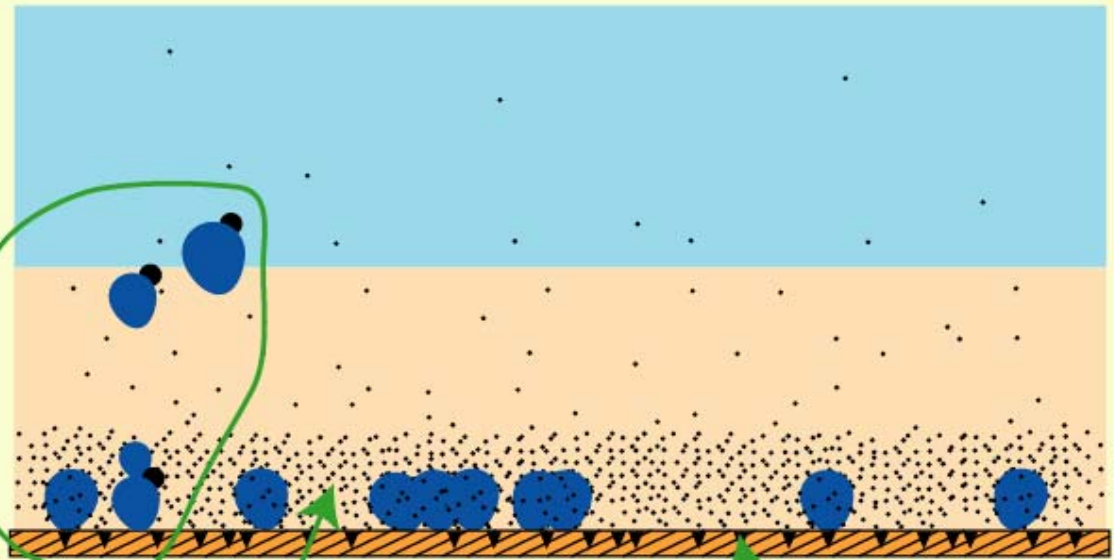
particle momentum transfer to bubbles

Bubbles grow through nanoparticles that are suspended  
in the lubricant excess layer, thus, performing surface  
work on the bubbles



# Enhancement or Degradation Realized Based on the Coupling of three heat transfer mechanisms:

*(1) boiling enhancement via nanoparticle interaction with bubbles (primarily momentum transfer effects)*



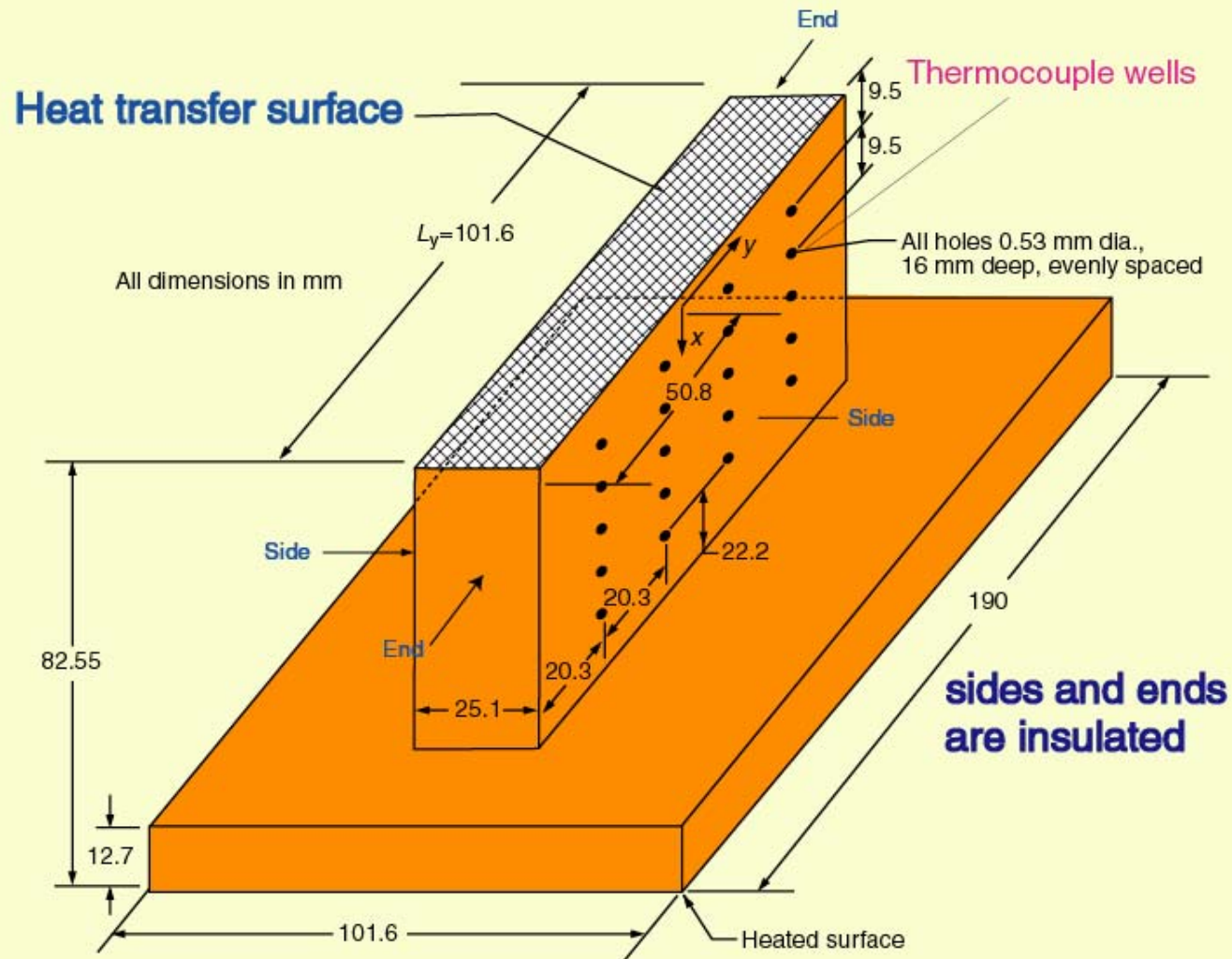
*(2) improved thermal conductivity of lubricant excess layer by the accumulation of highly conductive nanoparticles*

*(3) loss of nanosize nucleation sites due to nanoparticle filling of cavities.*

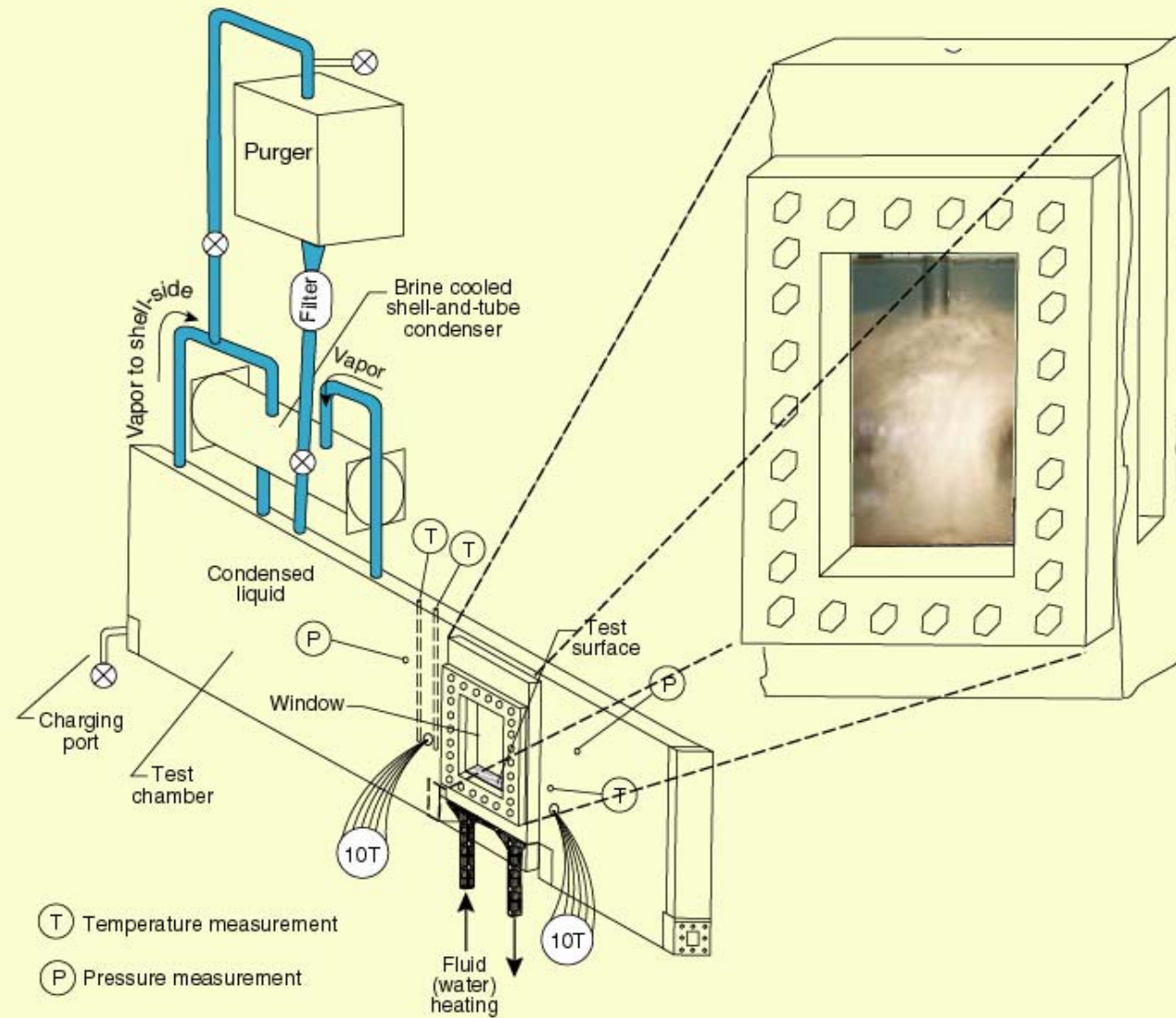
**Volume fraction determines if enough particles remain from mechanism (3) to be used in mechanisms (1) and (2)**



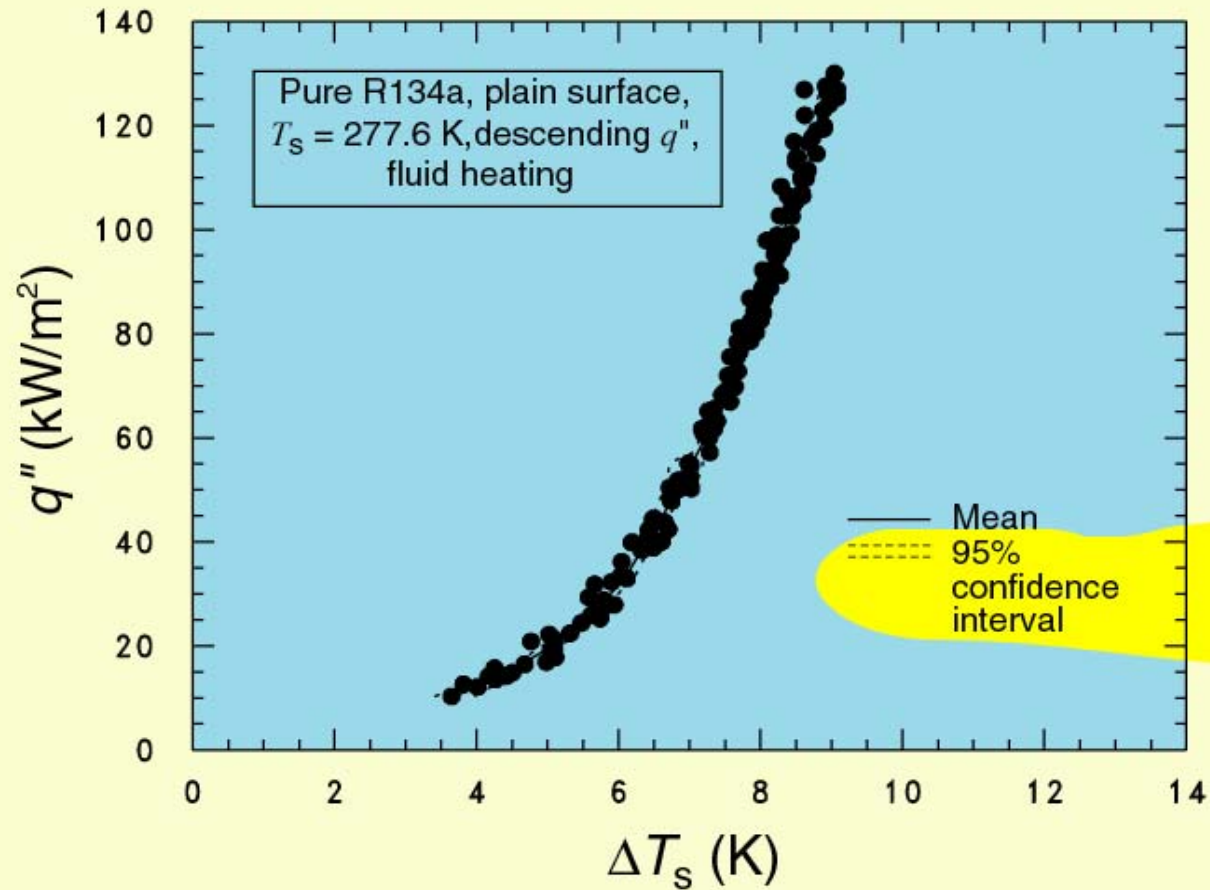
# OFHC Test Surface



# Schematic of Test Apparatus



# Pure R134a Pool Boiling



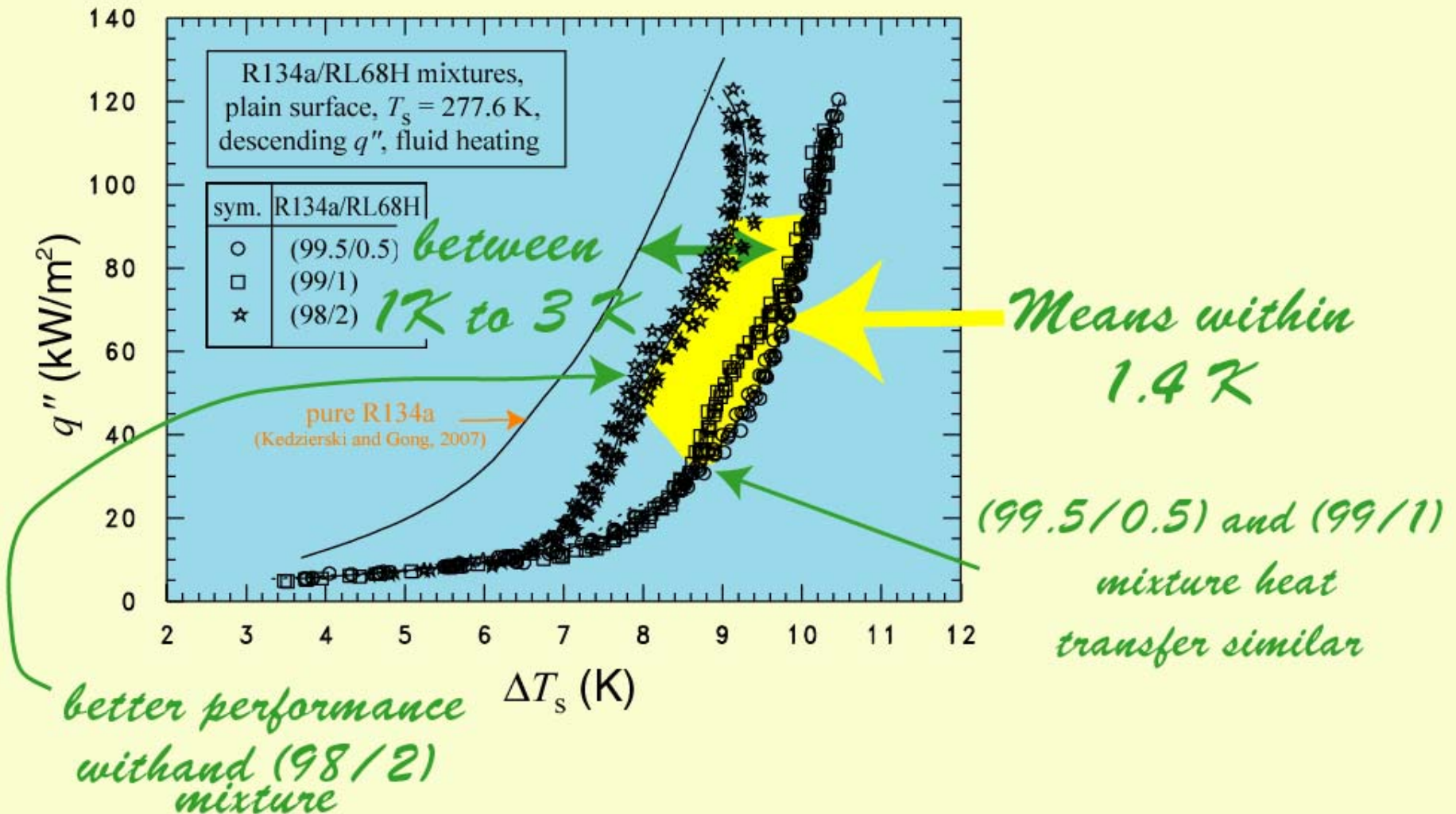
*145 pts. taken  
over 13 days*

*apprx.  
 $\pm 0.1$  K*



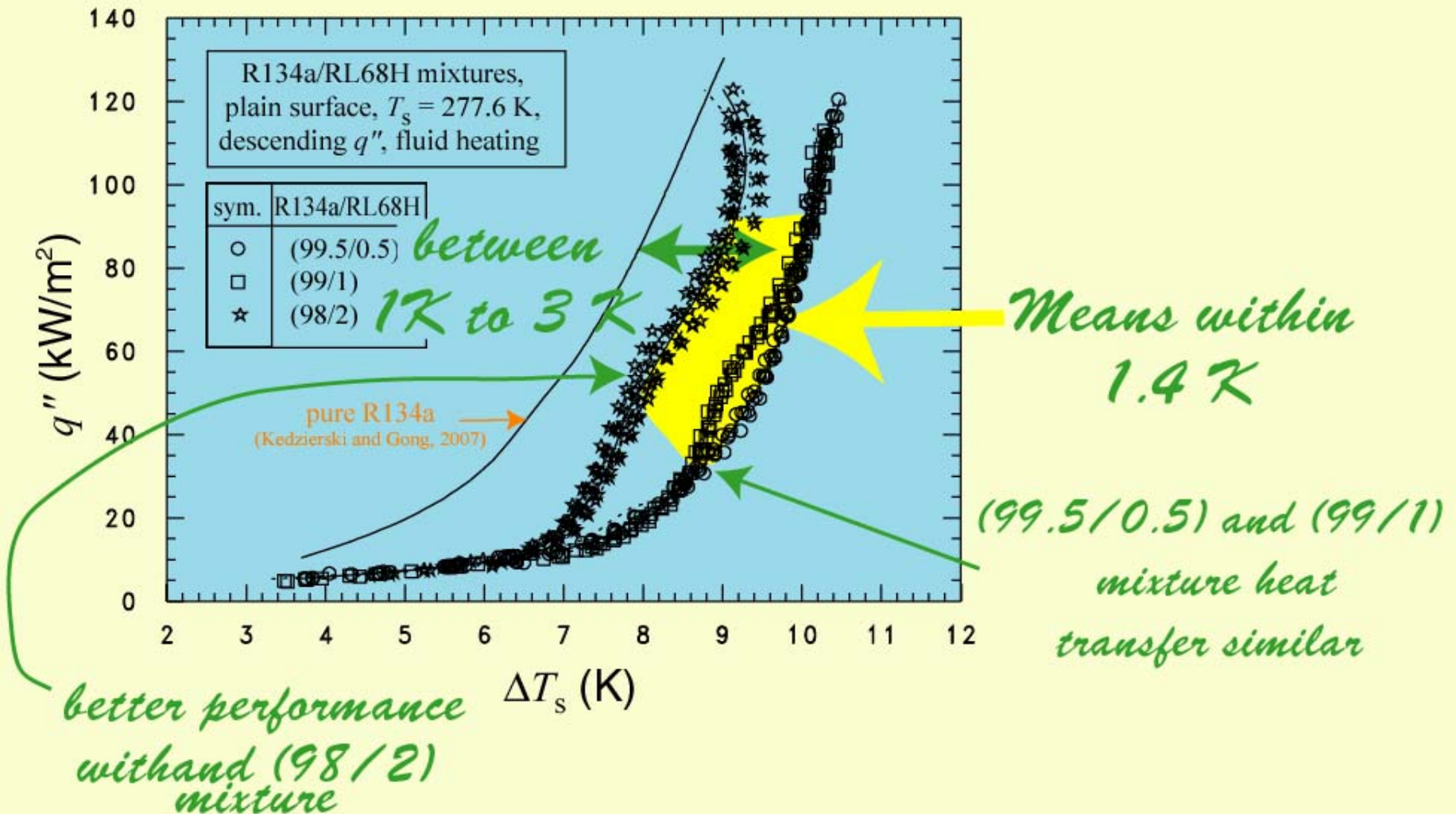
# Effect of Pure Lubricant on Boiling

## (Three Mixtures Tested)

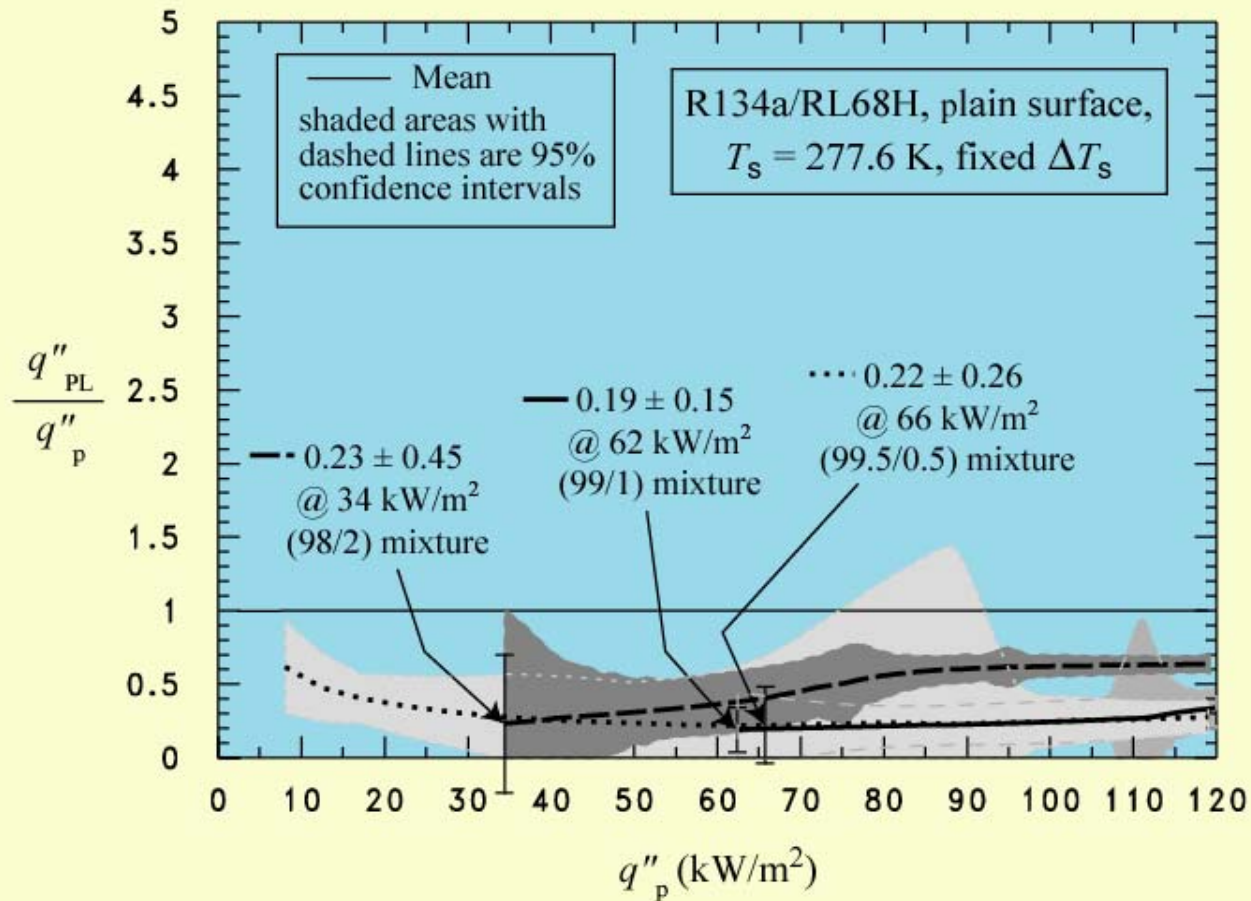


# Effect of Pure Lubricant on Boiling

## (Three Mixtures Tested)



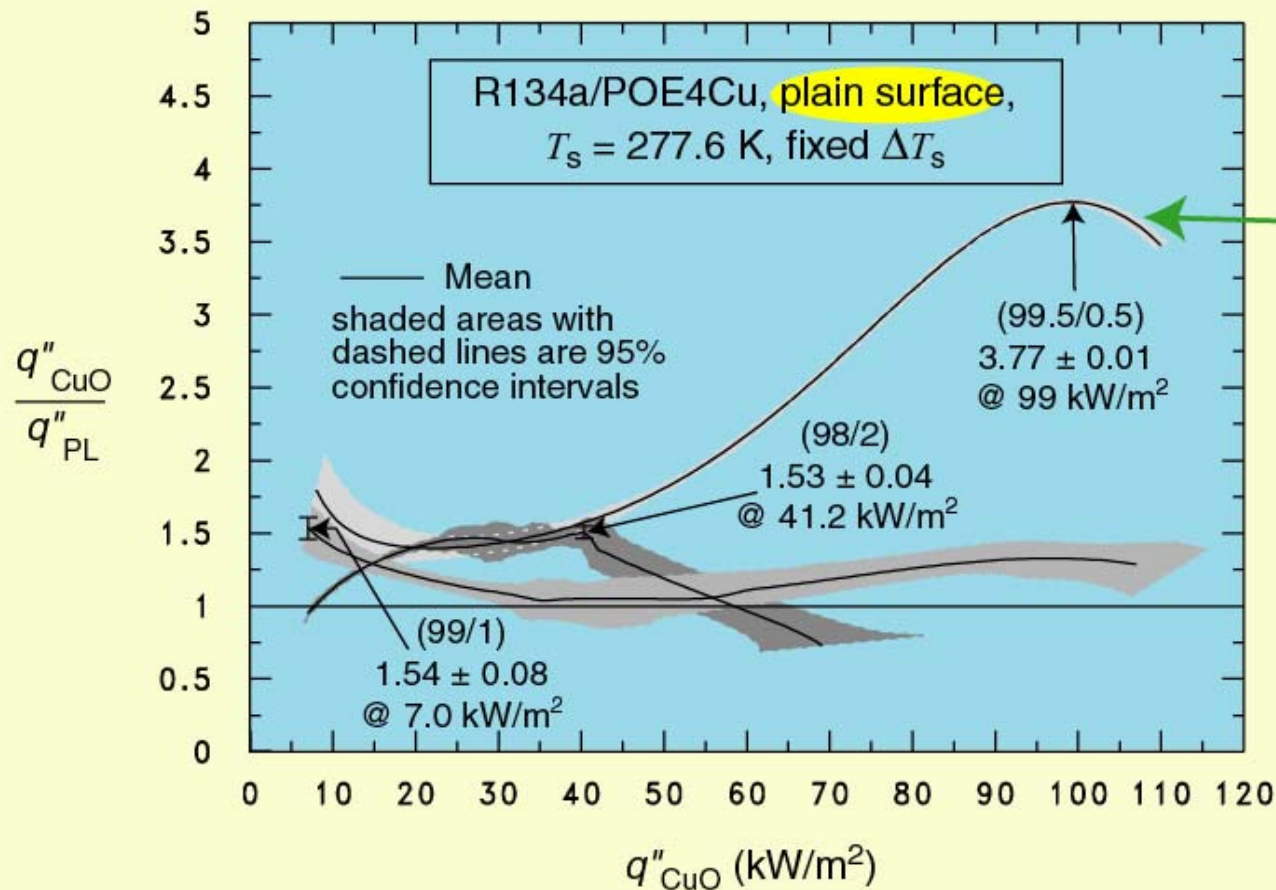
# Effect of Pure Lubricant on R134a Boiling



*Degradation similar  
for each lubricant  
mass fraction*



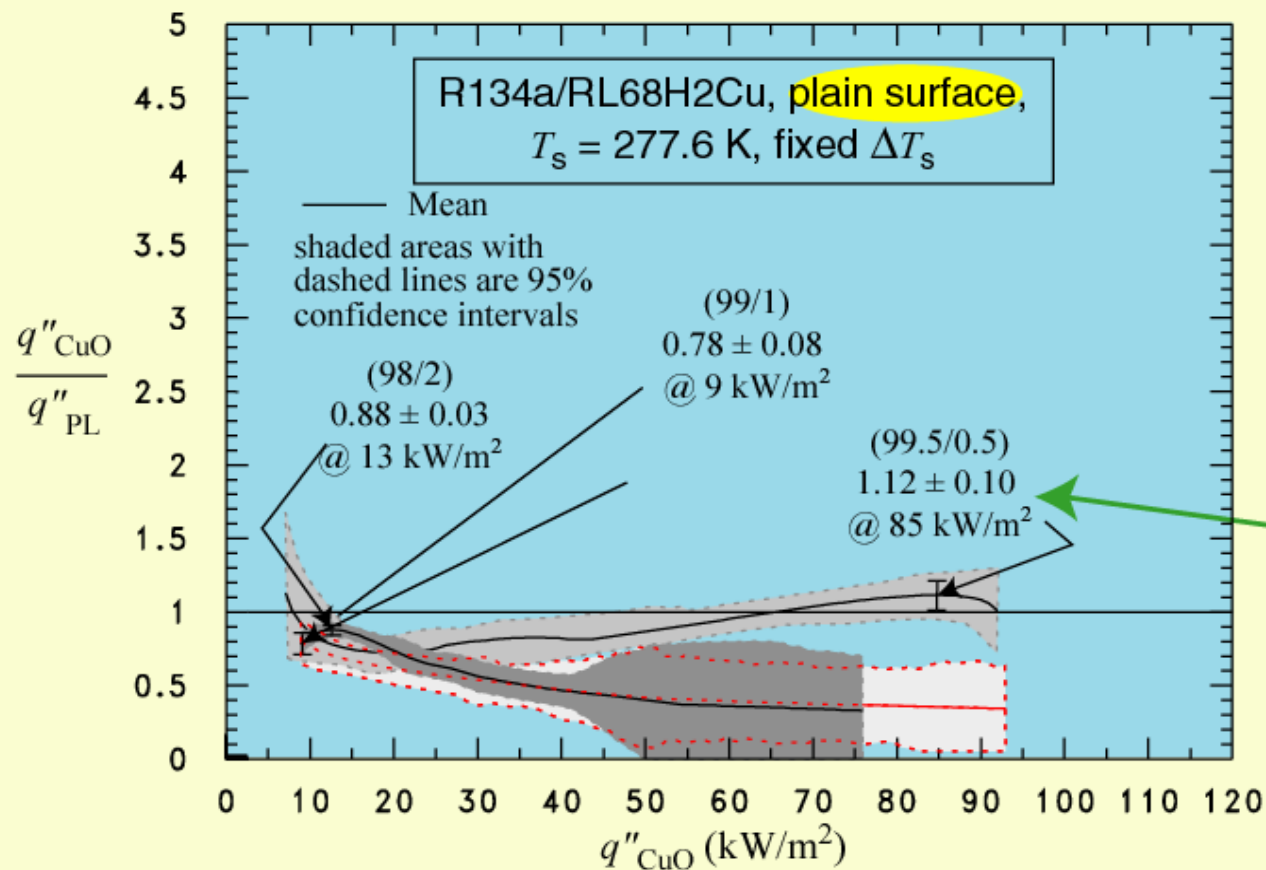
# Effect of 4% Vol. Fraction CuO Nanoparticles on Boiling



*0.5% mixture is  
roughly between  
1.5 and 3.75*

*Average ratio:  
0.5% → 2.4  
1% → 1.19  
2% → 1.12*

# Effect of 2% Vol. Fraction CuO Nanoparticles on Boiling

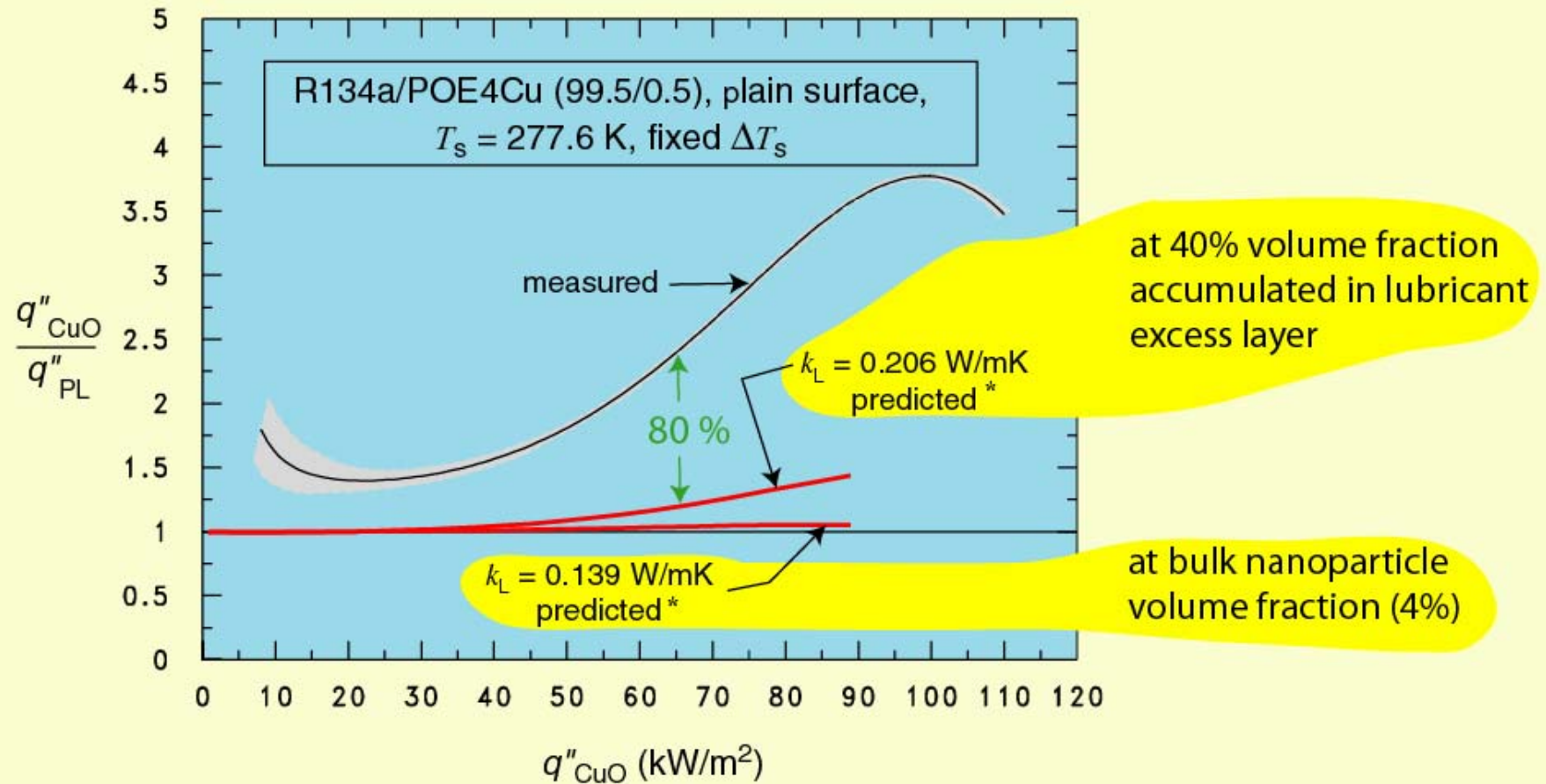


*Average ratio:*  
 0.5% → 0.9  
 1% → 0.4  
 2% → 0.5

*0.5% mixture is roughly between 0.73 and 1.12*

*Degradation may be due to fill of nano-size cavities causing a loss in active sites*

# Effect of Increased $k_L$ on Boiling



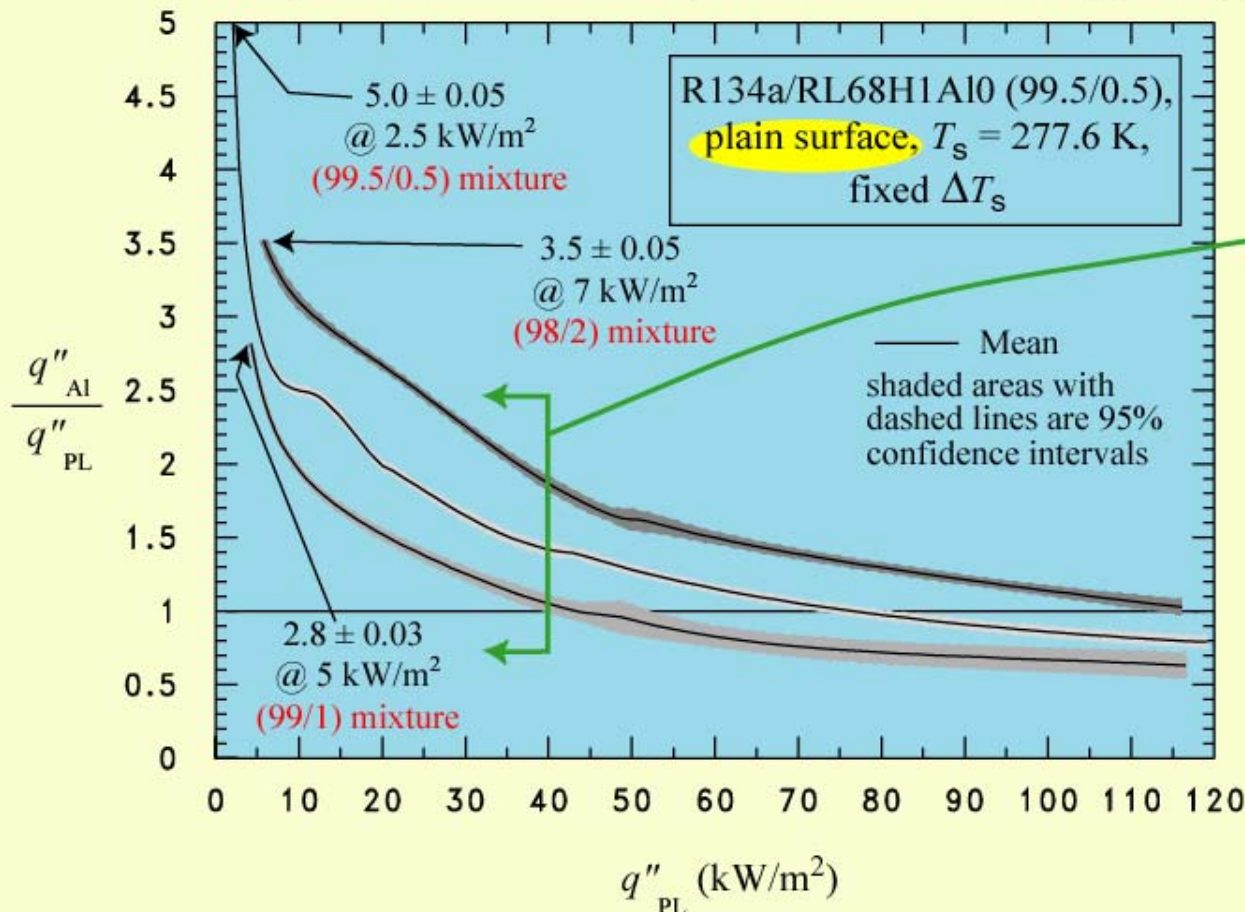
**Approx 20 % of the enhancement may be due to increased thermal conductivity**

\* Refrigerant/lubricant pool boiling model Kedzierski 2003 ICR



# Effect of $\text{Al}_2\text{O}_3$ Nanolubricant on R134a Boiling

Aluminum oxide nanoparticles provided the most favorable benefit to the 2 % mass fraction mixture



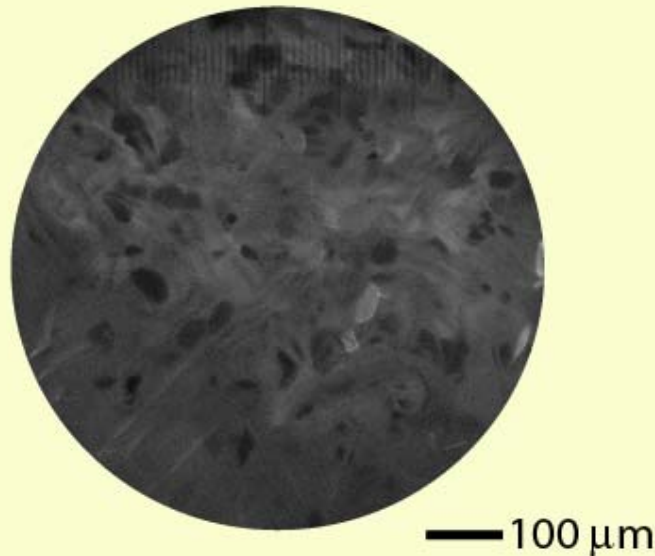
*Average ratio  
for  $< 40 \text{ kW/m}^2$ :*

- 0.5%  $\rightarrow 2.05$*
- 1%  $\rightarrow 1.49$*
- 2%  $\rightarrow 2.55$*

Enhancement occurred for the lowest heat fluxes giving the opportunity for chillers with lower approach temperatures

# Very Polydispersed Diamond Nanolubricant

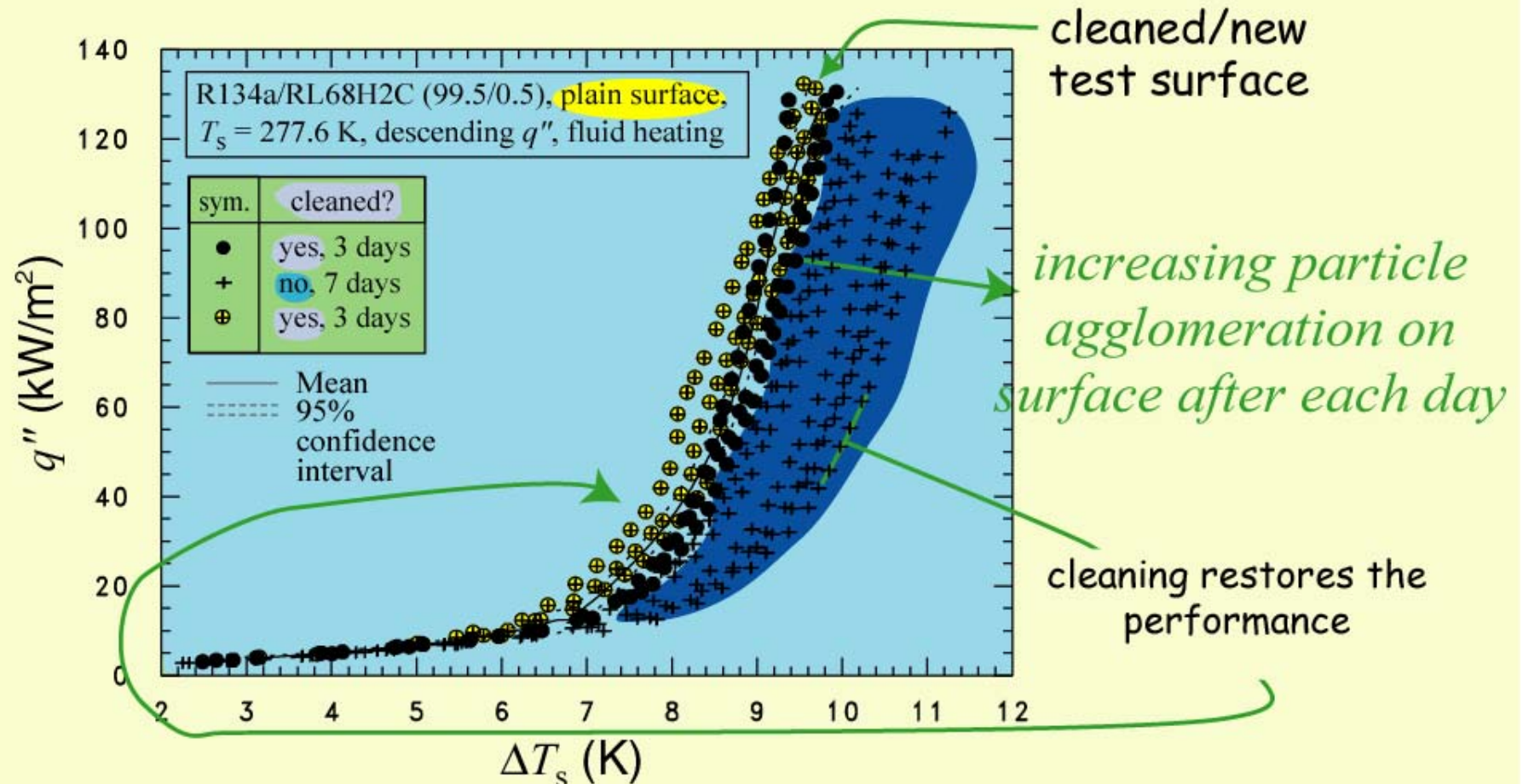
*Dynamic Light Scattering (DLS) and sieving technique  
using a syringe filter and an optical microscope*



The particles are dispersed from single 10 nm diameter particles to agglomerations of particles as large as 50 μm.



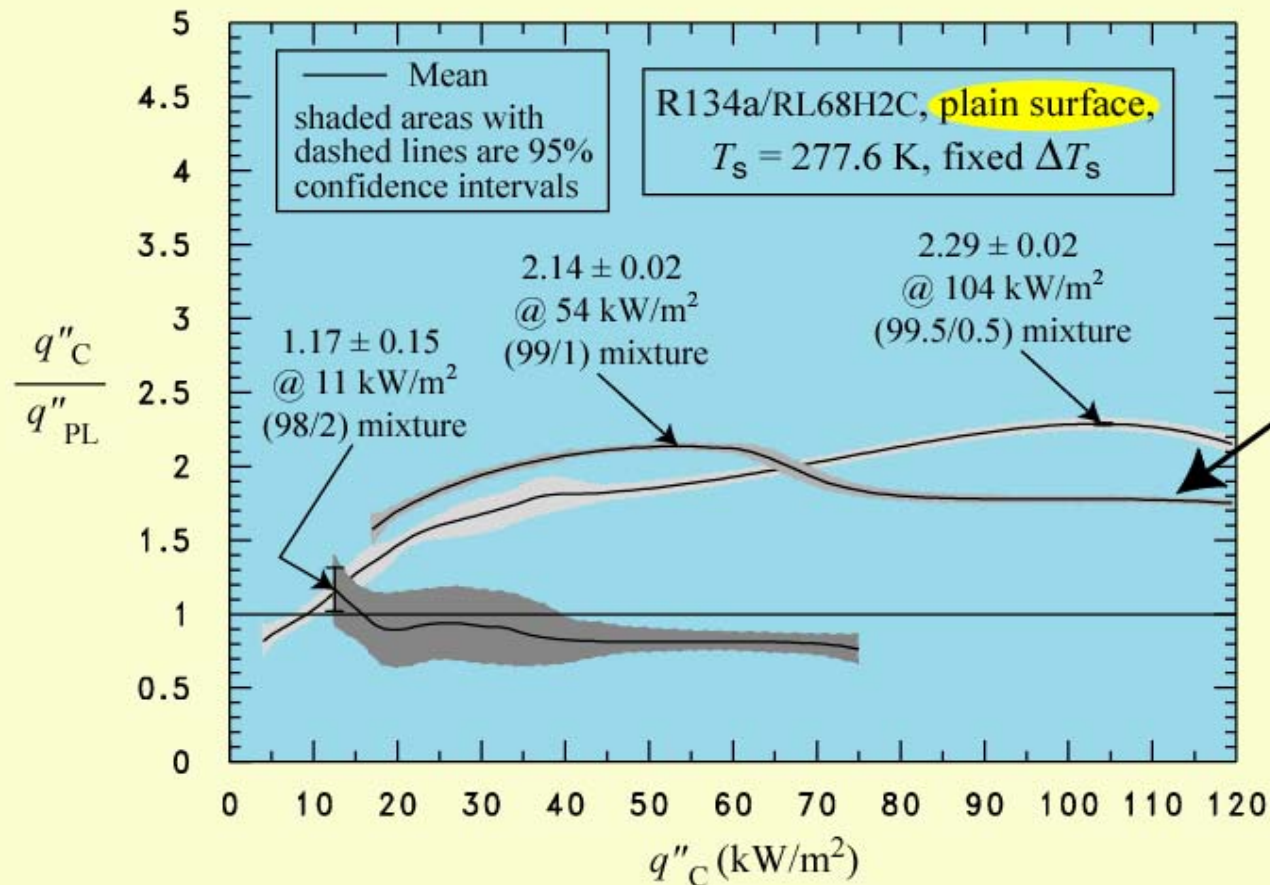
# Effect of Particle Agglomeration on R134a/Diamond Nanolubricant boiling



degradations increase as agglomerated nanoparticles settle out of the excess layer and into the cavities of the boiling surface



# Effect of Diamond Nanolubricant on Boiling (best performance)



*0.5% mixture is roughly between 0.8 and 2.29*

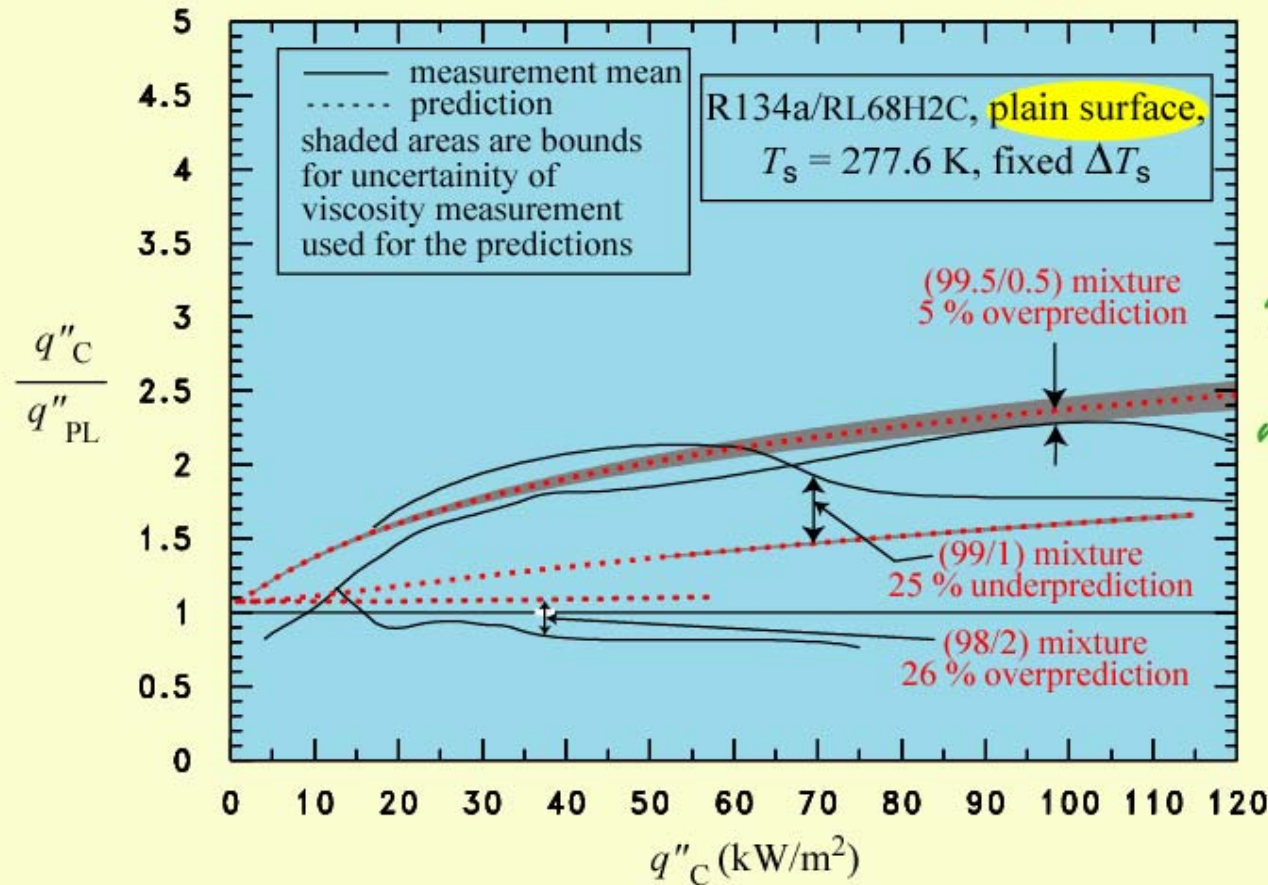
1% mixture is never less than 1.5

*Average ratio:*  
*0.5% → 1.98*  
*1% → 1.91*  
*2% → 0.81*

Sustainable improvement for wide heat flux range

# Increased Viscosity Caused Enhancement (not particle interaction)

*kinematic viscosity increased by 550%*

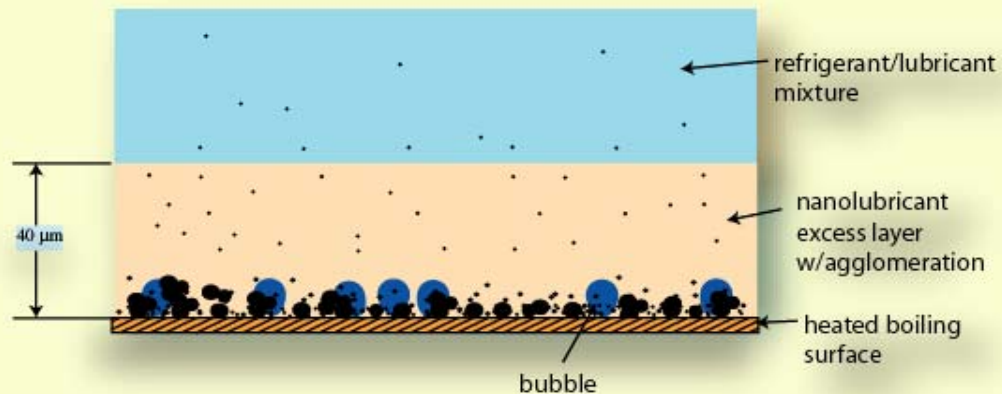


*Model can predict only  
property effects, not  
nanoparticle interaction  
with bubbles*

Refrigerant/lubricant boiling model predicts enhancement  
based on viscosity increase

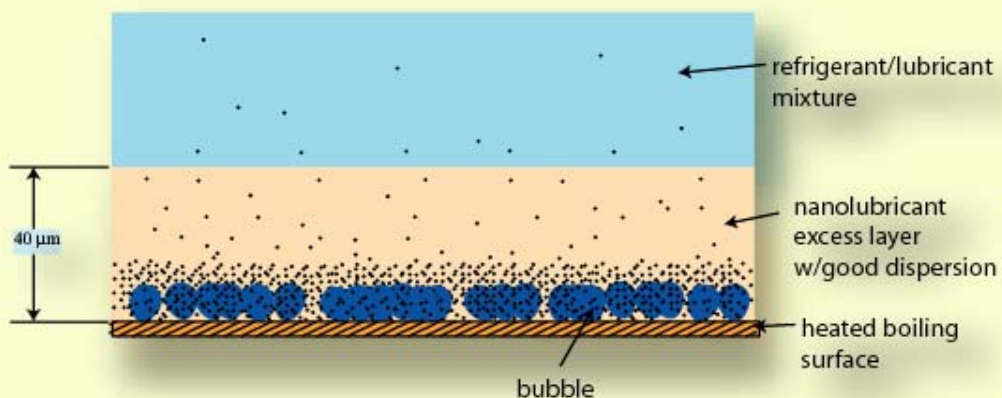


# Particle Interaction with Bubbles Depends on the Quality of the Dispersion



## Bad Dispersion

*Particles don't remain suspended in excess layer*

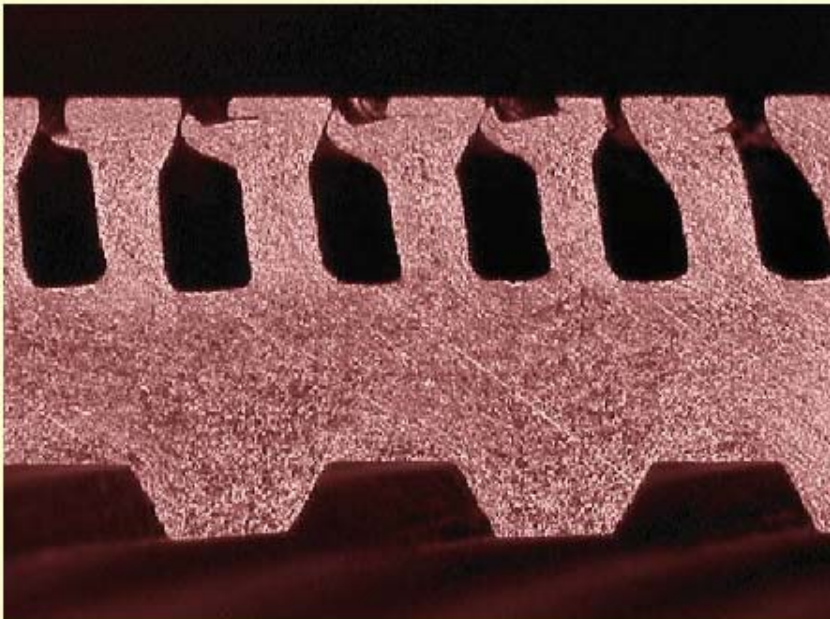


## Good Dispersion

*Particles remain suspended to interact with bubbles*



# Photograph of Reentrant Cavity Boiling Surface



— 0.1 mm

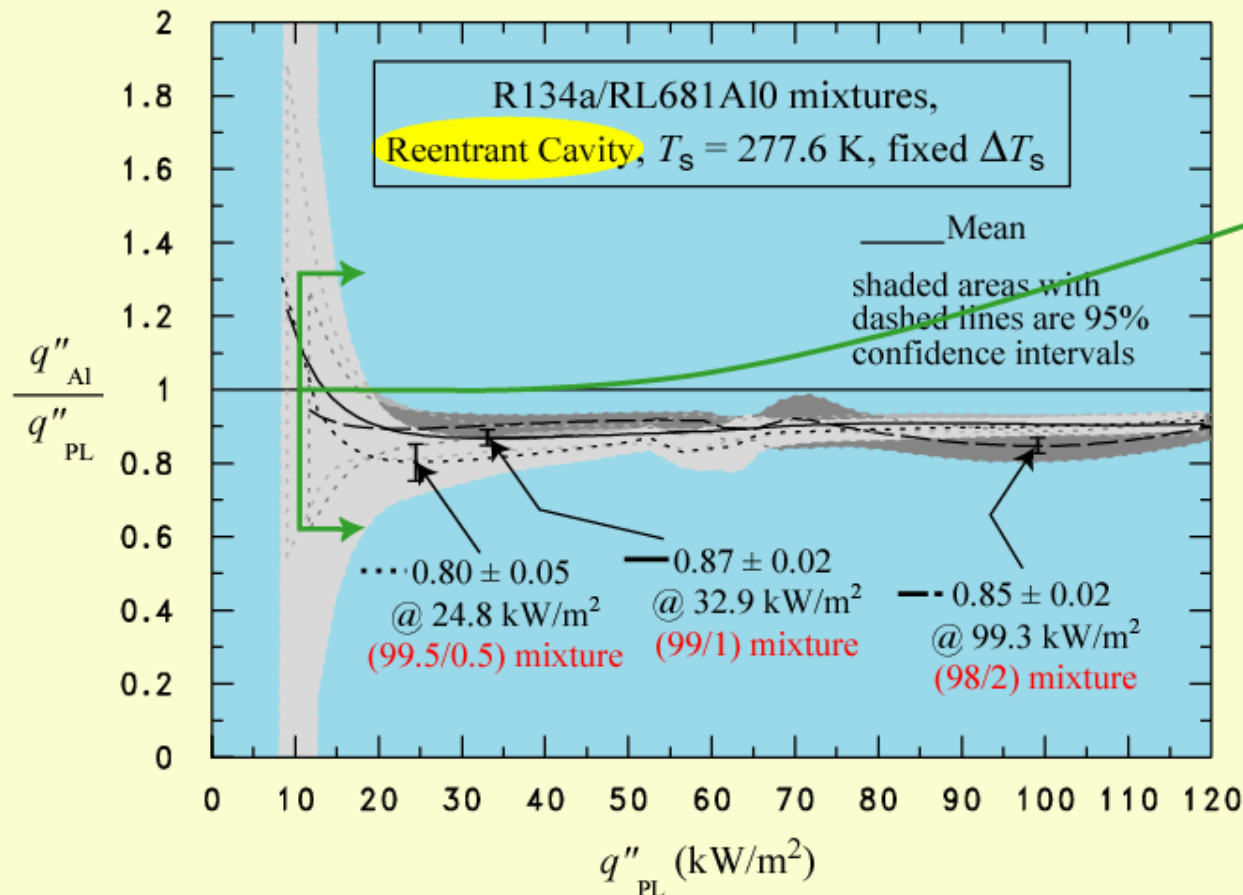
SIDE VIEW



— 0.1 mm

TOP VIEW

# Effect of $\text{Al}_2\text{O}_3$ Nanolubricant on Reentrant Cavity Boiling



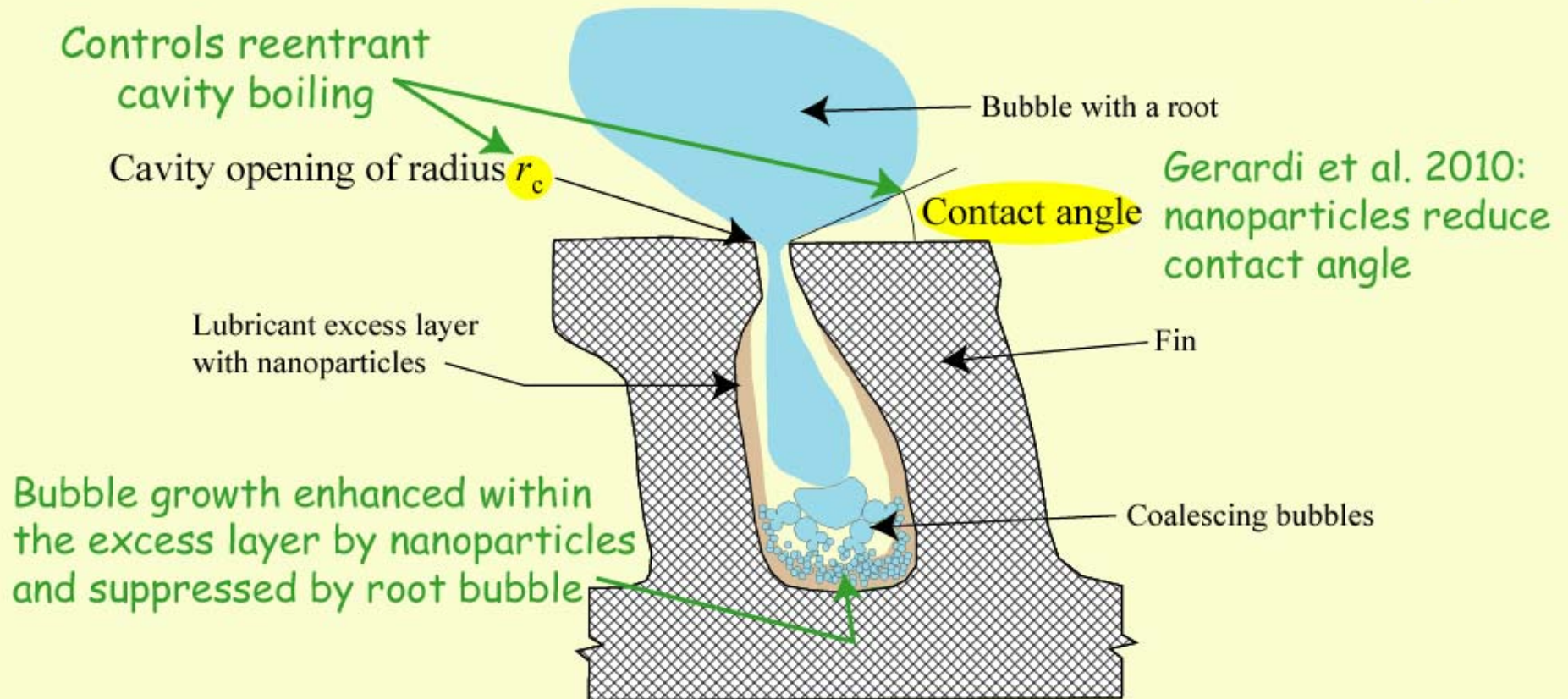
for  $> 10 \text{ kW/m}^2$   
average degradation:

0.5% → 13 %  
1% → 11 %  
2% → 11 %

Enhancement mechanism of the nanoparticles made redundant by the reentrant cavities of the boiling surface



# R134a/Nanolubricant Boiling Mechanism in Reentrant Cavity



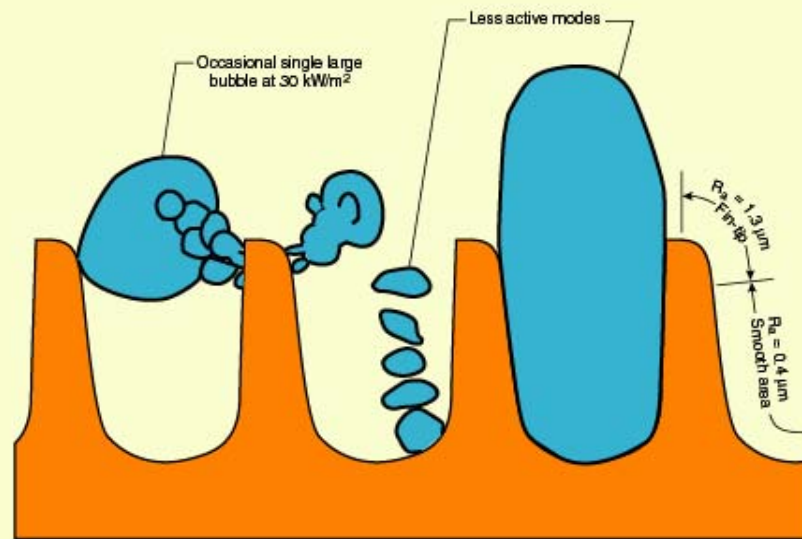
Positive effect of nanoparticles reduced because bubble nucleation in cavity less important and suppressed

Negative effect of reduced contact angle likely cause of 12 % degradation



# Current Work: Open Fin

*Trapezoidal finned tubes have water-side enhancements*



Four different modes of bubble evolution

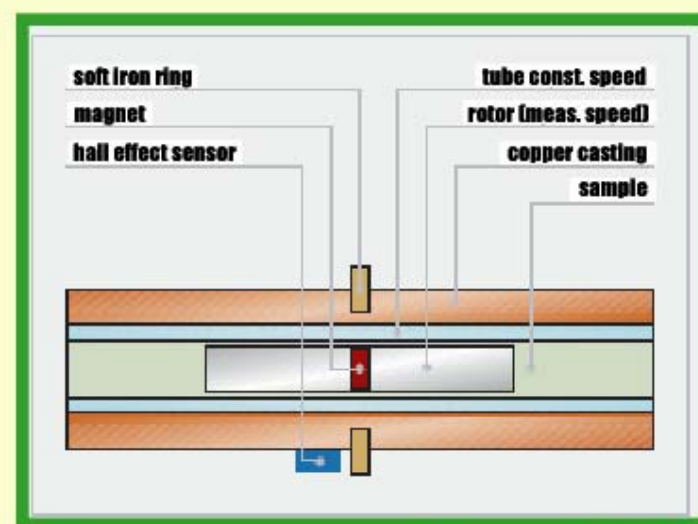
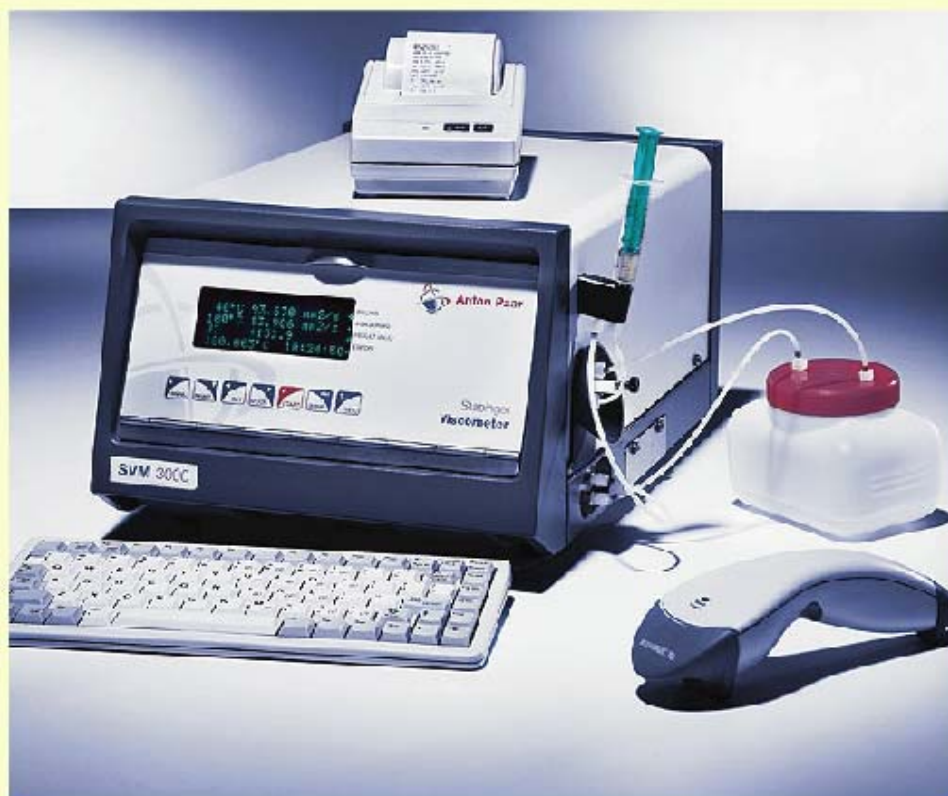
Performance governed more so by bubble nucleation

Improved potential for nanoparticles to enhance boiling performance

**Switch Gears from Boiling Measurements  
to Measurement of the Viscosity and  
the Density of Nanolubricants for  
Several  $\text{Al}_2\text{O}_3$  Mass Fractions and  
Several Surfactant Mass Fractions**

# Stabinger Viscometer

*Difference in speed and torque between the outer and inner cylinder is used to determine the dynamic viscosity.*

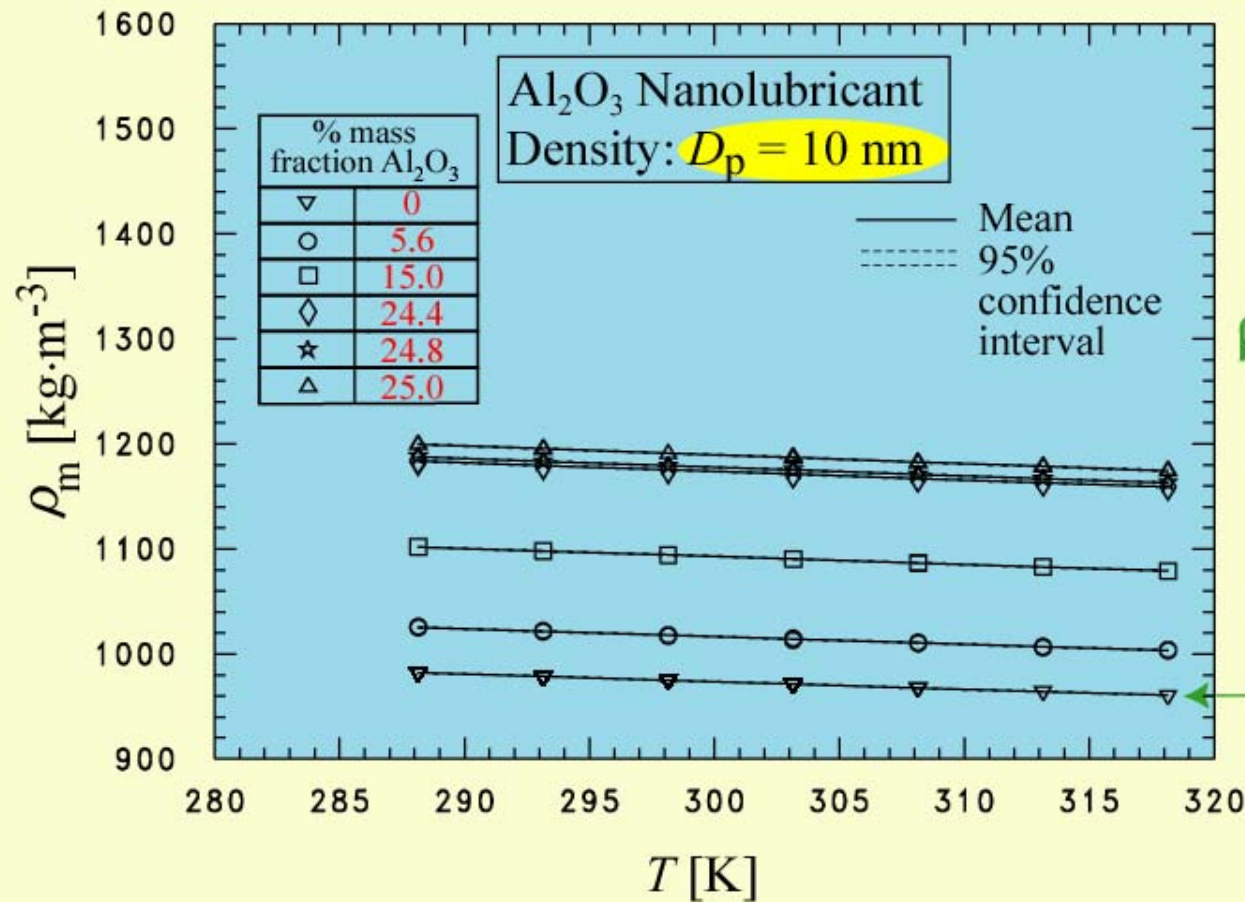


**rotating concentric cylinders**

Manufacturer quoted uncertainty for the kinematic viscosity and the density was  $\pm 0.35\%$  and  $\pm 0.5 \text{ kg}\cdot\text{m}^{-3}$



# Density of Nanolubricant for Several $\text{Al}_2\text{O}_3$ Mass Fractions

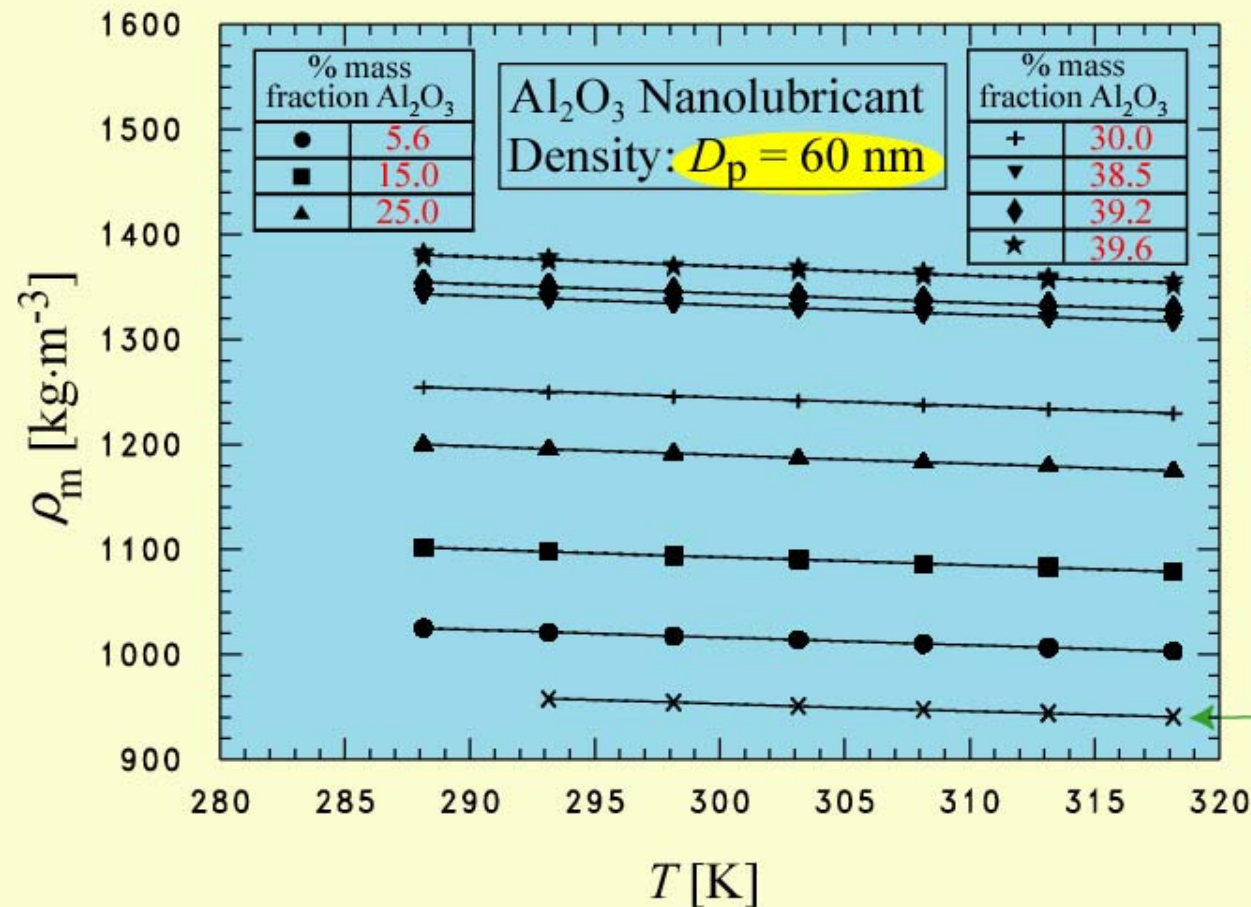


$$\rho_{np} = 3600 \text{ kg/m}^3$$

← neat lubricant

Density decreases with temperature and increases with mass fraction as expected

# Density of Nanolubricant for Several $\text{Al}_2\text{O}_3$ Mass Fractions



*larger mass fractions  
as compared to 10 up  
data*

50/50 surfactant/RL68H

Nanoparticle size does not appear to affect the density



# Linear Fit of Specific Volume with Respect to Temperature

$$\rho_m^{-1} [\text{m}^3 \cdot \text{kg}^{-1}] = B_0 + B_1 T [\text{K}]$$

Nanoparticle mass fraction

$x_{np}$	$x_s$	$x_L$	$D_p$ (nm)	$U_p$ (%)	Fitted Constant		Residual standard deviation of fit (%)
					$B_0$	$B_1$	
0	0	1.0	N/A	0.24	$0.7979 \times 10^{-3}$	$0.7647 \times 10^{-6}$	0.11
0.056	0.014	0.930	10	0.38	$0.7689 \times 10^{-3}$	$0.7164 \times 10^{-6}$	0.19
0.056	0.011	0.933	60	0.27	$0.7702 \times 10^{-3}$	$0.7132 \times 10^{-6}$	0.13
0.150	0.038	0.812	10	0.25	$0.7232 \times 10^{-3}$	$0.6399 \times 10^{-6}$	0.12
0.150	0.030	0.820	60	0.25	$0.7224 \times 10^{-3}$	$0.6429 \times 10^{-6}$	0.12
0.250	0.062	0.688	10	0.30	$0.6617 \times 10^{-3}$	$0.5964 \times 10^{-6}$	0.15
0.250	0.050	0.700	60	0.23	$0.6638 \times 10^{-3}$	$0.5887 \times 10^{-6}$	0.11
0.248	0.078	0.674	10	0.23	$0.6707 \times 10^{-3}$	$0.5945 \times 10^{-6}$	0.11
0.244	0.091	0.665	10	0.23	$0.6728 \times 10^{-3}$	$0.5975 \times 10^{-6}$	0.11
0.300	0.060	0.640	60	0.21	$0.6434 \times 10^{-3}$	$0.5334 \times 10^{-6}$	0.10
0.396	0.079	0.525	60	1.23	$0.5870 \times 10^{-3}$	$0.4767 \times 10^{-6}$	0.62
0.392	0.098	0.510	60	0.21	$0.5981 \times 10^{-3}$	$0.4869 \times 10^{-6}$	0.10
0.385	0.115	0.500	60	0.21	$0.6015 \times 10^{-3}$	$0.4964 \times 10^{-6}$	0.10
0	0.500	0.500	N/A	0.24	$0.8211 \times 10^{-3}$	$0.7607 \times 10^{-6}$	0.11
0	1.0	0	N/A	0.34	$0.8443 \times 10^{-3}$	$0.7567 \times 10^{-6}$	N/A

surfactant mass fraction

pure lubricant mass fraction

nanoparticle diameter



# All Specific Volume Predicted Within 1 % With Wasp et al. (1977):

$$\frac{1}{\rho_m} = \frac{x_s}{\rho_s} + \frac{x_L}{\rho_L} + \frac{x_{np}}{\rho_{np}}$$

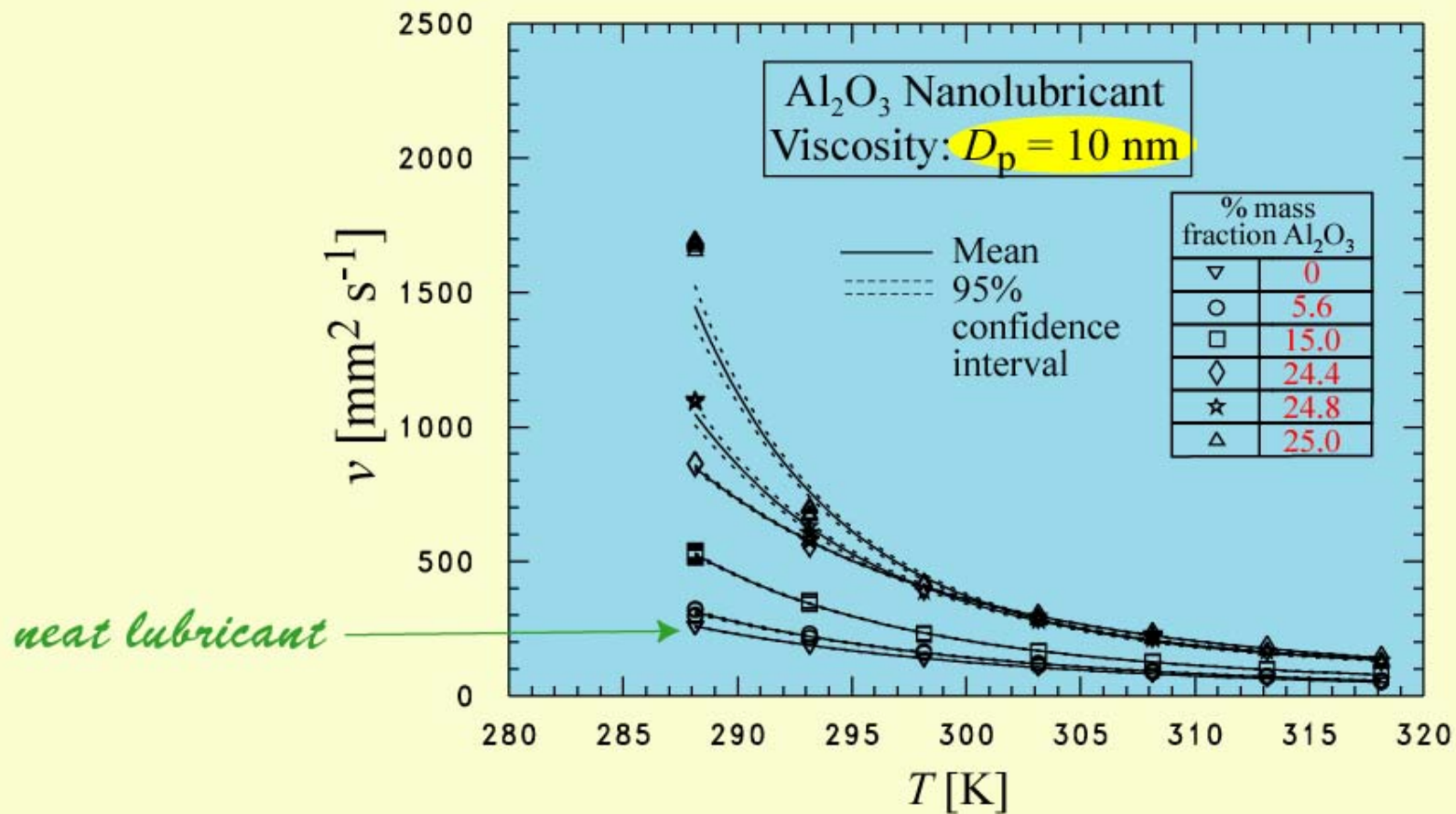
*MIXTURE* —  $\rho_m$  — *SURFACTANT* —  $\rho_s$  — *LUBRICANT* —  $\rho_L$  — *NANOPARTICLE* —  $\rho_{np}$

*within 0.5 % for mass fractions less than or equal to 0.3*

Substituting values from individual fits gives:

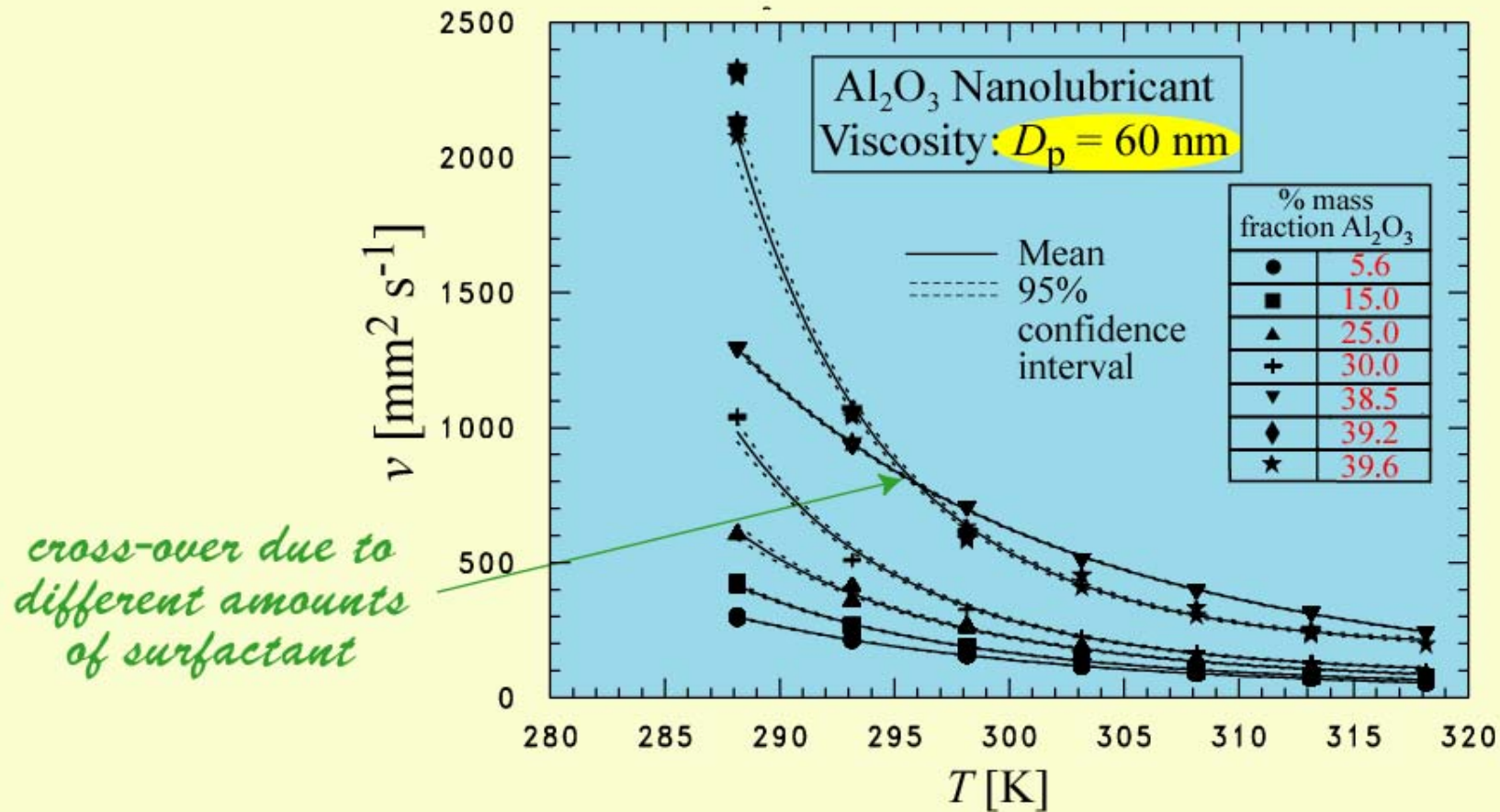
$$\frac{1}{\rho_m} [\text{kg}^{-1} \cdot \text{m}^3] = (7.647 \times 10^{-7} (1 - x_{np}) - 8.647 \times 10^{-9} x_s) T [\text{K}] + 7.979 \times 10^{-4} - 5.201 \times 10^{-4} x_{np} + 4.640 \times 10^{-5} x_s$$

# Viscosity of Nanolubricant for Several $\text{Al}_2\text{O}_3$ Mass Fractions



Larger uncertainties for larger mass fractions

# Viscosity of Nanolubricant for Several $\text{Al}_2\text{O}_3$ Mass Fractions



Nanoparticle size does affect the viscosity



# Normalized viscosity fitted to normalized temperature

$$\nu_0 = 1 \text{ mm}^2 \cdot \text{s}^{-1}$$

$$\frac{\nu}{\nu_0} = \exp \left( A_0 + \frac{A_1}{T_r} + A_2 \ln(T_r) + A_3 T_r^{A_4} \right)$$

*not significant*

$$T_r = T/273.15 \text{ K}$$

$x_{np}$	$x_s$	$x_L$	$D_p$ (nm)	$U_v$ (%)	Fitted Constant			Residual standard deviation of fit (%)
					$A_0$	$A_1$	$A_2$	
0	0	1.0	N/A	2.1	-52.1912	58.8418	36.8165	1.0
5.6	0.014	0.930	10	7.5	-69.7768	76.7281	52.0952	3.8
5.6	0.011	0.933	60	2.7	-60.8428	67.7102	44.1411	1.3
15.0	0.038	0.812	10	3.6	-146.202	154.079	119.893	1.8
15.0	0.030	0.820	60	4.2	-147.872	155.481	121.854	2.1
25.0	0.062	0.688	10	14.6	-358.951	368.915	309.006	7.4
25.0	0.050	0.700	60	12.8	-194.279	202.522	163.128	6.5
24.8	0.078	0.674	10	9.6	-237.389	246.384	201.732	4.8
24.4	0.091	0.665	10	2.7	-113.035	121.208	91.3062	1.3
30.0	0.060	0.640	60	9.8	-302.099	311.362	258.869	4.9
39.6	0.079	0.525	60	14.0	-386.581	396.955	335.349	7.1
39.2	0.098	0.510	60	1.7	-68.7064	76.9815	52.4197	0.8
38.5	0.115	0.500	60	2.5	-36.9608	45.1948	23.9985	1.2
0	0.500	0.500	N/A	4.8	-246.727	257.904	208.615	2.4

*residuals for the fits are within 8 % for all of the fluids*

# Model for Nanolubricant Viscosity

General mixing rule:

$$\ln v_m = x_L^{1.25} \ln v_L + x_{np}^{1.25} \ln v_{np} + x_s^{1.25} \ln v_s$$

*exponent changed from 1 to 1.25 for improved fit*

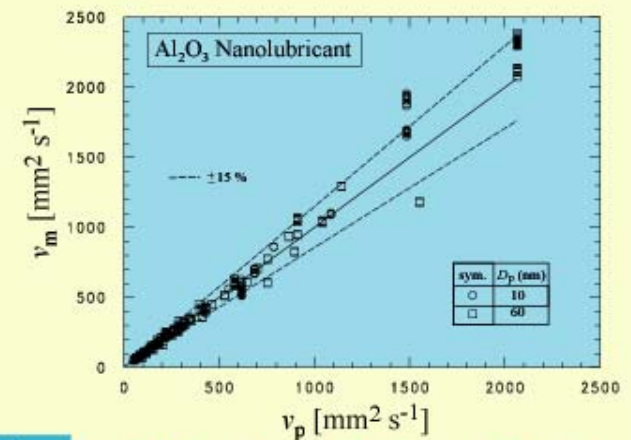
Pseudo-surfactant viscosity:

$$\ln v_s = 0.149 D_{np} [\text{nm}] - 87.2079 + \frac{7.1353}{T_r^{-66.12} + 0.074}$$

Pseudo-nanoparticle viscosity:

$$\ln v_{np} = (1.426 - 0.0071 D_{np} [\text{nm}]) \left( 4.7356 + \frac{1.4706}{T_r^{4.05} - 1.11} \right)$$

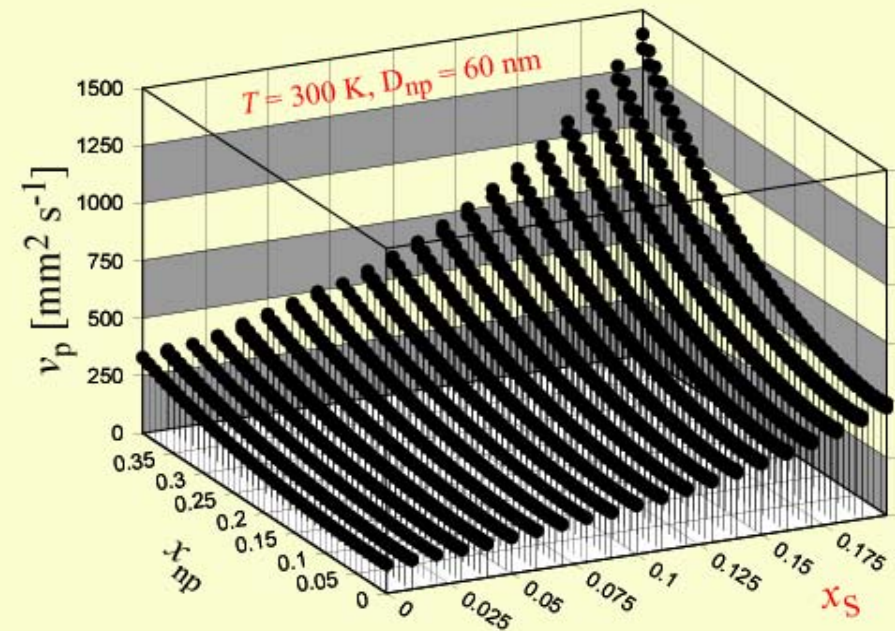
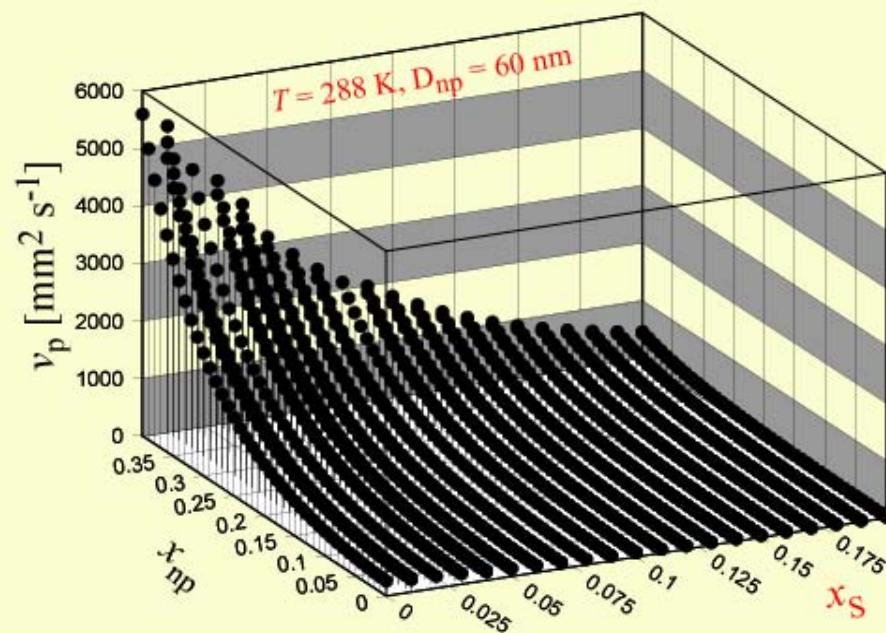
**Nanoparticle diameter** term accounts for the interaction between the nanoparticle and the surfactant



*95 % within 15 %*



# Effect of Temperature and Surfactant Mass Fraction on Viscosity



Between 300 K and 318 K, increase in  $x_s$  causes an increase in viscosity, while the opposite is true for temperatures between 288 K and 300 K



# **Boiling Conclusions**

# Conclusions Plain Surface

- In general, nanoparticles can be used to produce significant enhancement relative to the heat transfer of pure R134a/polyolester
- However, the choice of the nanoparticle material, size, and concentration is critical in order to achieve a sustained and significant enhancement
- A high quality nanolubricant dispersion is essential for obtaining a boiling enhancement via momentum transfer from nanoparticles to bubbles.
- A bad dispersion can give an boiling enhancement via increase viscosity, but it is short-lived.
- The thermal conductivity of the nanoparticle does not play a large role in the boiling heat transfer enhancement.

# Conclusions Plain Surface (cont.)

- A semi-empirical model is now available to predict the enhancement of refrigerant/lubricant pool boiling by assuming that the transfer of momentum from the nanoparticles to the bubbles is responsible for the boiling enhancement
- For heat fluxes greater than  $20 \text{ kW/m}^2$ , the model underpredicted the (99.5/0.5), the (99/1), and the (98/2) mixtures on average by approximately 25 %, 0.2 %, and 6 %, respectively.
- The model predicts that the maximum performance is approached for volume fraction and mass fractions nearing unity, and forever decreasing nanoparticle size.
- Future research is required to validate the model beyond the range of parameters investigated here.



# Conclusions: Reentrant Cavity

- $\text{Al}_2\text{O}_3$  nanoparticles caused, on average, a 12 % degradation in the boiling heat transfer relative to that for R134a/polyolester mixtures without nanoparticles for the three lubricant mass fractions that were tested.
- It was speculated that the degradation resulted from nucleation being less important and suppressed for reentrant cavity boiling and increased surface wetting (reduced contact angle).

# **Viscosity Conclusions**

# Conclusions

- Liquid kinematic viscosity and liquid density measurements of a synthetic polyolester based aluminum oxide ( $\text{Al}_2\text{O}_3$ ) nanoparticle dispersion (nanolubricant) have been presented at atmospheric pressure and for a temperature range from 288 K to 318 K.
- Viscosity and density measurements were made for the pure base lubricant along with twelve nanolubricants with differing nanoparticle mass fractions.
- The liquid kinematic viscosity was correlated with respect to temperature, nanoparticle mass fraction, surfactant mass fraction, and nanoparticle diameter.



# Conclusions (cont.)

- Pseudo-viscosities were developed to account for the interaction between the nanoparticle and the surfactant.
- A linear relationship was developed for liquid specific volume with respect to temperature
- Both the liquid density and the viscosity decreased with respect to temperature and increased with respect to the  $\text{Al}_2\text{O}_3$  nanoparticle mass fraction
- Depending on the temperature, the surfactant caused the viscosity to either increased or decrease with respect to  $x_s$
- The measurements are important for the design of nanolubricants for heat transfer and flow applications